

**Vehicle Weights and Dimensions Study**

**Volume 16**

**A Comparison of Various Computer  
Simulation Models for Predicting the  
Lateral Behaviour of Articulated Buses**

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The Technical Steering Committee will be considering the findings of these research investigations in preparing its "Final Technical Report" (Volume 1 & 2), scheduled for completion in December 1986.

## PREFACE

The report which follows constitutes one volume in a series of sixteen which have been produced by contract researchers involved in the Vehicle Weights and Dimensions Study. The research procedures and findings contained herein address one or more specific technical objectives in the context of the development of a consistent knowledge base necessary to achieve the overall goal of the Study; improved uniformity in interprovincial weight and dimension regulations.

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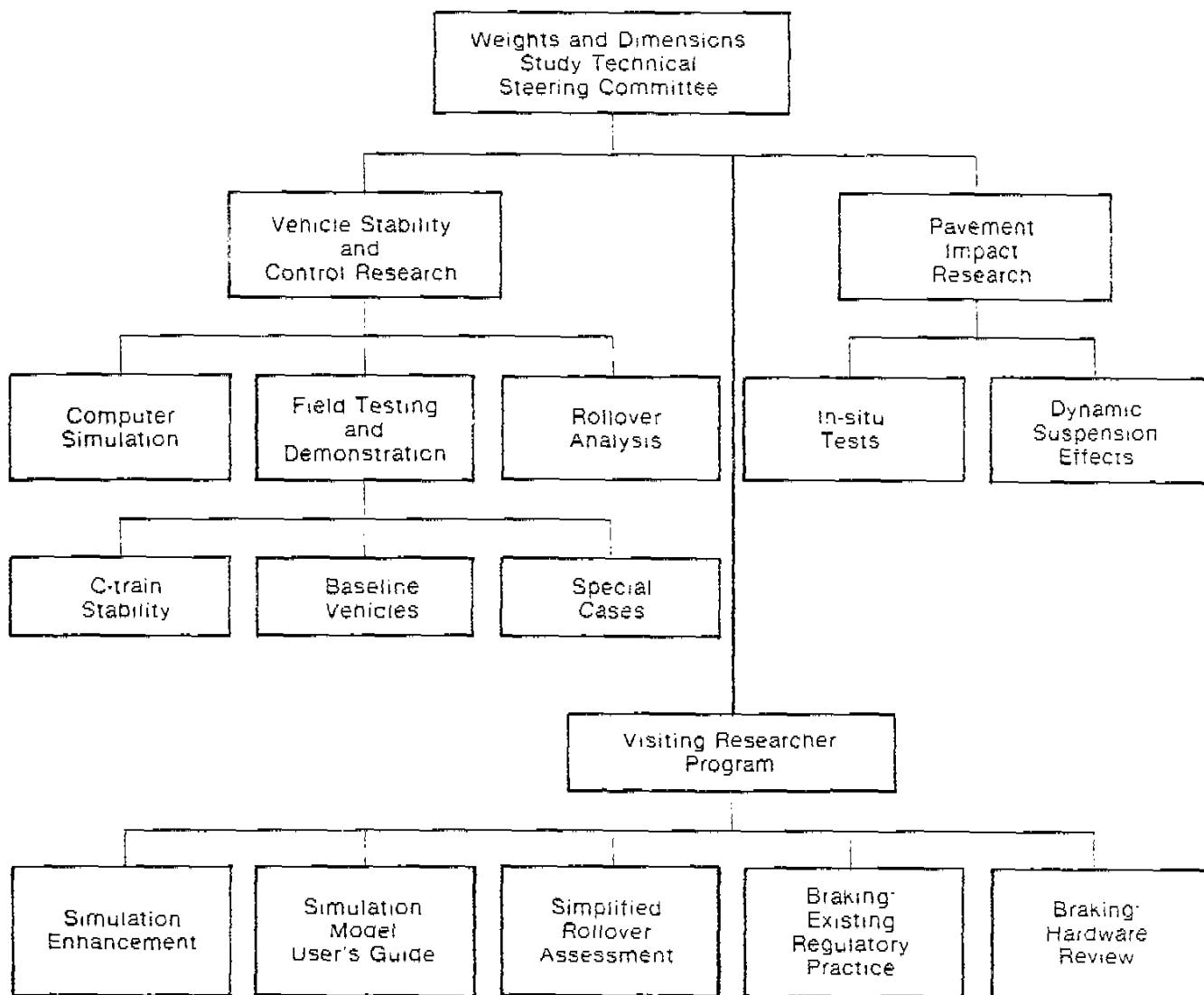
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## HEAVY VEHICLE WEIGHTS AND DIMENSIONS STUDY

### TECHNICAL WORK ELEMENTS OVERVIEW



**Volume 16**

**A Comparison of Various Computer Simulation Models for Predicting  
the Lateral Dynamic Behaviour of Articulated Vehicles**

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## TABLE OF CONTENTS

Foreword	i
Summary	ii
Introduction	1
1. Brief Description of the Various Simulation Models Used in the Study	3
1.1 The Linear Yaw Plane Model	3
1.2 The TBS Model	6
1.3 The Yaw/Roll Model	8
1.4 The Phase 4 Model	12
1.5 The Static Roll Model	16
2. Vehicle Configurations Used in the Simulation Study	23
3. A Comparison of the Capabilities of Various Computer Simulation Models in Predicting Steady-State Steering Responses	37
3.1 Vehicle Configuration 1	39
3.2 Vehicle Configuration 2	49
3.3 Vehicle Configuration 3	58
3.4 Vehicle Configuration 4	65
3.5 Vehicle Configuration 5	72
3.6 Vehicle Configuration 6	79

<b>4. A Comparison of Various Computer Simulation Models for Predicting Steering Responses in Lane-Change Type Transient Manoeuvres</b>	<b>86</b>
4.1 Steering Response of Vehicle Configuration 1 in a Lane-Change Manoeuvre	87
4.2 Steering Response of Vehicle Configuration 3 in a Severe Lane-Change Manoeuvre	95
4.3 Steering Response of Vehicle Configuration 6 in a Severe Lane-Change Manoeuvre	102
<b>5. A Parametric Sensitivity Study of the Static Roll Model</b>	<b>110</b>
5.1 Tractor Frame Torsional Stiffness	112
5.2 Coulomb Friction Present in the Tractor Frame	113
5.3 Equivalent Trailer Structural and Fifth Wheel Compliance	114
5.4 Separation of Fifth Wheel Plates (Lash)	115
<b>Discussions</b>	<b>118</b>
<b>References</b>	<b>122</b>
APPENDIX A - Vehicle Configuration 1 Input Data Files	A-1
APPENDIX B - Vehicle Configuration 2 Input Data Files	B-1
APPENDIX C - Vehicle Configuration 3 Input Data Files	C-1
APPENDIX D - Vehicle Configuration 4 Input Data Files	D-1
APPENDIX E - Vehicle Configuration 5 Input Data Files	E-1
APPENDIX F - Vehicle Configuration 6 Input Data Files	F-1
APPENDIX G - Basic Parameters of a 5-axle Dirt Truck	G-1

## FOREWORD

The work described in this report was performed under the Visiting Researcher Program (Computer Simulation Models) of the RTAC/CCMTA Vehicle Weights and Dimensions Study. This report, together with a previous report entitled "Computer Simulation of Heavy Vehicle Dynamic Behaviour - User's Guide to the UMTRI Models", by J.Y. Wong and M. El-Gindy, June 1985, forms the final Report to RTAC/CCMTA.

The study described in this report was performed by Dr. J.Y. Wong, Director, and Dr. M. El-Gindy, Senior Research Associate, Transport Technology Research Laboratory, Carleton University, Ottawa, Canada.

The guidance provided by the Project Manager and the Technical Steering Committee of the Vehicle Weights and Dimensions Study, and the assistance given by the staff members of the Engineering Research Division, University of Michigan Transportation Research Institute (UMTRI) during the course of the study are appreciated.

April 1986

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## SUMMARY

The objective of this study is to compare the capabilities and limitations of various simulation models for the evaluation of the directional behaviour of commercial articulated vehicles. The steady-state steering response and the lateral dynamic behaviour in lane-change (obstacle avoidance) manoeuvres of six articulated vehicles with different design features have been predicted using the Phase 4 model, the yaw/roll model, the TBS model, and the linear yaw plane model, developed by the University of Michigan Transportation Research Institute (UMTRI). A comparison of the predictions made by these models has been performed. These predictions have also been compared with a limited amount of experimental data available. In addition, the static roll model developed by UMTRI has been examined, and a parametric sensitivity study has been performed.

It is believed that the results of this study provide additional quantitative information upon which the appropriate areas of application of various simulation models may be defined.

## INTRODUCTION

With the increasing use of commercial articulated vehicles, such as tractor-semitrailers and truck-full trailers, in road transport, concerns for their safety in operation have been growing. This has stimulated intensive studies of the dynamic stability of this type of vehicle.

A variety of analytical (simulation) models for predicting the lateral dynamic behaviour of commercial articulated vehicles have been developed in recent years. These models vary greatly in capability, in complexity, in the number of degrees of freedom considered, and in the amount of input data required. For instance, the Phase 4 model developed by the University of Michigan Transportation Research Institute (UMTRI) incorporates up to 71 degrees of freedom and requires up to approximately 2300 lines of input data, dependent upon the vehicle configuration (1)(2). On the other hand, the linear yaw plane model developed also by UMTRI only includes the lateral and yaw motion of the tractor and articulation in the horizontal plane of the other sprung masses of the articulated vehicle, and only requires up to approximately 35 lines of input data. It should also be mentioned that not only a large amount of input data is required for a sophisticated model, but also special measuring equipment is needed to obtain a number of these input parameters. Quite often, even if the equipment is available, the cost of obtaining the required parameters will be considerable.

When a practising engineer wishes to evaluate the lateral dynamic behaviour of a vehicle, he will undoubtedly be faced with the problem of

how to select an appropriate model for a given task. A recent survey has pointed out that while a wide range of simulation models are available, a systematic evaluation of these models, particularly with respect to their areas of application, has been lacking (3). To fill this gap and to provide guidance for the practising engineer to select an appropriate model for a particular task, a detailed comparison of some of the more widely known simulation models, including the linear yaw plane model, the TBS model, the yaw/roll model, and the Phase 4 model developed at UMTRI has been performed.

The steady-state steering response and the lateral dynamic behaviour in lane-change (obstacle avoidance) type manoeuvres of six commercial articulated vehicles with different design features have been predicted using the four UMTRI computer simulation models. A comparison of the predictions made using these models and with a limited amount of experimental data available has been made. In addition, the static roll model developed by UMTRI has been examined, and a parametric sensitivity study has been performed.

The objective of this study is to compare the capabilities and limitations of various simulation models for the evaluation of the directional behaviour of commercial articulated vehicles. The results of the study will provide quantitative information upon which the appropriate areas of application of various models may be defined.

## 1. A BRIEF DESCRIPTION OF THE VARIOUS SIMULATION MODELS USED IN THE STUDY

A brief description of the basic features of the five simulation models used in the study, namely, the linear yaw plane model, the TBS model, the yaw/roll model, the Phase 4 model, and the static roll model developed at UMTRI, is given below (1)(2)(4).

### 1.1 The Linear Yaw Plane Model

The yaw plane model is a linear mathematical model for studying the directional behaviour of multiple articulated vehicles. The model was developed in 1978, originally for the purpose of analyzing the directional behaviour of double-bottom tankers.

Computer programs based on the model permit the evaluation of the following directional properties which are useful for studying a vehicle's directional stability and performance: (1) natural frequency and damping ratios of the natural modes of yaw motion (eigenvalues), (2) transient and steady-turning responses of each articulated unit of an articulated vehicle train, and (3) frequency response (by frequency response we mean the directional response of the vehicle to sinusoidal steer inputs).

At the present time, the computer programs permit the analysis of articulated vehicles which have up to three articulation points and a maximum of 11 axles. The computer codes can be easily expanded to the study of vehicles which have an even greater number of axles and articulation points.

In developing the equations of motion for the linear yaw plane model, the roll dynamics of the vehicle is neglected. Further, the vehicle is assumed to travel at a constant forward velocity. The degrees of freedom permitted in the model are therefore limited to lateral and yawing motion of the tractor and articulation in the horizontal plane of the other sprung masses of the multiple articulated vehicle.

The following are the assumptions made in the process of deriving the equations of motion:

- A. The cornering forces and aligning moments generated at the tire-road interface are assumed to be linear functions of the sideslip angle of the tire.
- B. Articulation angles made by the various elements of the vehicle train are small such that the following approximations hold:  
 $\sin \Gamma_i = \Gamma_i$ ,  $\cos \Gamma_i \approx 1$  (where  $\Gamma_i$ 's are the articulation angles).
- C. The motion of the vehicle takes place on a horizontal surface with uniform friction characteristics.
- D. There are no significant tire forces present in the longitudinal direction (either tractive or braking).
- E. Pitch and roll motions of the sprung masses are small and hence neglected.

- F. All joints are frictionless and articulation takes place about vertical axes.
- G. Steering system dynamics is not included in the model and the steering input is assumed to be given directly to the front wheels.
- H. In the case of tanker trains, the tanker compartments are assumed to be either completely full or completely empty, thereby avoiding sloshing of the liquid.
- I. Each element or unit of the articulated vehicle is assumed to be a rigid body (in the case of liquid filled tanks, all of the liquid is assumed to take part in the yawing motion, i.e., relative motion of the liquid with respect to the walls of the tank is neglected) and the unsprung masses are assumed to be rigidly attached to their respective sprung masses.

Due to the simple manner in which the vehicle is represented in the model, the input data required are moderate and computational costs are very low.

## 1.2 The TBS Model

This model is for predicting the directional response of commercial vehicles to steering inputs and/or braking (4). The simulation consists of two interactive computer programs, one for a straight truck and the other for a tractor-semitrailer. The model for the tractor-semitrailer has four degrees of freedom, namely, the longitudinal velocity, the lateral velocity, and the yaw velocity of the tractor, and the articulation angle of the semi-trailer relative to the tractor. There are no roll or pitch degrees of freedom. Load transfers, both longitudinal and lateral, are taken into consideration and are computed quasi-statically.

In this model, the hitch is assumed to transmit a yaw moment (but not a roll or pitch moment) through the friction in it. The hitch is modelled as a circular plate with a uniform pressure distribution, equal to the static load divided by the area of the plate. The coefficient of friction is assumed to be a constant. For "steel on steel", the value of the friction coefficient at the hitch is taken to be approximately 0.05.

The normal load on each wheel of the vehicle is equal to the sum of the static load and the load transfer (both longitudinal and lateral) taking place at any instant of time.

A simplified model for tandem axles is included. A quasi-static inter-axle load transfer is specified by entering an appropriate load transfer coefficient. The product of this coefficient and the braking force on one side of the tandem axles gives the inter-axle load transfer for that side.

For a free rolling tire, the model developed by Fiala is used to describe the relationship between lateral force and sideslip angle (4). For braking, two linear equations are used to describe the relationship between the braking effort coefficient and longitudinal slip, one for the range from zero to the peak value of the braking effort coefficient, and the other for the range from the peak value to the sliding value (100% skid) of the braking effort coefficient. For combined braking and cornering, an empirical equation is used to describe the functional relationships between the lateral and longitudinal forces and sideslip angle and longitudinal skid.

In the TBS model, antilock braking system can be accommodated. Dual tires are treated as two single tires, each sharing the vertical load on them equally and each yielding the same longitudinal and lateral forces.

Braking is handled in the model by specifying the time history of the attempted brake force for the brakes on either side of each axle. Since each side is considered separately, brake imbalance may be simulated. Steering inputs are entered in a tabular form. Each line of the table consists of the time followed by the average steer angle of the front wheels.

It should be mentioned that the model cannot handle the case when a wheel lifts off the ground. In other words, computations are stopped if this happens. Furthermore, if the articulation angle becomes larger than a user specified value, which may indicate "jackknifing" or "trailer swing", computations are also stopped.

### 1.3 The Yaw/Roll Model

The yaw/roll model was developed for the purpose of predicting the directional and roll response of single and multiple articulated vehicles engaged in steering manoeuvres which approach the rollover condition. It should be noted that the model does not permit the simulation of braking manoeuvres. The model is unique in the sense that it permits the analysis of unconventional vehicle layouts. The equations of motion are developed in such a fashion that it is possible to use the model for simulating vehicles with (1) any number of units and articulation points, (2) any number of placement of wheels and tires, and (3) any of the particular hitch mechanisms and constraints that are used in heavy-duty commercial vehicles.

In its present form, the computer code permits the simulation of vehicles with up to three articulation points (i.e., four sprung masses) and 11 axles. The computer program can be easily expanded to permit the analysis of vehicles with an even larger number of articulation points and axles.

In the model, the forward velocity of the lead unit is assumed to remain constant during the manoeuvre. Hence, each sprung mass is treated as a rigid body with five degrees of freedom: lateral, vertical, yaw, roll and pitch. The axles are treated as beam axles which are free to roll and bounce with respect to the sprung mass to which they are attached.

The following assumptions were made in the process of deriving the equations of motion:

- A. Only steering manoeuvres (in which the input is an angular displacement of the front wheels on the lead unit) are modelled.
- B. The vehicle moves over a horizontal surface possessing uniform frictional characteristics.
- C. The pitch motions of the sprung masses are small such that  $\sin \theta_s = \theta_s$  and  $\cos \theta_s = 1$ .
- D. The relative roll displacements between the sprung masses and the axles remain small, such that  $\sin(\phi_s - \phi_u) = \phi_s - \phi_u$  and  $\cos(\phi_s - \phi_u) = 1$ .
- E. The relative roll motion between the unsprung and sprung masses takes place about roll centres, R, which are located at fixed distances beneath the sprung masses (see Fig. 1.1).
- F. The line of action of the suspension springs remains parallel to the  $k_u$  axis (Fig. 1.1), with only compression and tensile forces being transmitted to the sprung mass. Figure 1.1 shows that the roll centre, R, is free to move in the  $k_u$  direction, such that any force on the axle in the  $j_u$  direction acts on the sprung mass at the roll centre, R. In the case of a suspension consisting of leaf springs, the springs are twisted (about a longitudinal axis) when the axle rolls relative to the sprung mass, with the resulting roll-resisting moment being treated as an auxiliary roll stiffness, KRS (see Fig. 1.1).

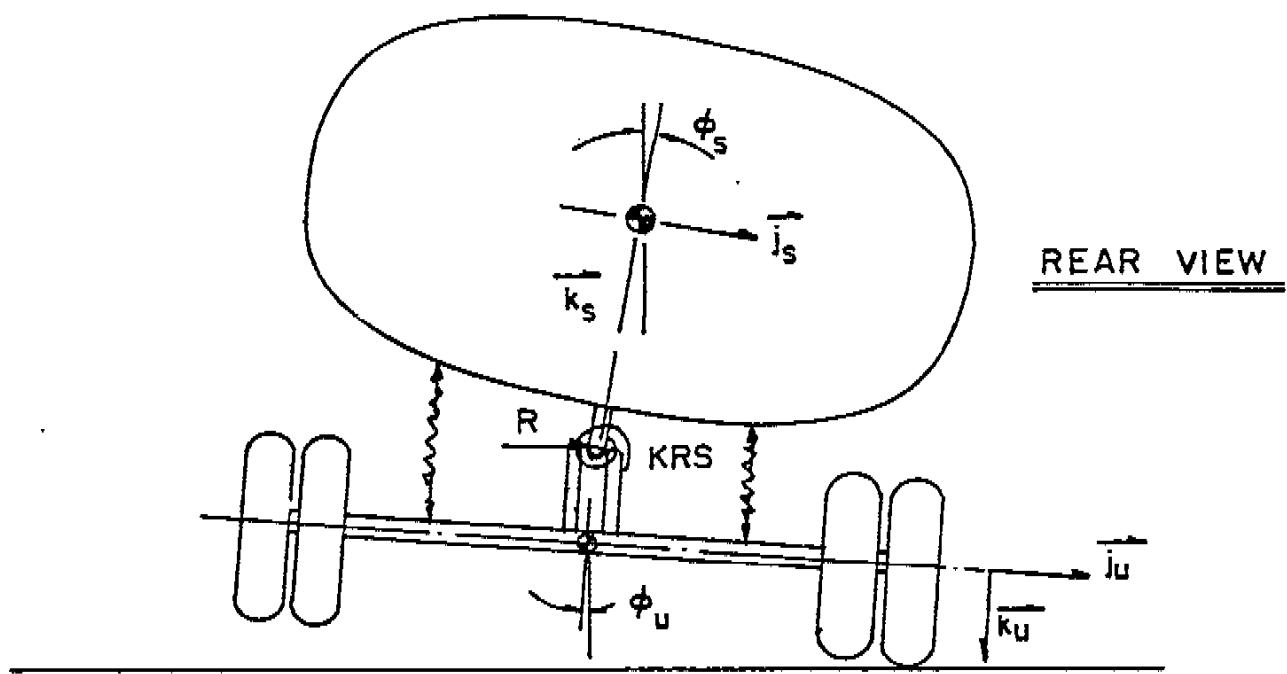


Figure 1.1 Idealized representation of axles and suspension springs

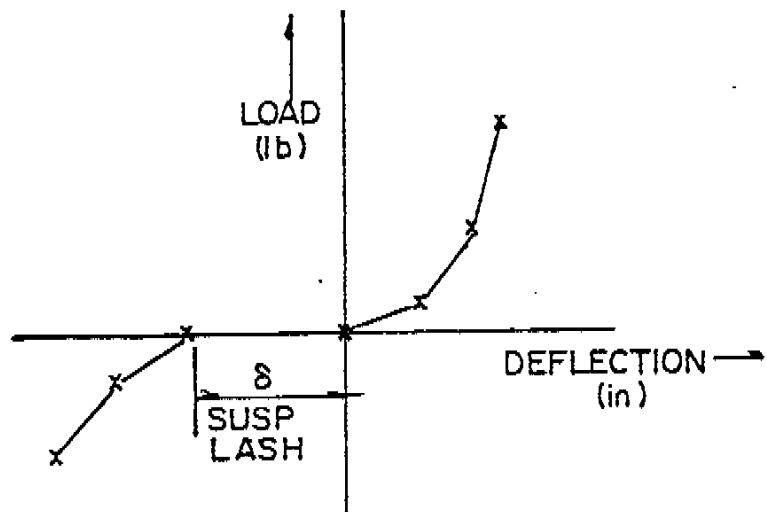


Figure 1.2 Idealized representation of suspension nonlinearities

- G. Nonlinearities in the force-displacement behaviour of a suspension, such as suspension lash, are approximated as shown in Figure 1.2.
- H. The forces acting on each axle are independent of the forces acting on adjacent axles, i.e., interaxle load transfers are neglected.
- I. The cornering force and aligning moment produced by a given tire is a nonlinear function depending only on sideslip angle and vertical load. The influence of wheel inclination (or camber) on lateral force generation has been neglected on the grounds that inclination angles remain small prior to wheels being lifted off the ground in a severe manoeuvre.
- J. The principal axes of inertia of the sprung and unsprung masses coincide with the respective body-fixed co-ordinate systems.

In its present form, the computer program permits the analysis of vehicles which are equipped with any of the four coupling mechanisms used in heavy-duty commercial vehicles, namely, the fifth wheel, inverted fifth wheel, kingpin, and pintle hook. Note that the "fifth wheel" and the "inverted fifth wheel" permit the lead and the trailing units to yaw and pitch with respect to one another, but are "stiff" in roll. On the other hand, the so-called "kingpin" connection permits only yaw motions between the lead and the trailing units. In the case of the "pintle hook", the trailing unit can roll, bounce, yaw, and pitch with respect to the lead unit.

#### 1.4 The Phase 4 Model

This model was developed in 1980 for simulating the braking and steering dynamics of trucks, tractor-semitrailers, doubles, and triples combinations. The goal in developing this program was to consolidate principal features of all existing computer models into a single program. In short, the Phase 4 model represents UMTRI's latest thinking in computer modelling of the braking and steering response of commonly used commercial vehicles.

The Phase 4 program is a time-domain mathematical simulation of a truck/tractor, a semitrailer, and up to two full trailers. The motions of the vehicles are represented by differential equations derived from Newtonian mechanics, that are solved for successive time increments by digital integration.

The program is written in a generalized fashion to allow simulation of a large number of vehicle configurations. The first vehicle is the power unit and may be a truck or tractor, both of which may carry payload. As a single unit with no payload, it is equivalent to an empty truck or bobtail tractor. With payload, it is a truck, which, with a semitrailer simulates a car hauler, dromedary tractor, etc. The second unit is always a semitrailer (i.e., the current model does not include a truck with full trailer). The third and fourth units are full trailers consisting of semi-trailers on either a fixed or converter dolly. Separate payload may be specified for each trailer.

The mathematical model incorporates up to 71 degrees of freedom. The number of degrees of freedom are dependent on the vehicle configuration and derive from the following:

- Six degrees of freedom (three translational and three rotational) for the truck/tractor sprung mass
- Three degrees of freedom for the semitrailer (the three other degrees of freedom of the semitrailer are effectively eliminated by dynamic constraints at the hitch)
- Five degrees of freedom for each of the two full trailers allowed
- Two degrees of freedom (vertical and roll) for each of the 13 axles allowed
- A wheel rotational degree of freedom for each of the 26 wheels allowed

The motion of each of the sprung masses is determined from the summation of forces and moments arising from the tires (acting through the unsprung mass and suspension), gravity, and the hitch point constraints. Small roll and pitch angle assumptions are made in deriving equations so that the simulation can be validly applied only up to a manoeuvre limit at which vehicle roll-over occurs.

The Phase 4 model can be used to simulate the following vehicle configurations:

Straight truck, empty and loaded  
Bobtail tractor  
Tractor-semitrailer (3 to 5 axles), empty and loaded  
Tractor-Semitrailer-full trailer (5 to 9 axles), empty and loaded  
Tractor-semitrailer-full trailer-full trailer (7 to 13 axles),  
empty and loaded.

For simulation of braking performance, the program incorporates state-of-the-art representation of truck air brake systems, antilock braking systems, and tire-road friction models. Typical examples of braking studies for which it can be, or has been used, are:

- A. Stopping distance performance
- B. Effects of brake timing
- C. Dynamic behaviour in braking
- D. Comparisons of antilock wheel control logic
- E. Influence of tire-road friction coupling
- F. Split friction surfaces
- G. Brake proportioning .
- H. Tandem-axle effects on braking limits.

For simulation of cornering performance behaviour, the program incorporated state-of-the-art representations of truck tire lateral force

characteristics (with roll-off effects during combined braking), and vehicle suspension properties of significance to cornering behaviour.

Typical examples of studies involving cornering are as follows:

- A. Understeer/oversteer properties of commercial vehicles
- B. Determining cornering limits
- C. Assessing tandem-axle effects on cornering
- D. Jackknife prediction
- E. Effects of suspension properties on cornering and cornering limits
- F. Accident simulation.

In addition to the above, the program can be operated open-loop (defined steer angle inputs) or closed-loop (defined path input), and on roads of specified grade or cross-slope.

The Phase 4 program is uniquely applicable in studies in which the influence of the following items are to be considered in detail:

- A. Spring force/deflection characteristics (hysteresis and free-play)
- B. Brake "fade" - brake temperature
- C. Brake hysteresis
- D. Load-leveler action in tandem suspensions
- E. Brake proportioning algorithms
- F. Steering system compliance (inputs at the steering wheel)
- G. Frame torsional stiffness.

## 1.5 The Static Roll Model

The static roll model was developed for the purpose of calculating the rollover threshold of articulated vehicles during steady turning manoeuvres. The dynamics of roll motion are not included in the model. Instead, the roll response in a steady turn is computed by repeatedly solving, for small increments of roll angle, a set of equations which describe the static equilibrium of the vehicle in the roll plane.

Figure 1.3 shows the side view of an example tractor-semitrailer as presented in the roll plane model. As shown in the Figure, axles with similar suspension properties are grouped together in such a fashion that all the axles on the vehicle are represented by a set of three composite axles. The composite axles are:

- 1) tractor front axle
- 2) tractor rear axle (either a single or tandem axles are combined and represented by one axle)
- 3) trailer axle (all the axles on the trailer are combined and represented by one axle).

The torsional compliance that exists in the tractor frame is represented by a torsional spring element, K<sub>FR</sub> (Fig. 1.3). Similarly, the structural compliance in the body of the trailer and the fifth wheel arrangement are lumped together and represented by another torsional element, K<sub>5</sub>. The idealized representation of the friction (CFR) present

in the tractor frame and the separation of the fifth wheel plates (LASH5) are shown in Figures 1.4 and 1.5 respectively.

In the model, the relative motion between the sprung mass and the axles is assumed to take place about roll centres which are at fixed distances beneath the sprung masses. As shown in Figure 1.6, the suspension springs are assumed to remain parallel to the  $\vec{k}_{u_i}$  axes of the axles and transmit only compressive or tensile forces.

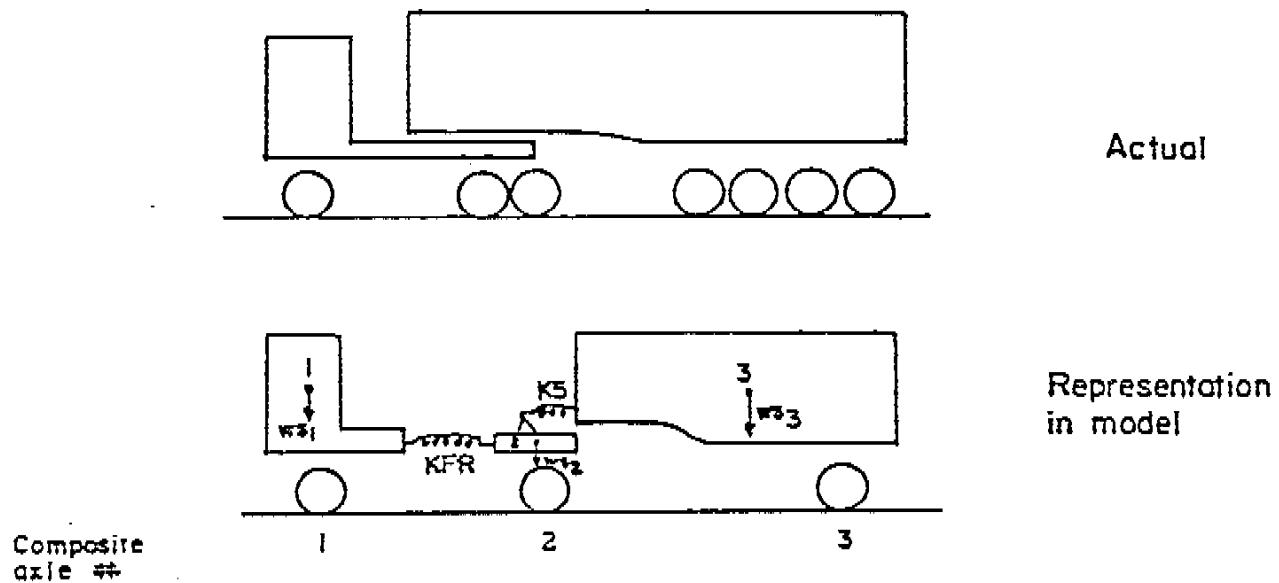


Figure 1.3 Side view of the representation of axles in the static roll model

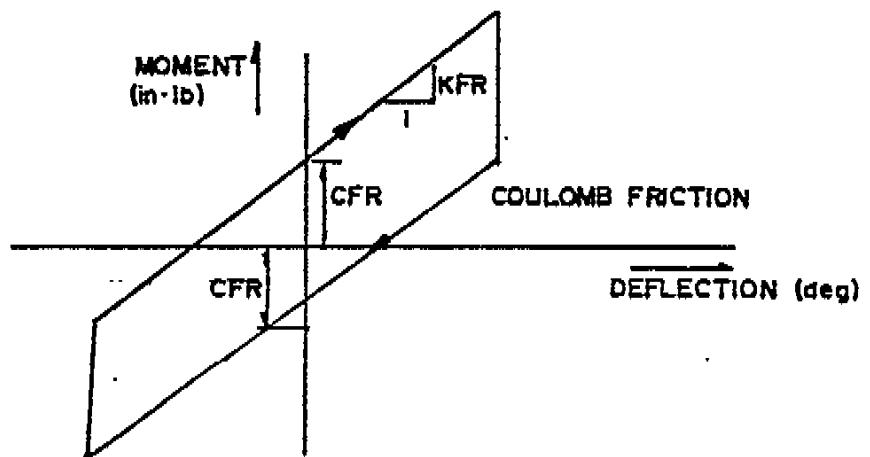


Figure 1.4 Idealized representation of tractor frame compliance

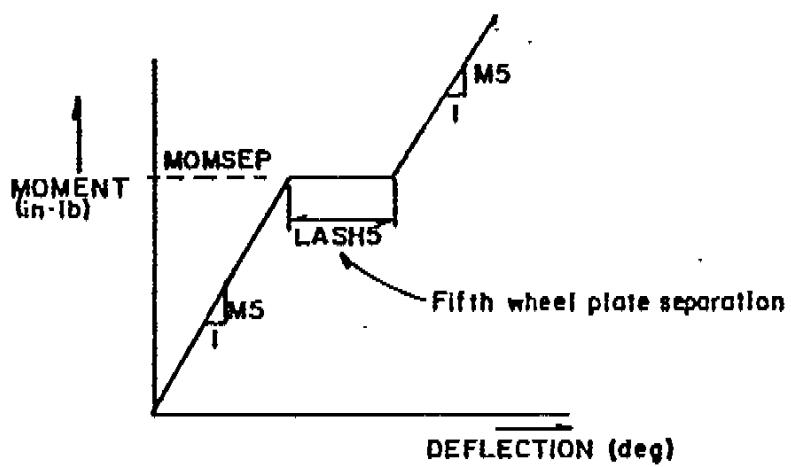


Figure 1.5 Idealized representation of fifth wheel compliance

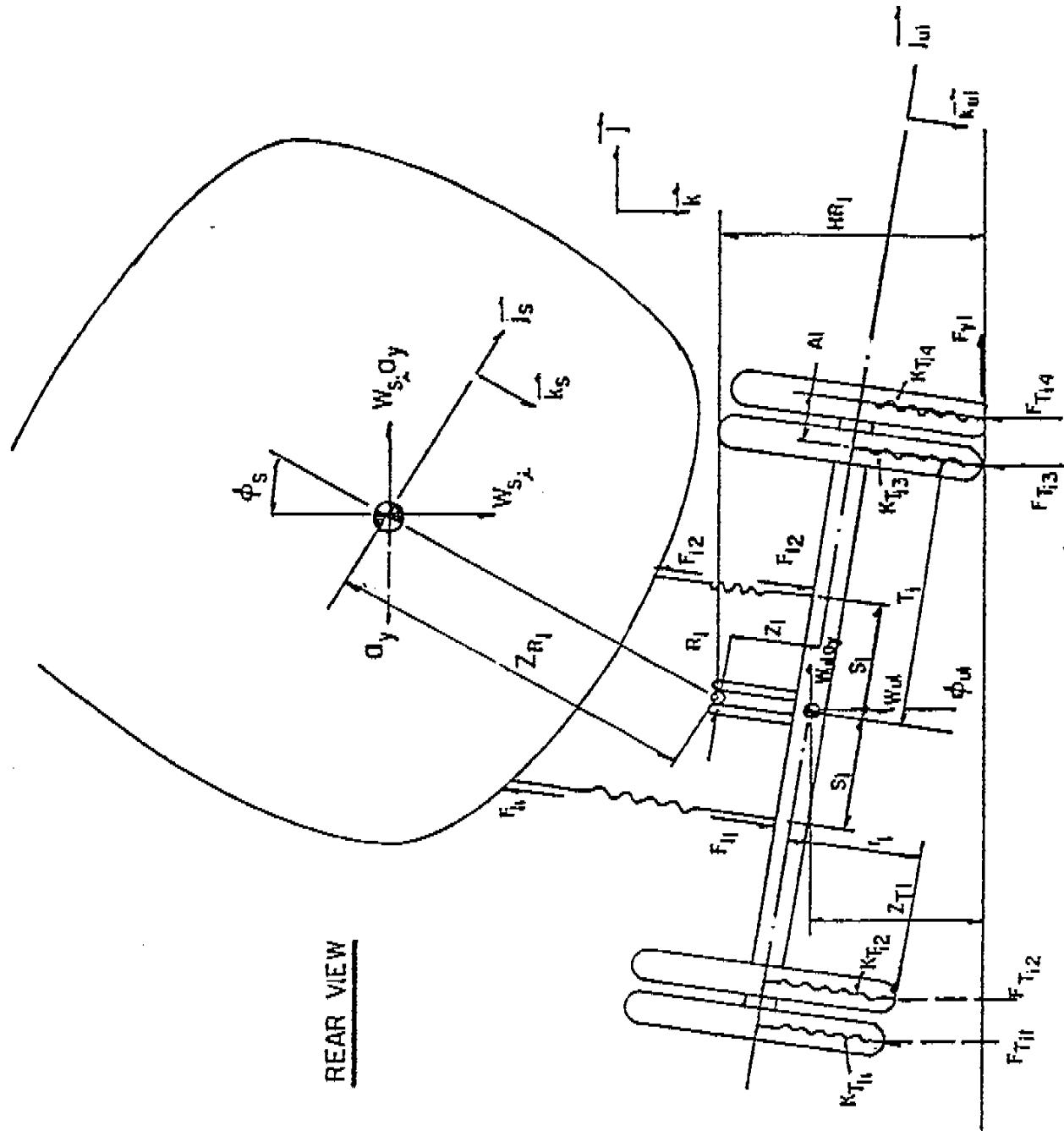


Figure 1.6

The roll centres are permitted to slide freely (with respect to the axles) along the  $\vec{k}_{u_i}$  axes. All axle forces which act in a direction parallel to the  $\vec{k}_{u_i}$  axes are taken up by the suspension springs, while all axle forces acting along the  $\vec{j}_{u_i}$  axes are assumed to act through the roll centre,  $R_i$ .

Suspension nonlinearities such as backlash and progressively hardening suspension springs are presented by a tabular load-deflection input format. The suspension forces and local spring rates at any given deflection are then computed by linear interpolation. Figure 1.7 shows the representation of a suspension spring in the model.

The other simplifying assumptions implicit in the derivation of the equations are listed below.

- A. The roll angles of the sprung mass and the axles are small, such that the small angle assumptions  $\sin(\phi) = \phi$  and  $\cos(\phi) = 1.0$  hold.
- B. The articulation angles are small so that the influence of articulation angle on roll response can be neglected.
- C. The total vertical load carried by each composite axle is assumed to remain constant during the rollover process. In order to accommodate any pitching motion that might take place during rollover, the sprung masses are permitted to take up different vertical deflections at each of the three axle locations.

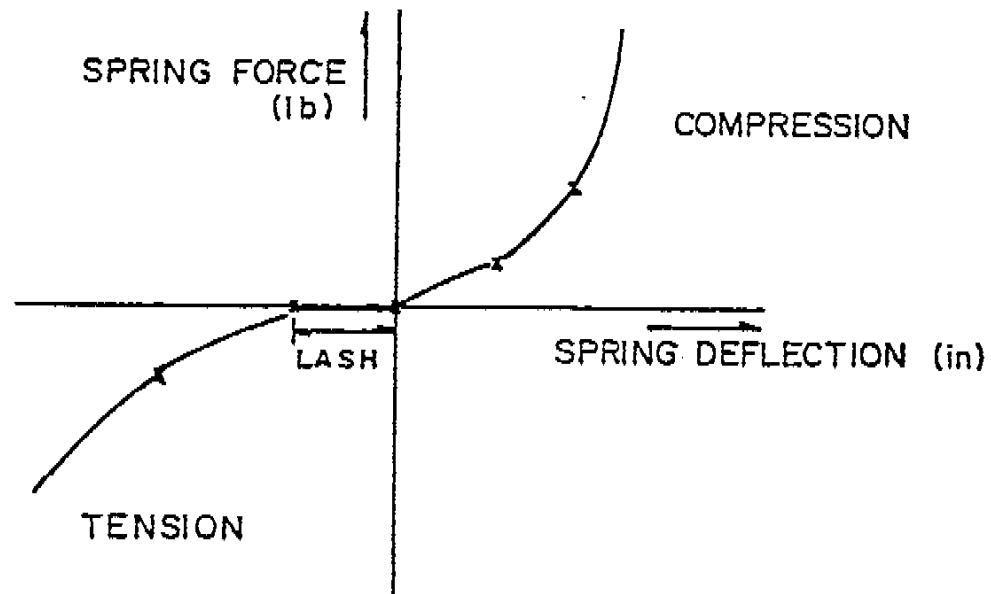


Figure 1.7 Idealized representation of suspension spring characteristics

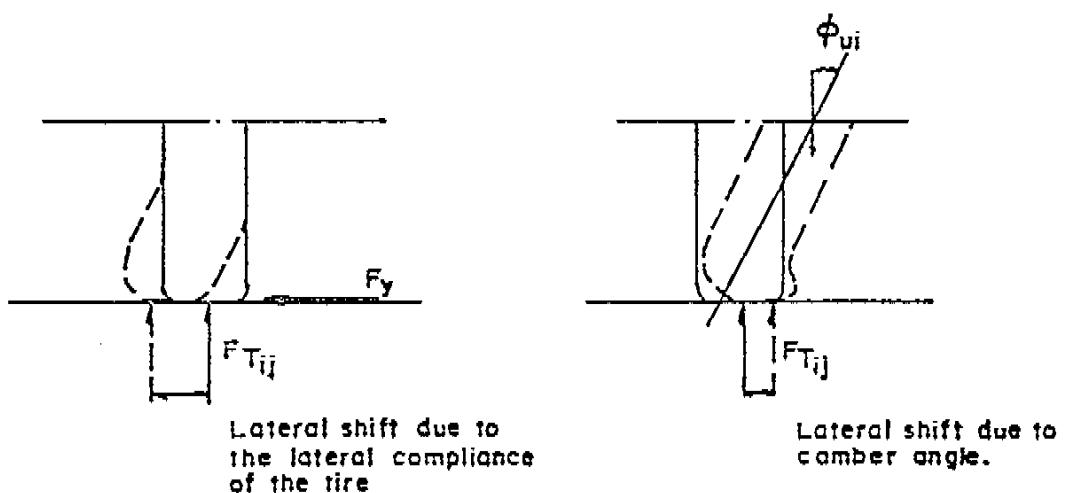


Figure 1.8 The effect of lateral compliance and camber angle on the centroid of the normal pressure distribution at the tire/road interface

D. The vertical load carried by each tire is assumed to act through the midpoint of its tread width. As shown in Figure 1.8, the effect of camber angle and the effect of the lateral compliance of the tire tend to have opposing effects on the lateral translation of the centroid of the normal pressure distribution at the tire-road interface. Both of these effects are small and tend to cancel out. In order to keep the analysis simple, the lateral translation of the normal load is neglected.

The model contains a fairly detailed description of suspension and tire properties, such that the accurate prediction of steady-turning rollover threshold is possible. Computational costs are low due to the fact that no roll dynamics is included in the model. Moreover, a single computer run is sufficient for computing the rollover threshold, as well as the roll response of the vehicle over the entire range of lateral acceleration levels up to the rollover threshold.

## 2. VEHICLE CONFIGURATIONS USED IN THE SIMULATION STUDY

To quantitatively compare their predictive ability, the computer simulation models described above were used to predict the responses of six different commercial articulated vehicles in two types of turning manoeuvres, namely steady-state turning and lane-change (obstacle avoidance) manoeuvres.

The six vehicle configurations selected include five tractor-semi-trailers and one five-axle double with 27 ft. trailer (that is, a two-axle tractor with a single-axle semitrailer and a two-axle full trailer). The basic parameters of the vehicles are summarized in Table 2.1.

A. Vehicle Configuration 1 is a fully-loaded three-axle tractor with a two-axle semitrailer. The basic parameters are similar to those of an International Harvester three-axle tractor (COF 4000 D) with a Fruehauf 45 - foot van-type semitrailer (FG8-F2-45') described in reference (5). A complete set of input data for the four computer simulation models, namely the Phase 4 model, the yaw/roll model, the TBS model, and the linear yaw plane model, is given in Appendix A.

B. Vehicle Configuration 2 is similar to Vehicle Configuration 1, but with the following modifications:

- a) The tractor frame has been stiffened, and its torsional stiffness has been increased from 20,000 to 120,000 in-lb/deg.
- b) An auxiliary roll-stiffening device ("sway bar") has been installed on the tractor front suspension. The device provides an additional roll stiffness of 109,385 in-lb/deg.

Table 2.1

## Basic Parameters of Various Vehicle Configurations

Vehicle Configuration	1	2	3	4	5	6
A. Tractor	3-axle	3-axle	3-axle	3-axle	3-axle	2-axle
Wheelbase	{in}	142.0	142.0	144.0	144.0	120.0
Curb weight on front suspension	{lb}	8,898	8,898	8,960	8,960	8,960
Curb weight on rear suspension	{lb}	7,118	7,118	6,540	6,540	4,240
Sprung mass C.G. height	{in}	39.7	39.7	44.0	44.0	44.0
Sprung mass roll moment of inertia	(in-lb-sec <sup>2</sup> )	18,166.6	18,166.6	15,000	15,000	15,000
Sprung mass pitch moment of inertia	(in-lb-sec <sup>2</sup> )	69,955.0	69,955.0	75,000	75,000	75,000
Sprung mass yaw moment of inertia	(in-lb-sec <sup>2</sup> )	69,955.0	69,955.0	75,000	75,000	75,000
Fifth wheel location, ahead of rear suspension centre (in)		0.0	0.0	14.35	14.35	0
Fifth wheel height (in)		48.5	48.5	48.0	48.0	48.0
Frame torsional stiffness (in-lb/deg)		20,000	120,000	50,000	50,000	50,000
Roll centre height of front suspension (in)		24.55	24.55	23.0	23.0	23.0
Auxiliary roll stiffness of front suspension (in-lb/deg)		0	109.385	1,500	0	0
Roll centre height of rear suspension (in)		22.0	22.0	29.0	29.0	29.0
Auxiliary roll stiffness of leading tandem axle (in-lb/deg)		0	0	6,000	0	0
Auxiliary roll stiffness of trailing tandem axle (in-lb/deg)		78,000	78,000	6,000	78,000	78,000
Front tires	Type A	Type A	Type C	Type C	Type C	Type C
Rear tires	Type A	Type A	Type C	Type C	Type C	Type C

Table 2.1

## Basic Parameters of Various Vehicle Configurations

	Vehicle Configuration	1	2	3	4	5	6
B. Semitrailer		Tandem Axle	Single Axle				
Wheelbase	(in)	410.0	410.0	432.0	432.0	432.0	252.0
Kingpin static load	{lb}	6,252.3	6,252.3	4,500	4,500	4,500	2,250
Curb weight on rear suspension	{lb}	11,068.7	11,068.7	7,500	7,500	7,500	3,750
Sprung mass C.G. height	(in)	69.0	69.0	60.0	60.0	60.0	60.0
Sprung mass roll moment of inertia	(in-lb-sec <sup>2</sup> )						
Sprung mass pitch moment of inertia	(in-lb-sec <sup>2</sup> )						
Sprung mass yaw moment of inertia	(in-lb-sec <sup>2</sup> )						
Payload weight	{lb}	789,869	789,869	750,000	750,000	750,000	93,750
Payload C.G. ahead of rear suspension centre	{in}	40,600	40,600	52,500	52,500	52,500	25,800
Payload C.G. height	(in)	64.5	64.5	85.0	85.0	85.0	85.0
Payload roll moment of inertia	(in-lb-sec <sup>2</sup> )						
Payload pitch moment of inertia	(in-lb-sec <sup>2</sup> )						
Payload yaw moment of inertia	(in-lb-sec <sup>2</sup> )						
Roll stiffness of fifth wheel	(in-lb/deg)						
Roll centre height							
Auxiliary roll stiffness of leading tandem axle (in-lb/deg)	Type B	0	0	10,000	0	0	10,000
Auxiliary roll stiffness of trailing tandem axle (in-lb/deg)	Type C	0	0	10,000	0	0	Type C
Tires							

Table 2.]

## Basic Parameters of Various Vehicle Configurations

Vehicle Configuration	1	2	3	4	5	6
C. Trailer						
Distance from dolly suspension to pintle hook (in)	80.0					
Turntable location ahead of dolly suspension						
Turntable centre (in)	0					
Wheelbase (in)	48.0					
Curb weight on front suspension (lb)	252.0					
Curb weight on rear suspension (lb)	4,250					
Sprung mass C.G. height (in)	4,250					
Sprung mass roll moment of inertia (in-lb-sec <sup>2</sup> )	60.0					
Sprung mass pitch moment of inertia (in-lb-sec <sup>2</sup> )	36,000					
Sprung mass yaw moment of inertia (in-lb-sec <sup>2</sup> )	115,000					
Payload weight (lb)	115,000					
Payload C.G. ahead of rear suspension centre (in)	26,500					
Payload C.G. height (in)	126.0					
Payload roll moment of inertia (in-lb-sec <sup>2</sup> )	85.0					
Payload pitch moment of inertia (in-lb-sec <sup>2</sup> )	65,000					
Payload yaw moment of inertia (in-lb-sec <sup>2</sup> )	375,000					
Centre height of front suspension (in)	375,000					
Auxiliary roll stiffness of front suspension (in-lb-deg)	29.0					
Roll centre height of rear suspension (in)	10,000					
Auxiliary roll stiffness of rear suspension (in-lb/deg)	29.0					
Tires						

The stiffening of the tractor frame and the increasing of the roll stiffness of the front suspension of the tractor modify the roll moment distribution between the tractor front and rear suspensions in a turning manoeuvre. High percentage of the roll moment will be supported by the front suspension, which causes more lateral load transfer from the inside tire to the outside tire on the tractor front axle. This produces an understeer effect for the tractor. Therefore, the increasing of the frame torsional stiffness and the roll stiffness of the front suspension of the tractor will result in a significant change in the handling behaviour of the vehicle.

It should be pointed out that although the frame stiffener and the auxiliary roll-stiffening device on the front suspension employed to achieve significant changes in roll moment distribution would never be used in practice, they do provide changes in vehicle handling characteristics suitable for studying the influence of altering the roll moment distribution and for evaluating the ability of various simulation models to predict these influences.

A complete set of input data for the four computer models for simulating this vehicle configuration is given in Appendix B.

C. In comparison with Vehicle Configuration 1 and 2, Vehicle Configuration 3 has different loading conditions, and is equipped with a different type of tire. Furthermore, the fifth wheel is located 14.35 in. ahead of the rear suspension centre of the tractor. These factors combined will make the handling behaviour of this vehicle configuration different from that of Vehicle Configurations 1 and 2.

D. Vehicle Configuration 4 is similar to Vehicle Configuration 3, but with the following modifications:

- a) Auxiliary roll-stiffening devices on the tractor front suspension and on its leading tandem axle suspension have been removed. Consequently, the auxiliary roll stiffness of the two suspensions are reduced to zero.
- b) A different auxiliary roll-stiffening device is installed on the trailing tandem axle of the tractor and its roll stiffness is increased to 78,000 in-lb/deg.

These two factors combined produce an oversteer effect for the tractor.

A complete set of input data for the four computer models for simulating this vehicle configuration is given in Appendix D.

E. Vehicle Configuration 5 is similar to Vehicle Configuration 4, with the exception that the fifth wheel centre is located at the centre of the rear suspension of the tractor. The rearward shift of the fifth wheel centre, as compared with that in Vehicle Configuration 4, tends to produce an oversteer effect for the tractor.

A complete set of input data for the computer models for simulating this vehicle configuration is given in Appendix E.

F. Vehicle Configuration 6 is a five-axle double with 27 ft trailers.

- It consists of a two-axle tractor, a single axle semitrailer and a two-axle full trailer. This vehicle configuration is included in the study

to provide an additional case for evaluating the predictive ability of the various computer simulation models. It should be mentioned that only the linear yaw plane model, the yaw/roll model, and the Phase 4 model were used in the simulation of this vehicle configuration, as the TBS model was designed solely for simulating the tractor-semitrailer.

Three sets of input data for the three computer models for simulating this vehicle configuration are given in Appendix F.

As can be seen from Table 2.1, three types of tire, referred to as Type A (Firestone Rib 10.00 x 22F), Type B (Freuhauf Rib 10.00 x 20F), and Type C, are used in the study. The variations of the cornering (lateral) force and aligning torque with sideslip angle for the three types of tire are shown in Figs. 2.1, 2.2, and 2.3, respectively. It can be seen that Types A and B have lower concerning stiffness (or the equivalent) than Type C. The aligning torque of Type A is higher than those of Types B and C at sideslip angles less than approximately  $8^{\circ}$  with normal load of 9,000 lb, and at sideslip angles less than approximately  $10^{\circ}$  with normal load of 6,000 lb. Types B and C have similar aligning torque-sideslip angle characteristics at normal loads of 6,000 and 9,000 lb.

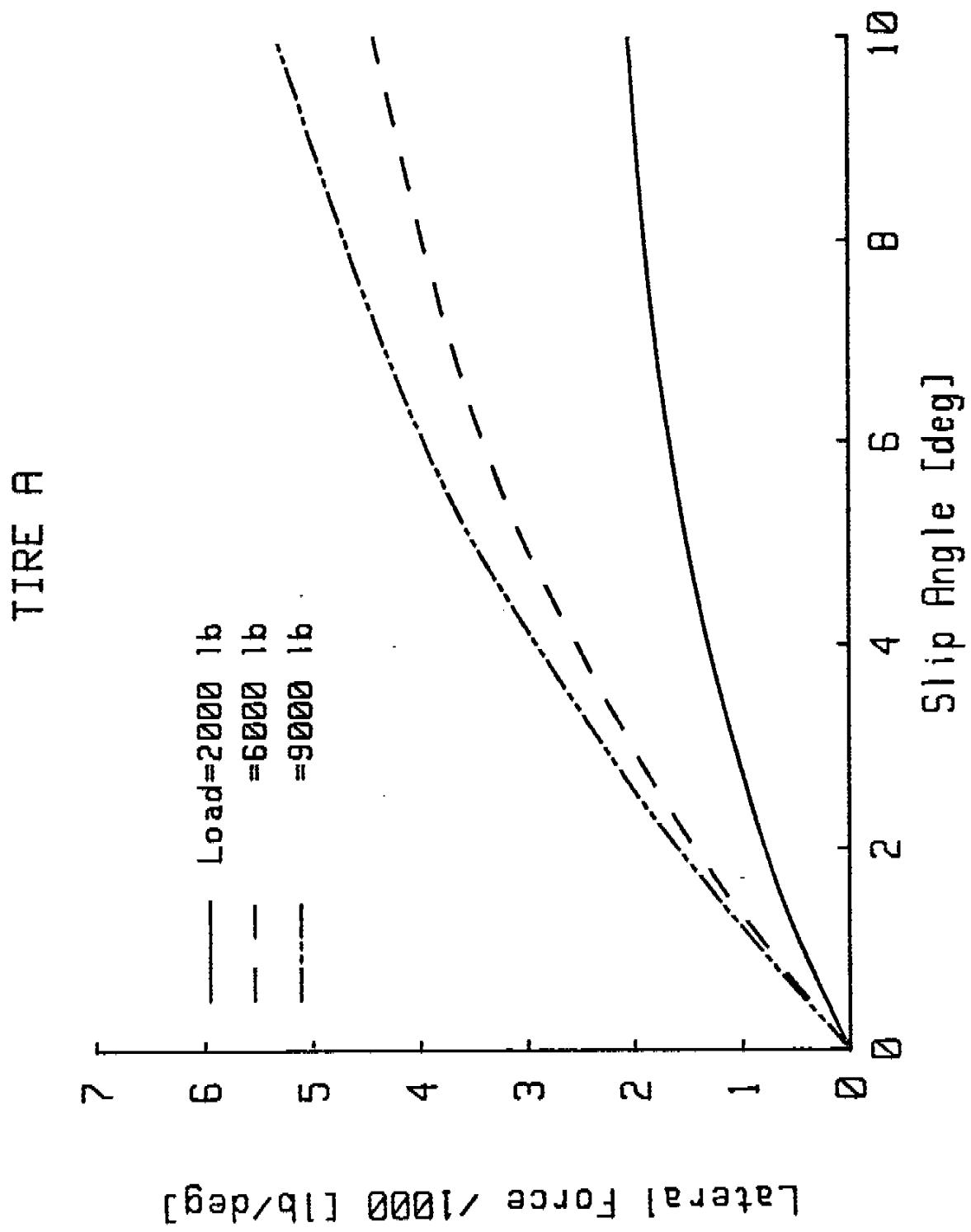


Fig. 2.1 (a) Firestone 10.00 x 22F

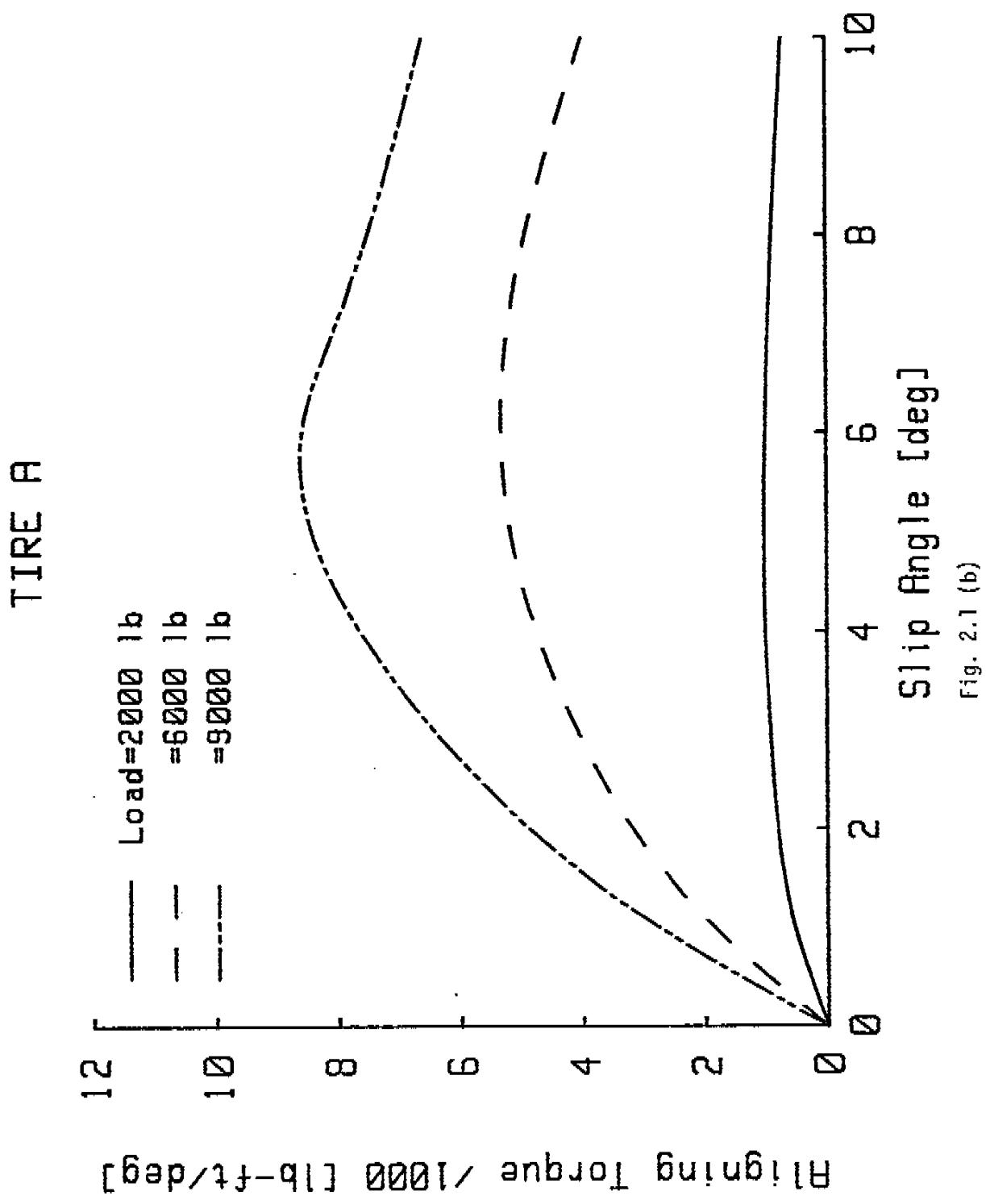


Fig. 2.1 (b)

TIRE B

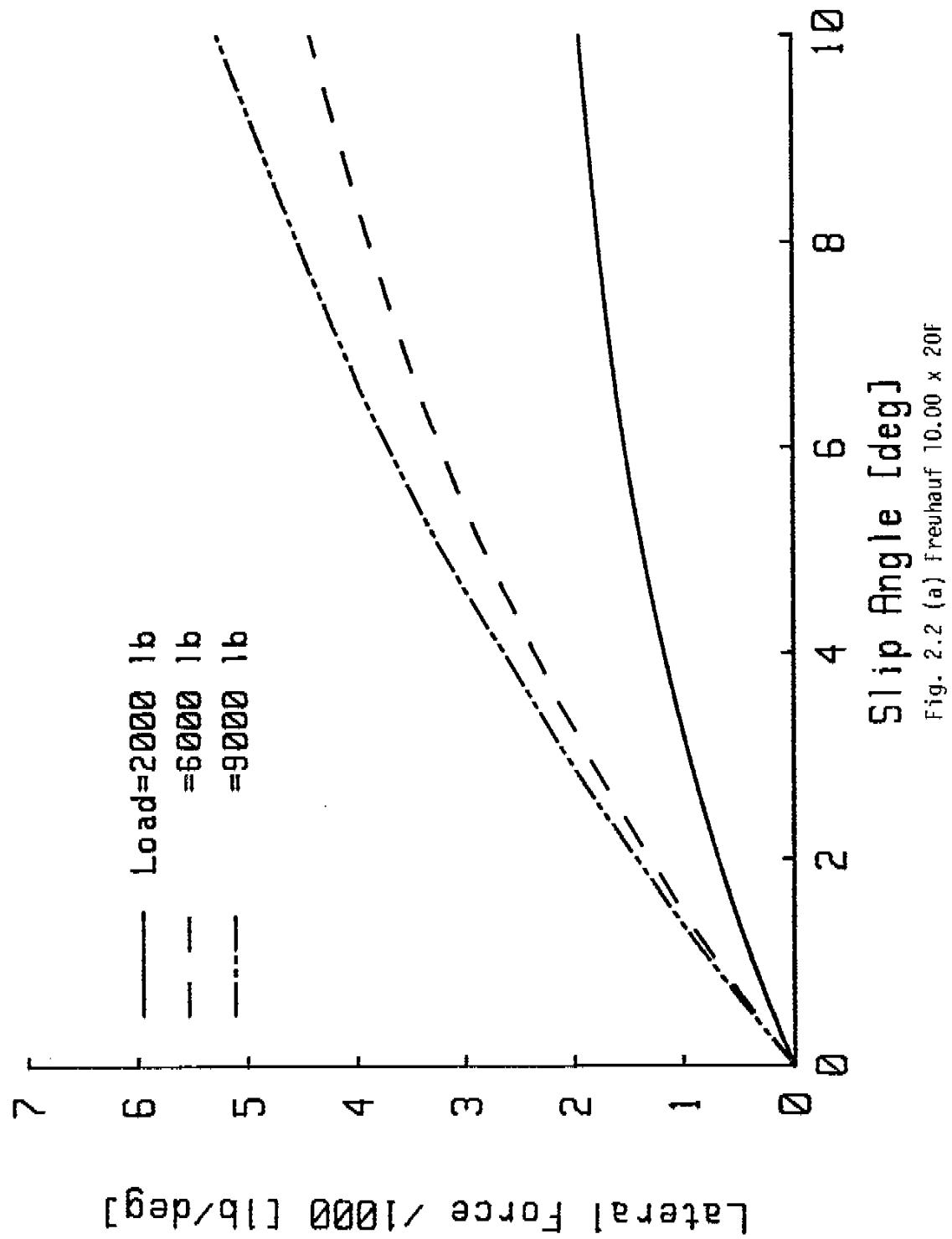
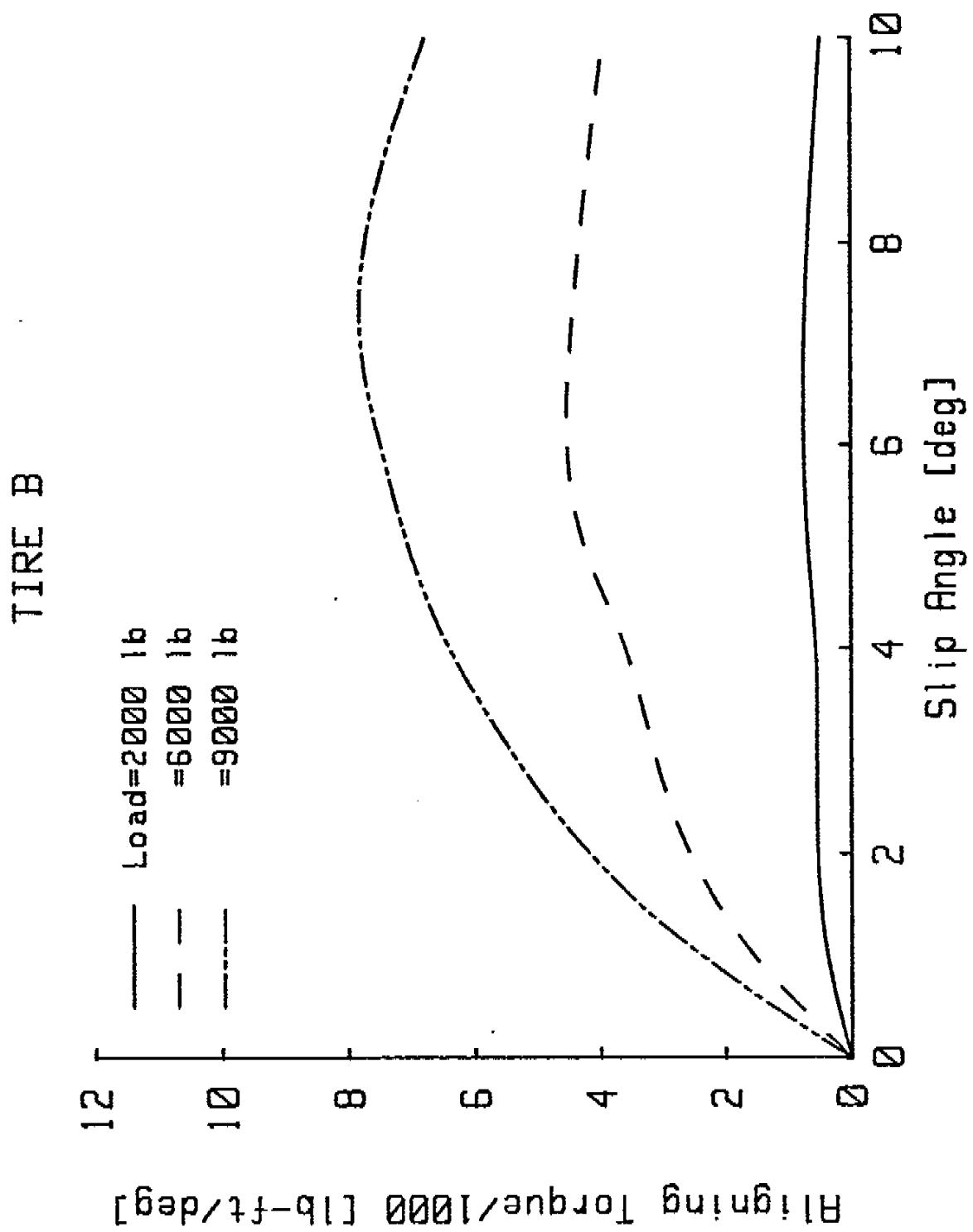


Fig. 2.2 (b)



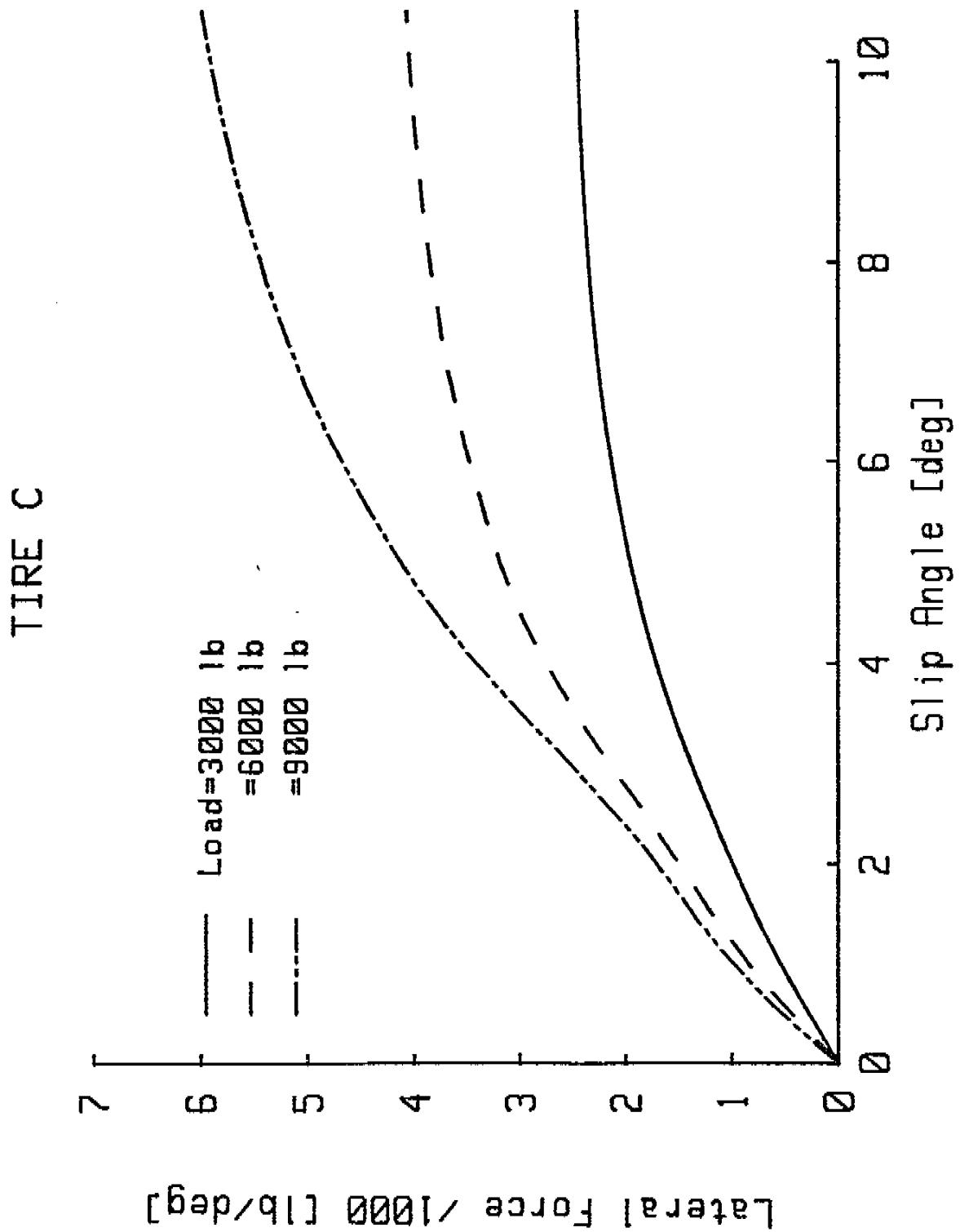


Fig. 2.3 (a)

TIRE C

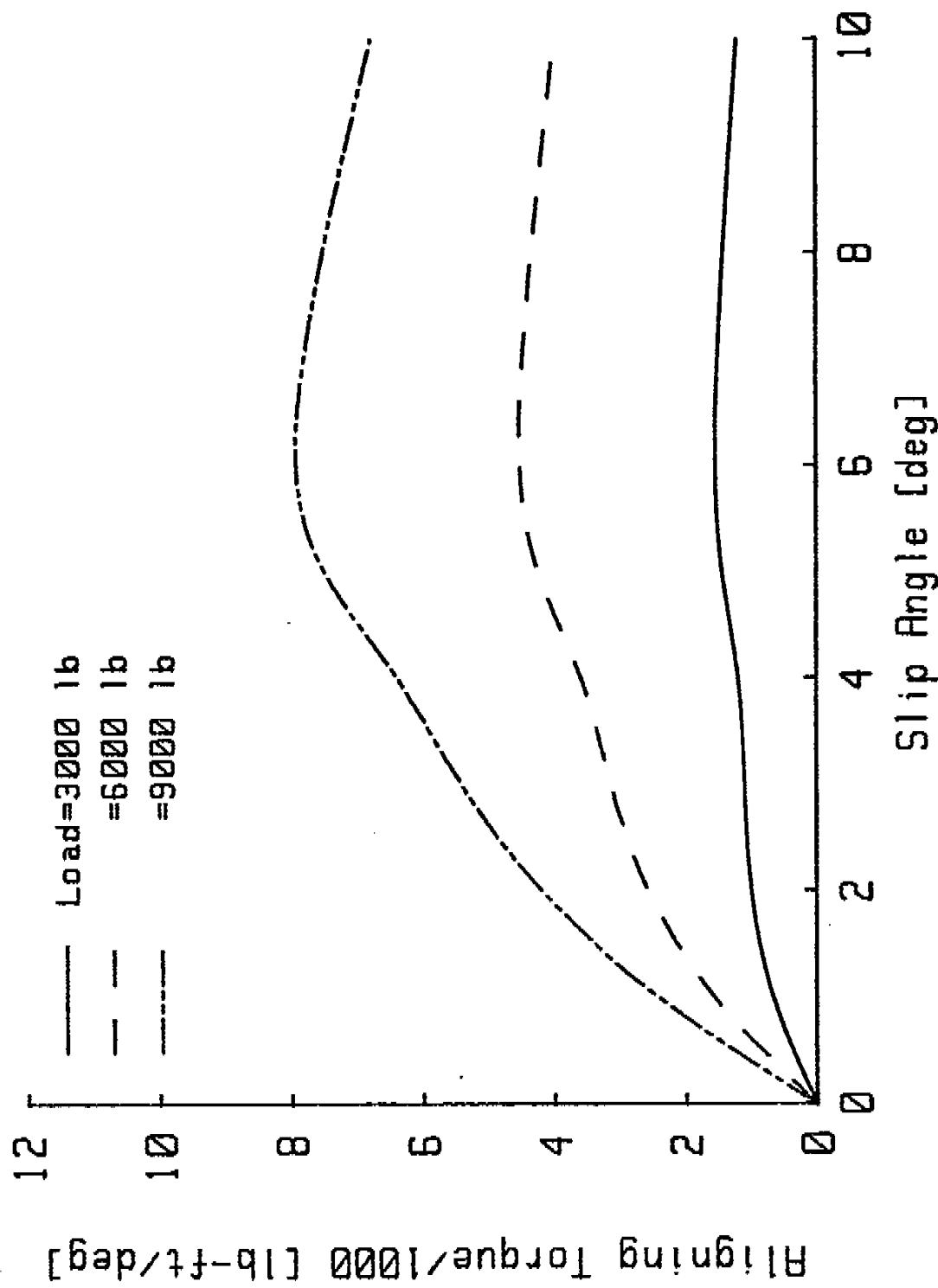


Fig. 2.3 (b)

It has been shown that among other factors the characteristics of the tires and their placement on the vehicle, the distribution of roll stiffness among the suspensions, the location of the fifth wheel, and the location of the centre of gravity of the payload have significant effects on the handling of an articulated vehicle. The six vehicles and the three types of tire selected in this study, therefore, represent a range of vehicle configurations that are expected to have substantially different lateral dynamic behaviour in both the linear and nonlinear regimes. These will provide a wide range of conditions under which the predictive ability of the various computer simulation models may be evaluated and compared quantitatively.

### 3. A COMPARISON OF THE CAPABILITIES OF VARIOUS COMPUTER SIMULATION MODELS IN PREDICTING STEADY-STATE STEERING RESPONSES

The steady-state steering responses of the six vehicle configurations described in the preceding section were predicted using the four computer simulation models, that is, the linear yaw plane model, the TBS model, the yaw/roll model, and the Phase 4 model. In the simulations, ramp inputs of front-wheel steering angle as shown in Fig. 3.1 were applied. After the response of the vehicle reached a steady-state, the lateral acceleration and yaw rate of the tractor as functions of the front-wheel steering angle were determined, and the handling diagram for the tractor was plotted (5)(6). The handling diagram illustrates the relationship between the lateral acceleration of the tractor  $a_y$  and the parameter  $(\ell/R - \delta)$ , where  $\ell$  is the wheelbase of the tractor,  $R$  is the turning radius, and  $\delta$  is the front-wheel steering angle. The turning radius can be derived from the ratio of the yaw rate,  $r$ , to forward speed of the vehicle,  $V$ . The steady-state steering behaviour of the vehicle (neutral steer, understeer, or oversteer) can then be determined from the handling diagram. If the slope of the  $a_y$  versus  $(r\ell/V - \delta)$  curve is negative, then understeer behaviour is indicated. If the slope of the curve is positive, then the vehicle exhibits oversteer characteristics. On the other hand, if the slope of the curve is infinite, neutral steer is indicated.

The steering responses of the six vehicle configurations in steady-state turns predicted using the four computer simulation models are given below.

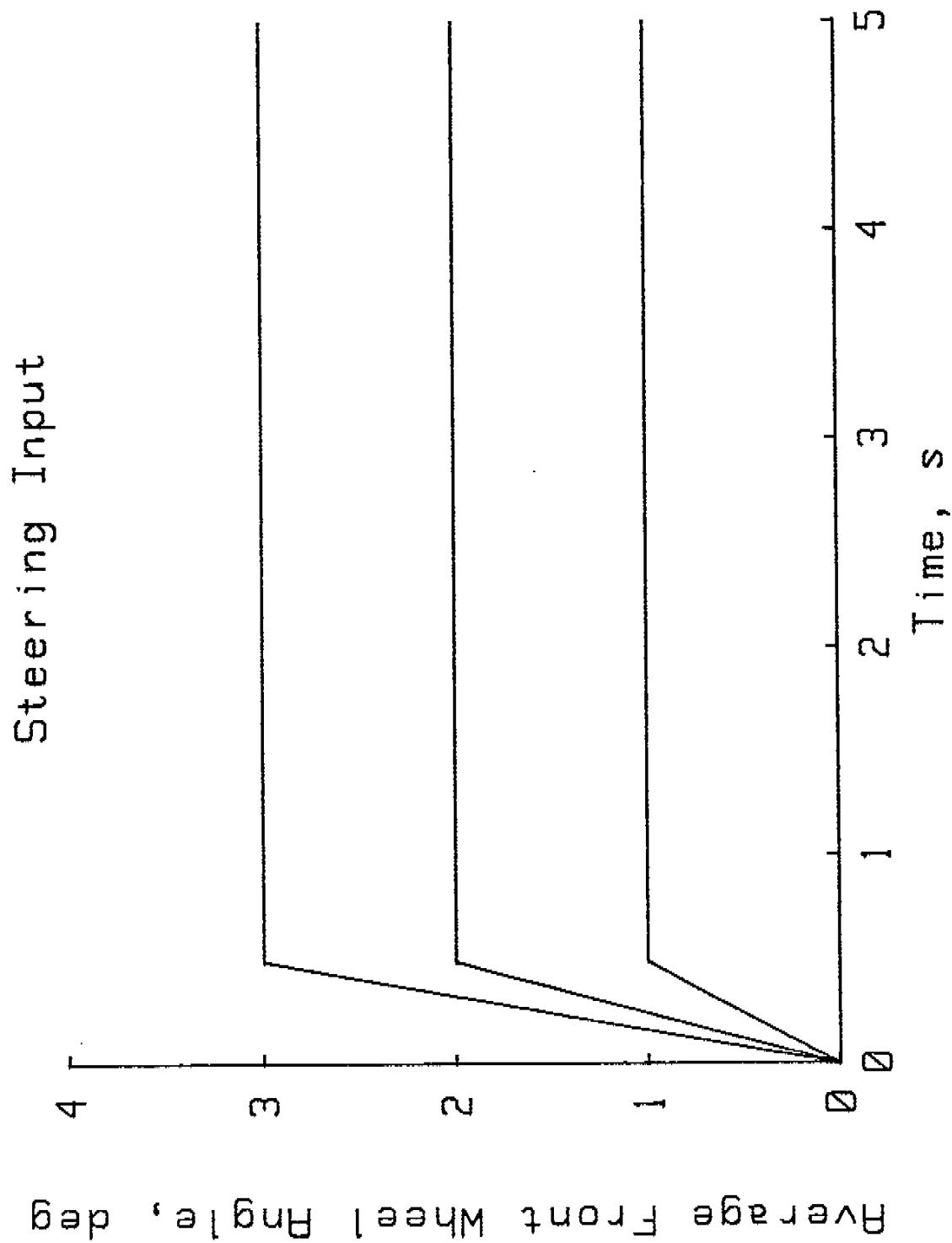


Figure 3.1 Ramp Input of Front-wheel Steering Angle Used in the Simulations

### 3.1 Vehicle Configuration 1

The steering responses of this vehicle configuration in steady-state turns at a forward speed of 43 mph (63.07 ft/s) on a dry, smooth asphalt surface were predicted using the four computer simulation models. The predicted relationships between the lateral accelerations of the tractor and the front-wheel steering angle, together with the measured data reported in reference (5), are shown in Fig. 3.2. The predicted yaw rates of the tractor as a function of the front-wheel steering angle, together with the measured ones, are shown in Fig. 3.3. Based on the predicted lateral accelerations and yaw rates of the tractor, a handling diagram is drawn as shown in Fig. 3.4. For comparison, the measured data are also shown. It should be noted that the square symbol in the figure represents the values calculated from the measured yaw rates of the tractor shown in Figs. 3.2 at a forward speed of 63.07 ft/sec, whereas the triangular symbol represents the values taken from reference (5). All the measured data shown in Figs. 3.2, 3.3 and 3.4 are for left-turn manoeuvres. The predicted values of yaw rate,  $\dot{r}$ , lateral acceleration,  $a_y$ , roll angle of the tractor,  $\phi_1$ , roll angle of the semitrailer,  $\phi_2$ , and articulation angle of the semitrailer with respect to the tractor  $\Gamma_1$  and the corresponding measured values for various average front-wheel steering angles,  $\delta_{av}$ , are given in Table 3.1

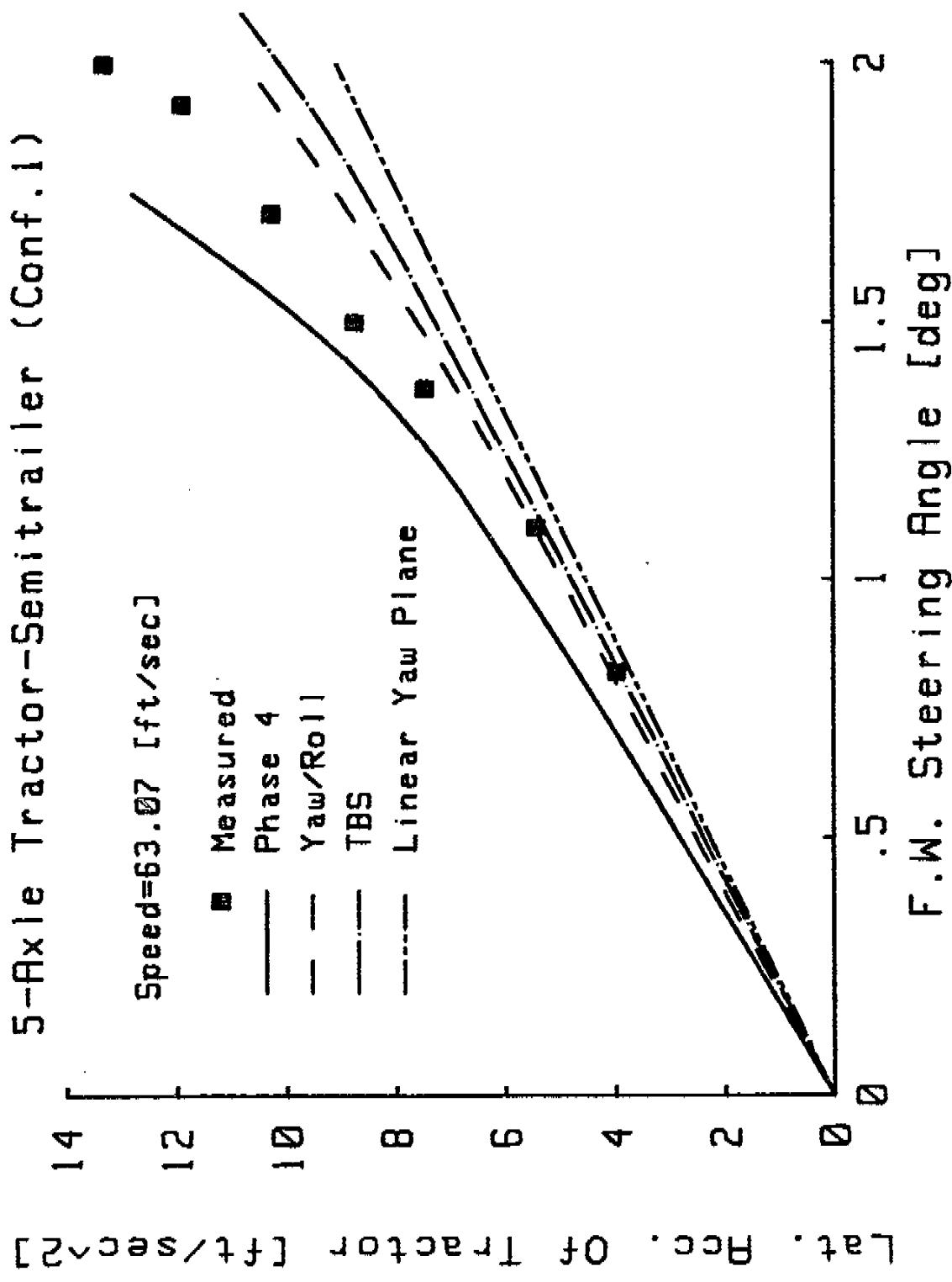


Fig. 3.2 Steady-state Lateral Acceleration Response to  
Steering Input of Vehicle Configuration 1  
Predicted by Various Models

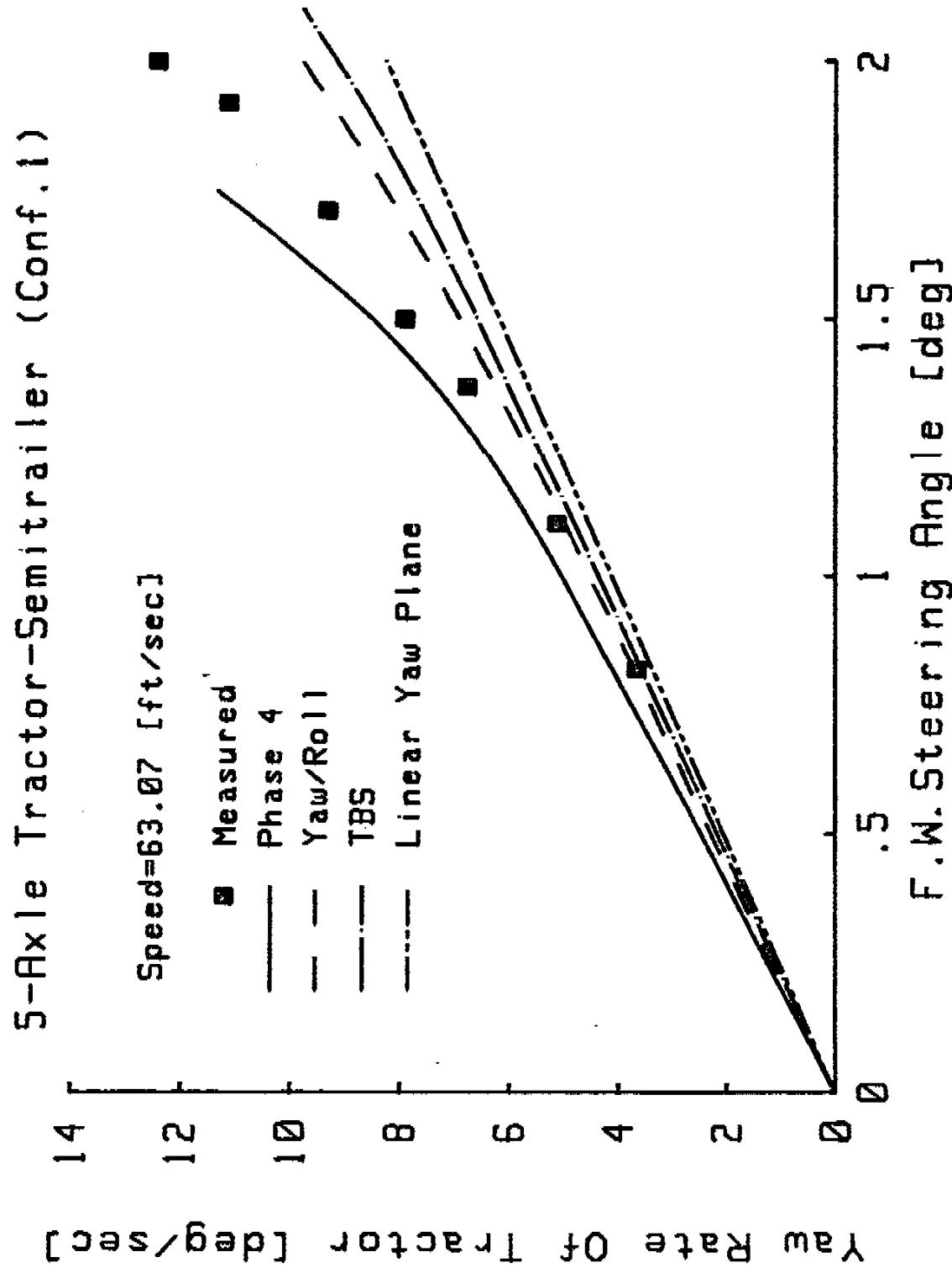


Fig. 3.3 Steady-state Yaw Rate Response to Steering Input of Vehicle Configuration 1 Predicted by Various Models

### 5-Axle Tractor-Semitrailer (Conf. 1)

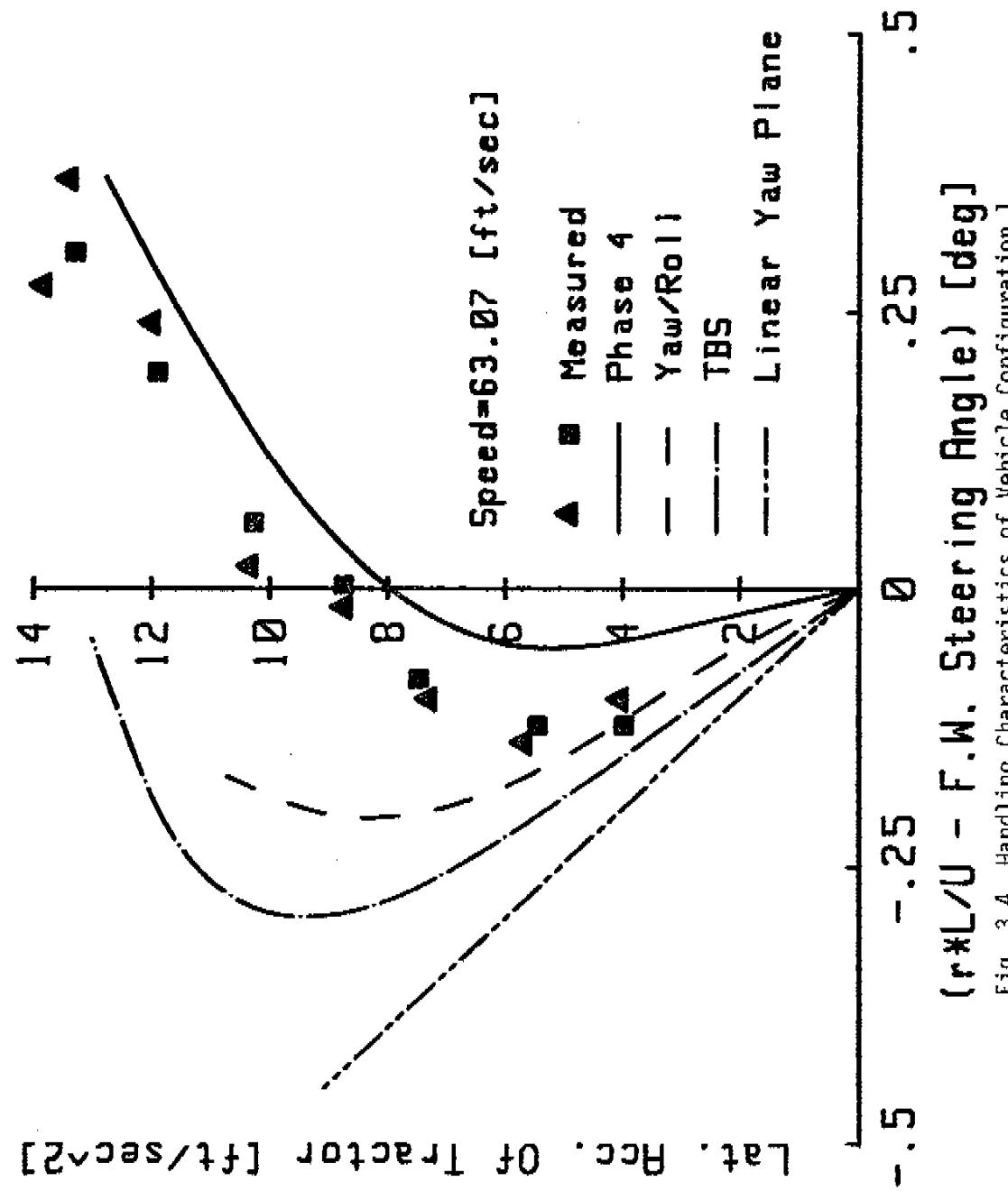


Fig. 3.4 Handling Characteristics of Vehicle Configuration 1  
Predicted by Various Models

TABLE 3.1  
VEHICLE CONFIGURATION 1

A. Phase 4 Model

$\delta_{av}$ [deg]	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \lambda / U - \delta_{av}$ [deg]	$\phi_1$ [deg]	$\phi_2$ [deg]	$\Gamma_1$ [deg]
0.5	62.99	2.48	2.82	-0.034	-1.18	-0.94	1.26
1.0	63.07	5.05	5.77	-0.052	-2.39	-1.94	2.49
1.5	63.06	8.49	9.65	+0.093	-3.94	-3.23	4.05
1.75	63.07	11.31	12.80	+0.372	-5.15	-4.26	4.83

B. Yaw/Roll Model

$\delta_{av}$ [deg]	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \lambda / U - \delta_{av}$ [deg]	$\phi_1$ [deg]	$\phi_2$ [deg]	$\Gamma_1$ [deg]
0.5	63.07	2.29	2.52	-0.07	-0.76	-0.76	1.15
1	63.07	4.52	4.97	-0.15	-1.50	-1.50	2.24
1.5	63.07	6.91	7.608	-0.203	-2.29	-2.29	3.40
1.75	63.07	8.25	9.08	-0.202	-2.74	-2.75	4.01
2.0	63.07	9.76	10.73	-0.168	-3.24	-3.25	4.62

C. TBS Model

$\delta_{av}$ [deg]	U [ft/sec]	r [deg/sec]	$A_y$ [ft/sec <sup>2</sup> ]	$r^* \cdot \ell/U - \delta_{av}$ [deg]	$r_1$ [deg]
0.6	62.97	2.61	2.87	- 0.109	1.3
1.0	63.07	4.36	4.80	- 0.182	2.16
1.2	63.2	5.25	5.80	- 0.217	2.6
1.5	63.07	6.59	7.27	- 0.263	3.26
1.9	63.07	8.55	9.45	- 0.295	4.13
2.1	63.08	9.75	10.81	- 0.271	4.51
2.2	63.08	10.63	11.81	- 0.205	4.53
2.25	63.05	11.75	13.02	- 0.044	4.56

D. Linear Yaw Plane Model

$\delta_{av}$ [deg]	U [ft/sec]	r [deg/sec]	$A_y$ [ft/sec <sup>2</sup> ]	$r^* \cdot \ell/U - \delta_{av}$ [deg]	$r_1$ [deg]
0.5	63.07	2.06	2.27	- 0.113	1.03
1.5	63.07	6.18	6.80	- 0.340	3.08
2.0	63.07	8.24	9.08	- 0.454	4.12

## E. Measurements

$\delta_{av}$ (deg)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	Calculated *		Measured **	
			$r^* \ell/U - \delta_{av}$ (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell/U - \delta$ (deg)	
0.82	3.66	3.97	-0.1333		4.04	- 0.1
1.1	5.13	5.44	-0.1375		5.68	- 0.139
1.37	6.77	7.47	-0.0998		7.3	- 0.1
1.5	7.9	8.75	+0.0178		8.75	- 0.017
1.71	9.3	10.27	+0.0349		10.34	+ 0.02
1.92	11.12	11.9	+0.1664		12.0	+ 0.24
2.055	12.4	13.3	+0.2715		13.4	+ 0.37
2.2	12.83	13.7	+0.202		13.84	+ 0.274

\* Calculations based on the measured yaw rates, a forward speed of 63.07 ft/sec, and a wheelbase of 142 in.

\*\* Measured values taken from reference (5)

All measured values are for left-turn manoeuvres

It can be seen from Fig. 3.2 that the lateral accelerations of the tractor predicted using the linear yaw plane model, the TBS model and the yaw/roll model are reasonably close within the range up to 2 degrees of front-wheel steering angle (equivalent to lateral acceleration of approximately 0.3 g). However, there is a significant difference between the lateral accelerations predicted using the Phase 4 model and those predicted using the other three models for front-wheel steering angles greater than 1.5 degrees (equivalent to lateral acceleration of approximately 0.2 g.)

From Fig. 3.2 or Table 3.1, it can be noted that at front-wheel steering angle  $\delta_{av} = 1.0^\circ$ , the differences between the measured lateral acceleration and the predicted ones using the Phase 4 model, the yaw/roll model, the TBS model, and the linear yaw plane model are 17.5%, 1.2%, 2.3%, and 7.7%, respectively. At front-wheel steering angle  $\delta_{av} = 1.5^\circ$ , the corresponding differences are 10.3%, 13.1%, 13.8%, and 22.3%, respectively.

As can be seen from Fig. 3.3 or Table 3.1, the differences between the measured yaw rates and the predicted ones using the Phase 4 model, the yaw/roll model, the TBS model and the linear yaw plane model show similar trend as that between the measured lateral accelerations and the predicted ones using the four computer simulation models described above.

Based on the data shown in Figs. 3.2 and 3.3, and Table 3.1, it appears that for lateral accelerations below 0.2 g, the four computer

simulation models give similar predictions and the predicted values agree reasonably well with the measured ones. For the lateral acceleration range between 0.2 g and 0.3 g, the Phase 4 model, the yaw/roll model and the TBS model seem to have similar error of prediction, in comparison with the measured values. However, the Phase 4 model overestimates the responses, while the yaw/roll model and the TBS model underestimate them. For lateral accelerations higher than 0.2 g, the linear yaw plane model gives higher error of prediction than the other three models. This is primarily due to the fact that in the linear yaw plane model a linear tire model is used and the load transfer and its effects on tire characteristics have been entirely neglected.

It should also be mentioned that the lateral acceleration which causes an inside tire to lift off the ground predicted by the Phase 4 model, the yaw/roll model and the TBS model is considerably lower than the measured one reported in reference (5).

Fig. 3.4 illustrates the steady-state handling characteristics of Vehicle Configuration 1 as predicted by the four computer simulation models. It can be seen that the lateral acceleration at which the vehicle changes from understeer to oversteer, referred to as the "transition acceleration", predicted using the Phase 4 model is just under 0.2 g. The transition accelerations predicted using the yaw/roll model and the TBS model are approximately 0.25 g and 0.3 g, respectively, while the measured one is approximately 0.2 g. Below the transition acceleration, the Phase 4 model underestimates the understeer level (or "understeer gradient") as compared with the measured data, whereas the TBS model and the linear

yaw plane model overestimate the understeer level to varying degrees. The yaw/roll model appears to give the best prediction of the understeer level. It should be mentioned that since a linear tire model is used, the linear yaw plane model is unable to predict any variation of the handling behaviour of the vehicle with lateral acceleration and the predicted understeer level remains a constant.

From Table 3.1, it can be seen that the yaw/roll model, the TBS model, and the linear yaw plane model give similar predictions of the articulation angle of the semitrailer with respect to the tractor, while the Phase 4 model gives a higher prediction than the other three models. It should also be mentioned that the roll angles of the spring masses of the tractor and the semitrailer predicted by the yaw/roll model are essentially the same, whereas those predicted using the Phase 4 model are considerably different.

### 3.2 Vehicle Configuration 2

As mentioned in Section 2, this vehicle is a modified version of Vehicle Configuration 1, with much higher torsional stiffness of the tractor frame and roll stiffness of the tractor front suspension. These two major modifications tend to make the tractor highly understeer.

It should be pointed out that among the four computer simulation models, only the Phase 4 model takes into account the effects of both the tractor frame torsional stiffness and the roll stiffnesses of the suspensions. The yaw/roll model and the TBS model only take into account the effects of the roll stiffnesses of the suspensions. The tractor frame is assumed to be rigid in these two simulation models. The linear yaw plane model does not include either the tractor frame torsional stiffness or the roll stiffnesses of the suspensions. Consequently, these changes do not affect the predictions made by the linear yaw plane model. The steady-state steering response of Vehicle Configuration 2 predicted using the linear yaw plane model is, therefore, the same as that of Vehicle Configuration 1 described previously.

The predicted steady-state lateral accelerations and yaw rates of the tractor using the four computer simulation models, are shown in Figs. 3.5 and 3.6, respectively. Based on the predictions, a handling diagram is drawn as shown in Fig. 3.7. For comparison, the measured data taken from reference (5) are also included in the figures. The predicted and the measured values for various front-wheel steering angles are also given in Table 3.2.

### 5-Axle Tractor-Semitrailer (Conf. 2)

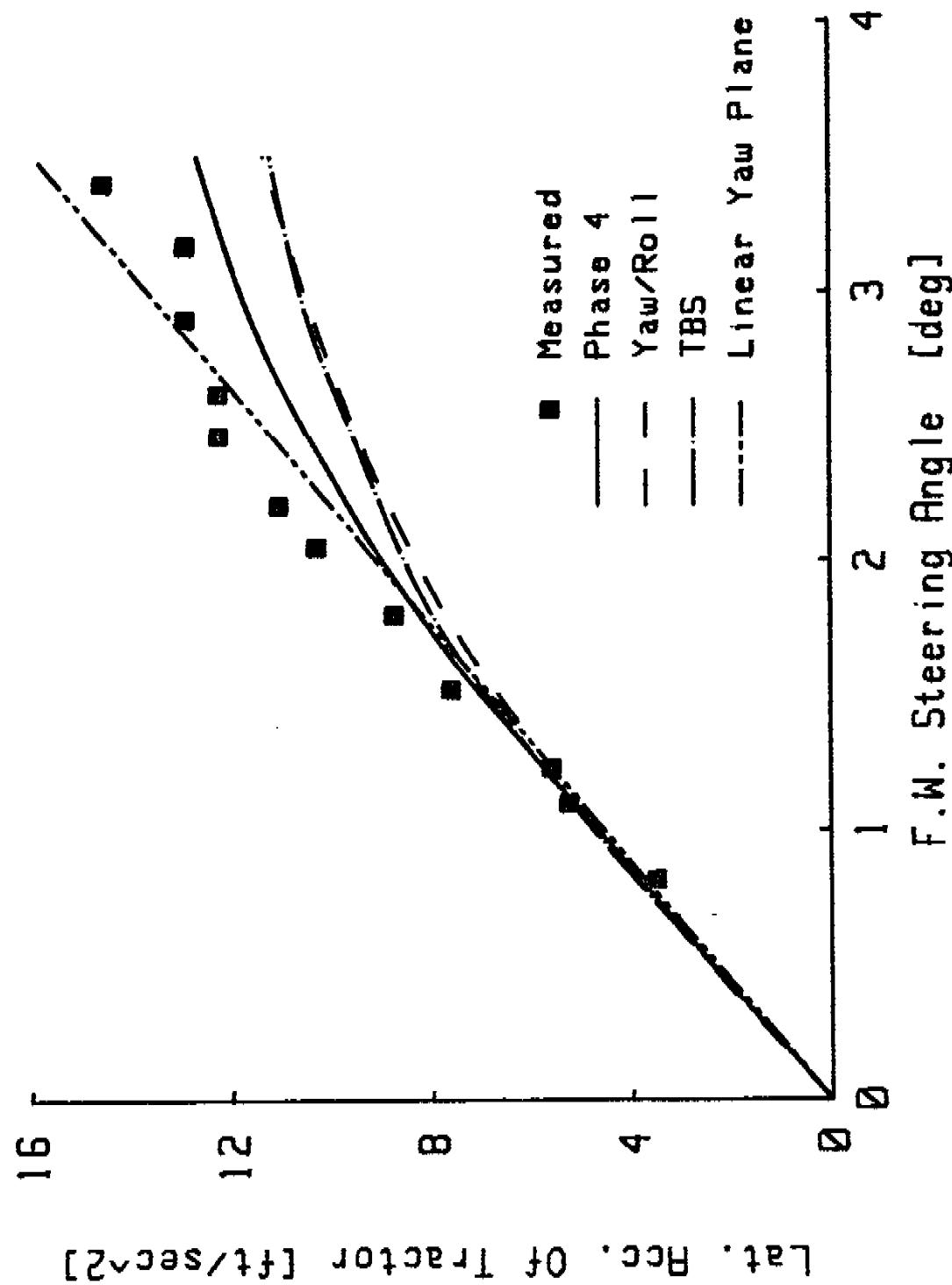


Fig. 3.5 Steady-state Lateral Acceleration Response  
to Steering input of Vehicle Configuration 2  
Predicted by Various Models

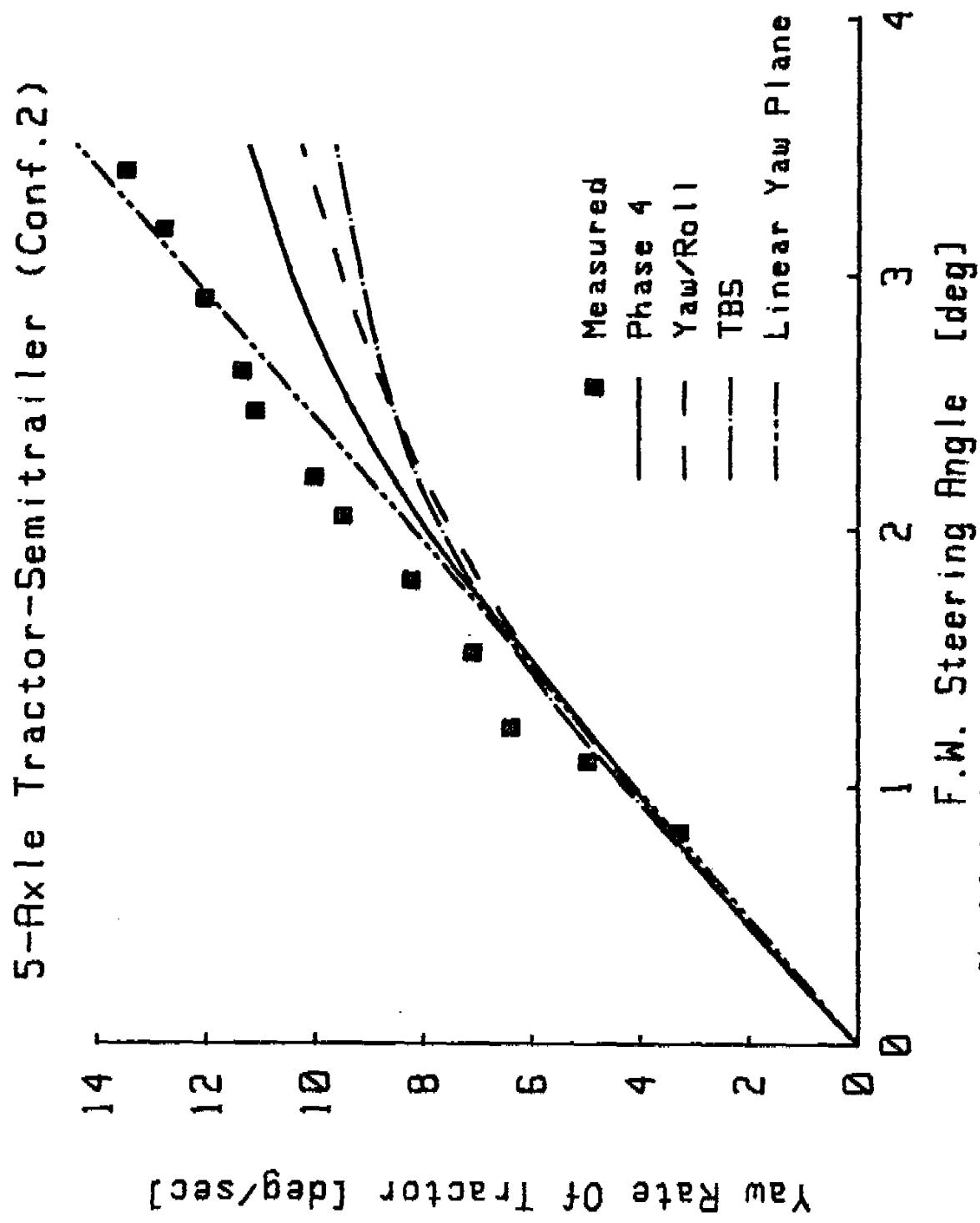


Fig. 3.6 Steady-state Yaw Rate Response to Steering Input of Vehicle Configuration 2 Predicted by Various Models

### 5-Axle Tractor-Semitrailer (Conf. 2)

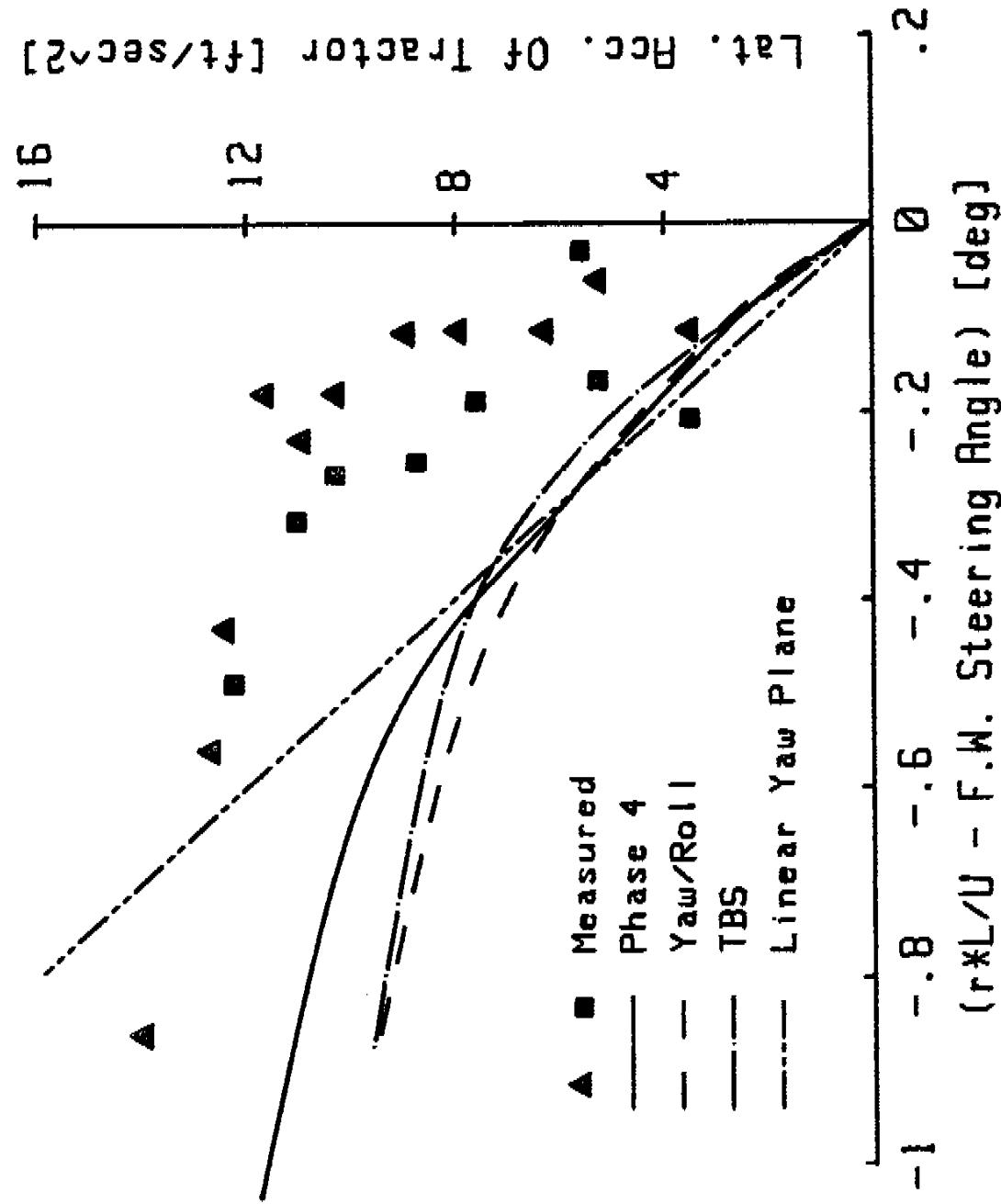


Fig. 3.7 Handling Characteristics of Vehicle Configuration 2 Predicted by Various Models

TABLE 3.2  
VEHICLE CONFIGURATION 2

A. Phase 4 Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	63.07	2.18	2.368	-0.091	-0.50	-0.71	1.06
1.0	63.11	4.15	4.65	-0.221	-0.95	-1.37	2.00
1.5	63.10	6.06	6.79	-0.363	-1.39	-2.00	2.85
2.0	63.07	7.98	8.96	-0.502	-1.83	-2.64	3.72
2.5	63.05	9.40	10.55	-0.735	-2.15	-3.10	4.23
3.0	63.07	10.47	11.78	-1.035	-2.39	-3.46	4.55
3.5	63.10	11.25	12.66	-1.390	-2.55	-3.69	4.68

B. Yaw/Roll Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	63.07	2.20	2.42	-0.087	-0.58	-0.58	1.11
1.0	63.07	4.20	4.61	-0.212	-1.11	-1.11	2.08
1.5	63.07	6.05	6.65	-0.364	-1.59	-1.60	2.99
2.0	63.07	7.53	8.29	-0.587	-1.98	-2.0	3.72
2.5	63.07	8.66	9.53	-0.875	-2.28	-2.3	4.25
3.0	63.07	9.55	10.51	-1.208	-2.52	-2.54	4.63
3.5	63.07	10.29	11.32	-1.569	-2.71	-2.74	4.93

C. TBS Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	63.05	2.17	2.39	-0.092	1.08
1.0	63.11	4.31	4.75	-0.191	2.14
1.5	63.07	6.24	6.88	-0.329	3.07
2.0	63.08	7.7	8.53	-0.555	3.74
2.5	63.06	8.63	9.63	-0.880	4.12
2.0	63.05	9.19	10.41	-1.275	4.31
3.5	63.12	9.66	11.2	-1.689	4.4

D. Linear Yaw Plan Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	63.07	2.06	2.27	-0.113	1.03
1.5	63.07	6.18	6.80	-0.340	3.08
2.0	63.07	8.24	9.08	-0.454	6.12
3.5	63.07	14.43	15.89	-0.794	7.21

E. Measurements

$\delta_{av}$ (deg)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	Calculated *		Measured **	
			$r^* l / U - \delta_{av}$ (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* l / U - \delta$ (deg)	
0.82	3.27	3.5	-0.206	3.5	-0.113	
1.1	4.98	5.25	-0.165	5.25	-0.06	
1.23	6.40	5.6	-0.029	6.3	-0.113	
1.52	7.10	7.6	-0.187	7.93	-0.113	
1.8	8.24	8.75	-0.254	8.94	-0.116	
2.05	9.5	10.3	-0.267	10.27	-0.18	
2.2	10.03	11.04	-0.318	10.93	-0.23	
2.46	11.12	11.66	-0.373	11.63	-0.18	
2.62	11.36	12.25	-0.488	12.4	-0.43	
2.9	12.05	12.90	-0.639			
3.17	12.8	12.90	-0.768	12.68	-0.56	
3.4	13.5	14.54	-0.867	14.0	-0.86	

\* Calculations based on the measured yaw rates, a forward speed of 63.07 ft/sec, and a wheelbase of 142 in.

\*\* Measured values taken from reference (5).

It can be seen from Figs. 3.5 and 3.6 that within the range up to 2 degrees of front-wheel steering angle (equivalent to lateral acceleration of approximately 0.25 g), all four simulation models give practically the same predictions, and the predicted values agree very well with the measured data. Within the range between 2 and 4 degrees of front-wheel steering angle, the linear yaw plane model gives the best overall predictions of lateral acceleration and yaw rate of the tractor, among the four simulation models. For instance, at front-wheel steering angle  $\delta_{av} = 3^\circ$ , the differences between the measured lateral acceleration and the predicted ones using the linear yaw plane model, the Phase 4 model, the yaw/roll model and the TBS model are 3.7%, 10.3%, 19.9% and 20.7%, respectively.

Fig. 3.7 shows a comparison of the steady-state handling characteristics of the vehicle predicted using the four simulation models and those measured (5). It can be seen that up to a lateral acceleration of approximately 0.25 g, all four models give essentially the same predictions. Beyond lateral acceleration of 0.25 g, the yaw/roll model and the TBS model give the highest prediction of understeer level (or "understeer gradient"), while the linear yaw plane model gives the lowest prediction. It can also be noted that while the handling characteristics predicted by the Phase 4 model, the yaw/roll model, and the TBS model and the measured ones show similar trend, in quantitative terms there is a significant difference between them. It can, therefore, be said that the agreement between the measured and the predicted handling characteristics using the four models is, in general, poor for lateral acceleration above 0.15 g.

From Table 3.2, it can be seen that all the four models give similar predictions of the articulation angle between the tractor and semitrailer for front-wheel steering angle less than  $3.0^{\circ}$ . It should also be mentioned that the roll angles of the sprung masses of the tractor and semitrailer predicted by the yaw/roll model are practically the same, while those predicted using the Phase 4 model are noticeably different.

### 3.3 Vehicle Configuration 3

As can be seen from Table 2.1, the major differences between this vehicle and Vehicle Configurations 1 and 2 are the location of the fifth wheel, which is 14.35 in. ahead of the tractor rear suspension centre for Vehicle Configuration 3, the characteristics of the tires, roll stiffness of the fifth wheel, and the auxiliary roll stiffnesses of the tractor suspensions and the tandem suspensions of the semitrailer.

The steady-state steering responses of this vehicle were predicted using the four computer simulation models at a forward speed of 35.8 mph (52.5 ft/sec) on a dry, smooth asphalt surface. The predicted steady-state lateral accelerations and yaw rates as functions of front-wheel steering angle of the tractor are shown in Figs. 3.8 and 3.9, respectively. Based on the predictions, a handling diagram, illustrating the handling characteristics of the vehicle predicted using the four simulation models, is shown in Fig. 3.10. For this vehicle configuration, no measured data are available. Therefore, the results presented are primarily for illustrating the differences in the predictive ability of various computer simulation models. The predicted values at various front-wheel steering angles are also given in Table 3.3.

As can be seen from the results shown in Figs. 3.8 and 3.9 and in Table 3.3, all four simulation models give similar predictions of lateral acceleration and yaw rate up to front-wheel steering angle of approximately  $2.6^{\circ}$  (or equivalent to lateral acceleration of approximately 0.28 g). The difference between the highest value of lateral acceleration predicted using the TBS model and the lowest one predicted using the linear yaw plane model at front-wheel steering angle of  $2.6^{\circ}$  is approximately 10.6%.

### 5-Axle Tractor-Semitrailer (Conf. 3)

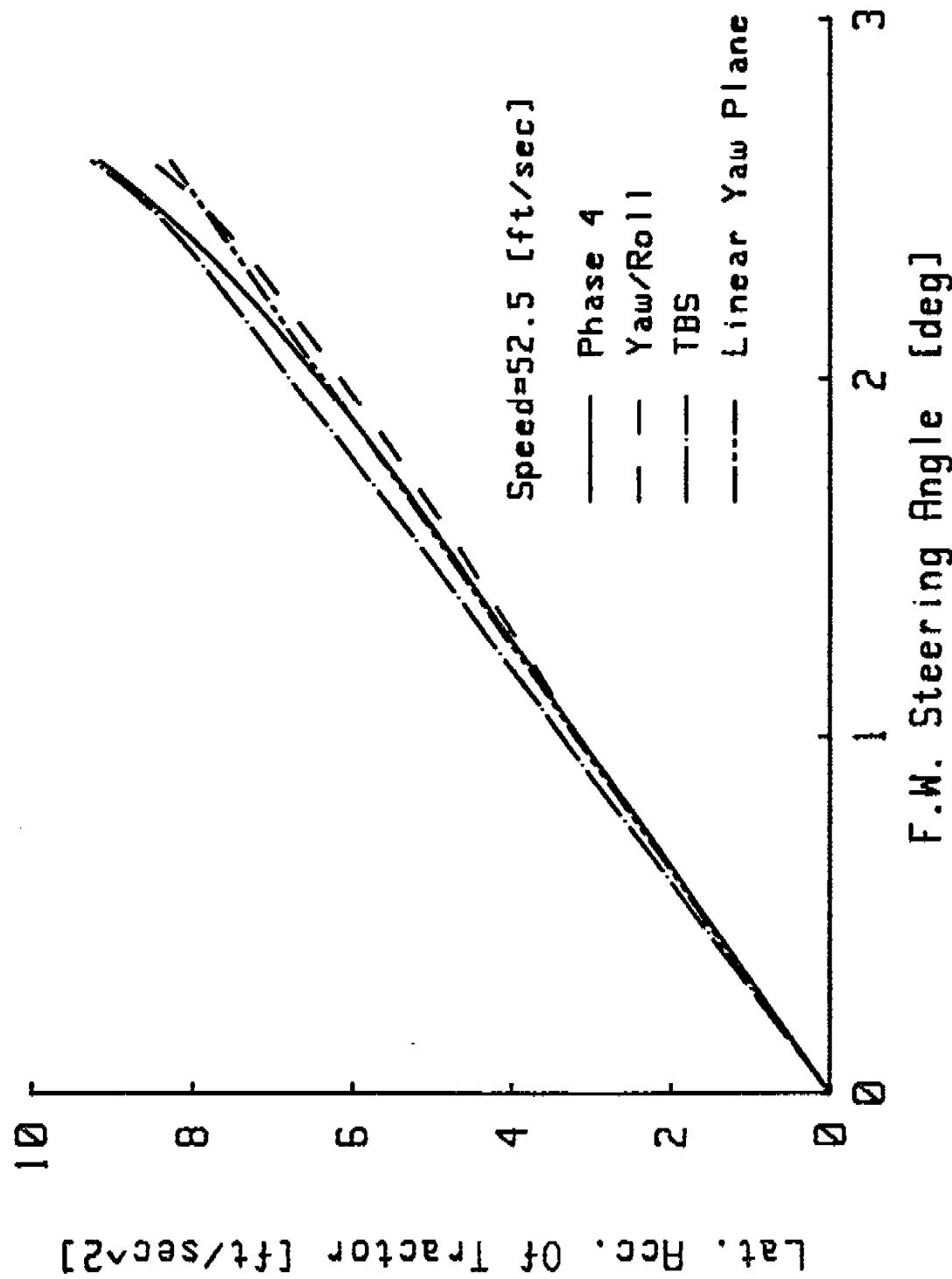


Fig. 3.8 Steady-state Lateral Acceleration Response to Steering Input of Vehicle Configuration 3 Predicted by Various Models

### 5-Axle Tractor-Semitrailer (Conf. 3)

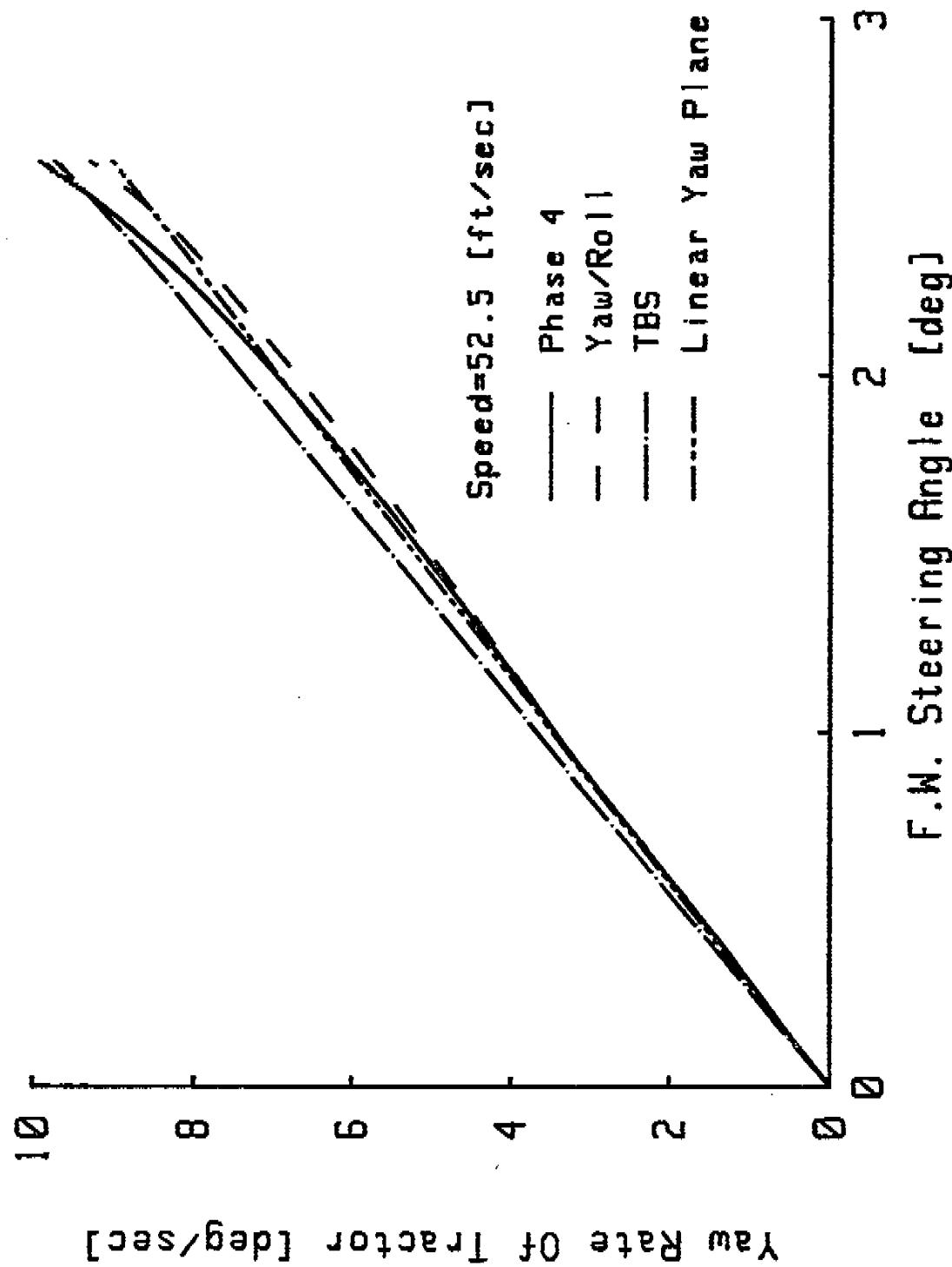


Fig. 3.9 Steady-state Yaw Rate Response to Steering  
Input of Vehicle Configuration 3  
Predicted by Various Models

### 5-Axle Tractor-Semitrailer (Conf. 3)

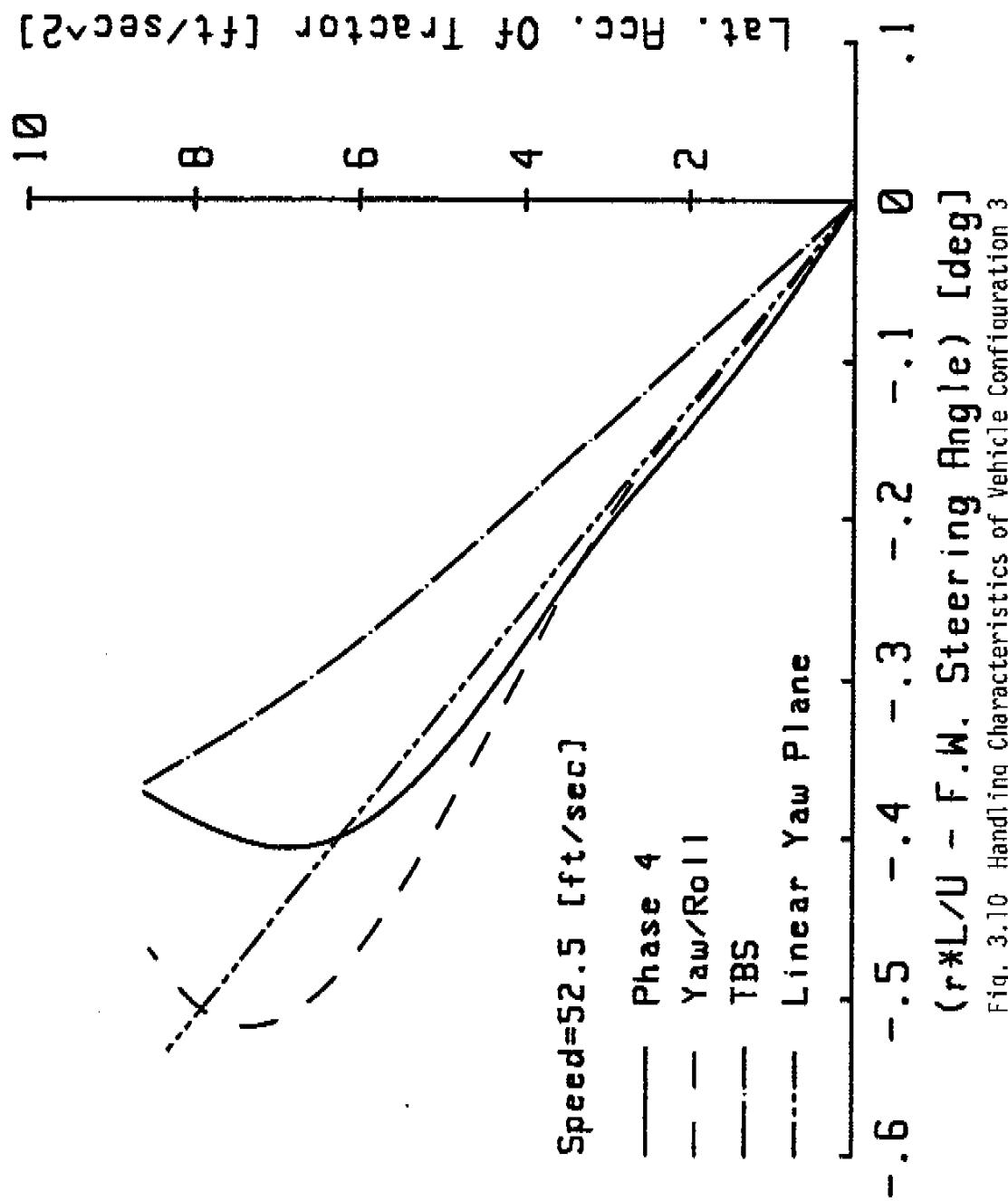


Fig. 3.10 Handling Characteristics of Vehicle Configuration 3

Predicted by Various Models

TABLE 3.3  
VEHICLE CONFIGURATION 3

A. Phase 4 Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.68	1.69	1.55	-0.115	-0.64	-0.57	1.11
1.0	52.7	3.44	3.15	-0.216	-1.56	-1.39	2.25
1.5	52.6	5.11	4.74	-0.332	-2.56	-2.22	3.33
2.0	52.5	6.98	6.43	-0.404	-3.41	-3.01	4.52
2.5	52.51	9.31	8.60	-0.372	-4.97	-4.5	6.06
2.6	52.49	9.95	9.16	-0.325	-6.63	-6.4	6.56

B. Yaw/Roll Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.73	1.59	-0.105	-0.66	-0.7	1.15
1.0	52.5	3.44	3.148	-0.213	-1.33	-1.41	2.27
1.5	52.5	5.02	4.6	-0.352	-1.96	-2.07	3.31
2.0	52.5	6.73	6.17	-0.461	-2.67	-2.83	4.4
2.5	52.5	8.74	8.01	-0.502	-3.62	-3.85	5.64
2.6	52.5	9.32	8.53	-0.469	-4.09	-4.33	6.02

C. TBS Model

$\delta_{av}$ (Deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.84	1.68	-0.079	1.21
1.0	52.5	3.68	3.37	-0.158	2.43
1.5	52.5	5.52	5.06	-0.238	3.64
2.0	52.58	7.4	6.8	-0.308	4.81
2.5	52.53	9.33	8.59	-0.367	5.65
2.6	52.5	9.76	9.27	-0.369	5.57

D. Linear Yaw Plane Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
1.0	52.5	3.48	3.19	-0.204	2.29
2.0	52.5	6.96	6.37	-0.409	4.58
2.6	52.5	9.04	8.29	-0.533	5.9

From Fig. 3.10, it can be seen that up to lateral acceleration of approximately 0.15 g, the Phase 4 model, the yaw/roll model and the linear yaw plane model give similar predictions of the understeer level of the vehicle, whereas the TBS model predicts lower understeer level. Both the Phase 4 model and the yaw/roll model predict a transition from understeer to oversteer at a lateral acceleration of approximately 0.2 g, while the TBS model does not predict a transition below 0.25 g of lateral acceleration. As mentioned previously, since a linear tire model is used, the linear yaw plane model is unable to predict any variation of the handling behaviour of the vehicle with lateral acceleration, and the predicted understeer level remains constant.

From Table 3.3, it can be seen that the Phase 4 model gives higher prediction of the articulated angle between the tractor and semitrailer than the models at high front-wheel steering angles. It should also be mentioned that the roll angles of the sprung masses of the tractor and semitrailer predicted by the Phase 4 model are higher than those predicted by the yaw/roll model.

### 3.4 Vehicle Configuration 4

As shown in Table 2.1, the major differences between this vehicle and Vehicle Configuration 3 are the removal of the auxiliary roll-stiffening device from the tractor front suspension and the installation of a much stiffer roll-stiffening device on the tractor trailing tandem axle. The combined effect is the lowering of the understeer level of this vehicle, in comparison with Vehicle Configuration 3.

The steady-state lateral accelerations and yaw rates as functions of front-wheel steering angle of the tractor predicted using the four computer simulation models are shown in Fig. 3.11 and Fig. 3.12, respectively. The handling characteristics of the vehicle predicted by the computer simulation models are illustrated in Fig. 3.13. The predicted values are also listed in Table 3.4.

As can be seen from the predicted results shown in Figs. 3.11 and 3.12, and in Table 3.4, all four simulation models give essentially the same predictions of lateral acceleration and yaw rate up to front-wheel steering angle of 2 degrees, or lateral acceleration of approximately 0.2 g. Beyond that the differences between the predicted values obtained using the four models increase noticeably. For instance, at a front-wheel steering angle of 2.5 degrees, the differences between the predicted lateral accelerations using the Phase 4 model and those using the yaw/roll model, the TBS model and the linear yaw plane model are 7.5%, 14%, and 20.9%, respectively.

### 5-Axle Tractor-Semitrailer (Conf. 4)

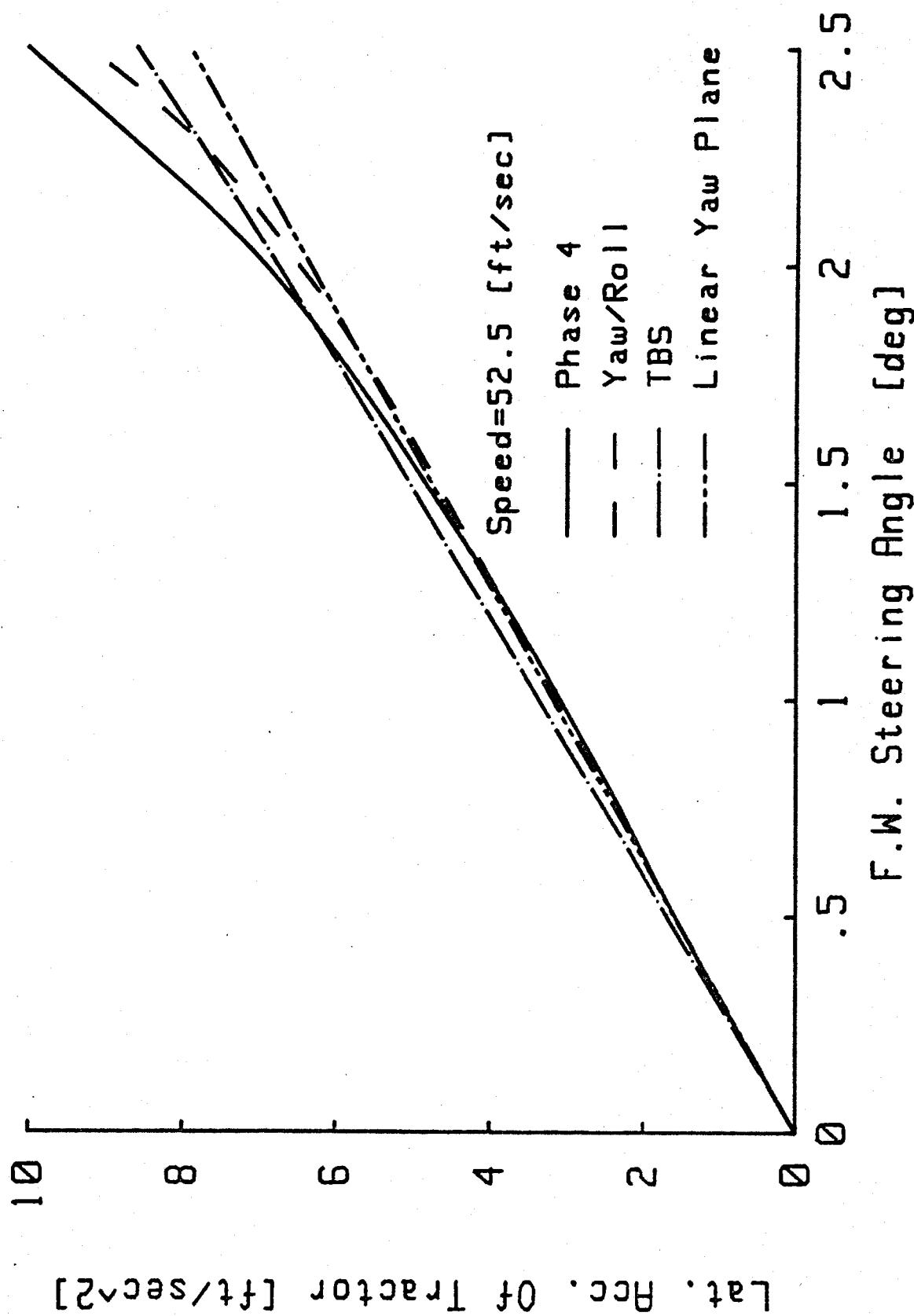


Fig. 3.11 Steady-state Lateral Acceleration Response to Steering Input of Vehicle Configuration 4 Predicted by Various Models

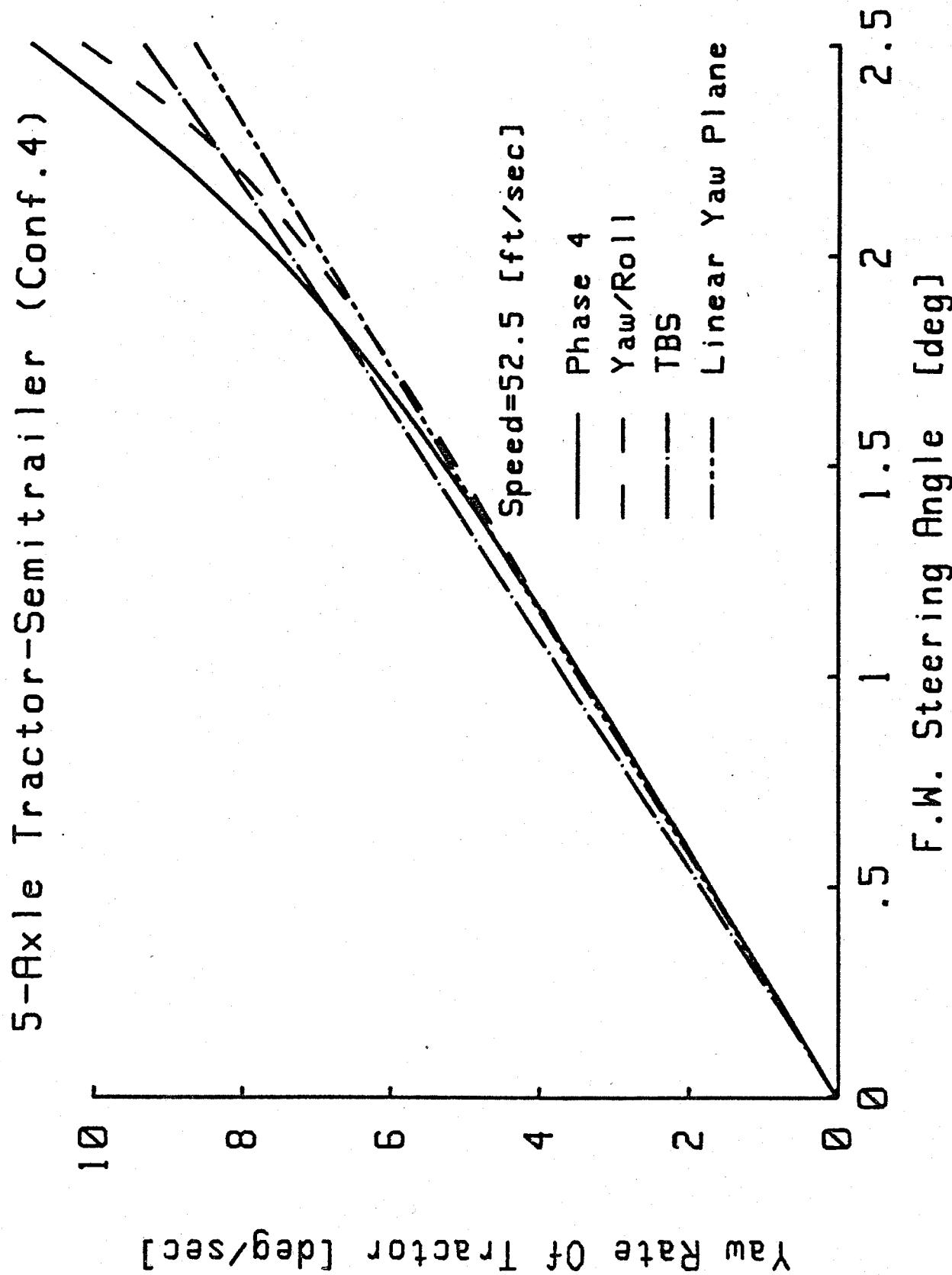


Fig. 3.12 Steady-state Yaw Rate Response to Steering Input of Vehicle Configuration 4 Predicted by Various Models

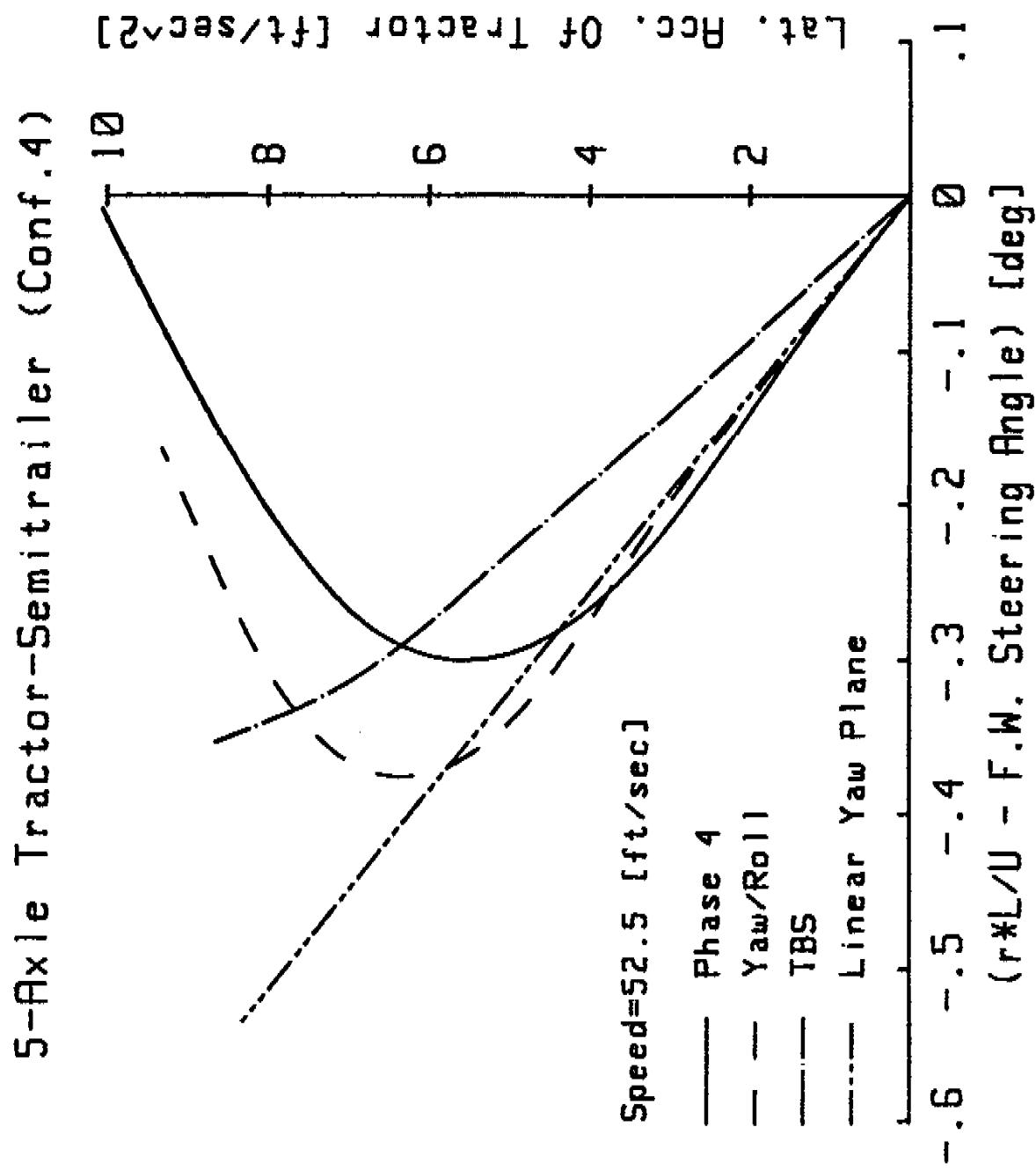


Fig. 3.13 Handling Characteristics of Vehicle Configuration 4 Predicted by Various Models

TABLE 3.4  
VEHICLE CONFIGURATION 4

A. Phase 4 Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.54	1.71	1.58	-0.109	-0.75	-0.59	1.12
1.0	52.76	3.43	3.08	-0.219	-1.69	-1.37	2.28
1.5	52.59	5.30	4.85	-0.291	-2.49	-2.08	3.52
1.75	52.6	6.36	5.82	-0.299	-2.07	-2.51	4.20
2.0	52.61	7.59	6.96	-0.268	-3.58	-3.04	5.04
2.25	52.51	9.10	8.44	-0.199	-4.35	-3.72	6.11
2.5	52.5	10.90	10.06	-0.008	-8.16	-7.90	7.41

B. Yaw/Roll Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.73	1.58	-0.105	-0.6	-0.65	1.15
1.0	52.5	3.45	3.16	-0.211	-1.22	-1.33	2.30
1.5	52.5	5.14	4.70	-0.325	-1.83	-1.99	3.43
1.75	52.5	6.06	5.54	-0.365	-2.16	-2.35	4.04
2.0	52.5	7.11	6.52	-0.375	-2.57	-2.80	4.75
2.25	52.5	8.39	7.66	-0.332	-3.09	-3.39	5.62
2.5	52.5	10.22	9.31	-0.164	-4.18	-4.59	6.89

C. TBS Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \delta / U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.84	1.69	- 0.079	1.21
1.0	52.5	3.68	3.37	- 0.158	2.43
1.5	52.5	5.53	5.07	- 0.236	3.64
2.0	52.5	7.4	6.8	- 0.308	4.83
2.5	52.52	9.39	8.65	- 0.353	5.66

D. Linear Yaw Plane Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \delta / U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
1.0	52.5	3.48	3.19	- 0.204	2.29
2.0	52.5	6.96	6.37	- 0.409	4.58
2.5	52.5	8.7	7.96	- 0.511	5.72
2.6	52.5	9.04	8.29	- 0.537	5.9

From Fig. 3.13, it can be seen that up to a lateral acceleration of approximately 0.125 g, the Phase 4 model, the yaw/roll model and the linear yaw plane model give essentially the same predictions of the understeer level of the vehicle, whereas the TBS model predicts a lower level of understeer. The Phase 4 model predicts a transition from understeer to oversteer at a lateral acceleration of approximately 0.175 g, while the yaw/roll model predicts the transition occurring at a slightly higher lateral acceleration of approximately 0.2 g. The TBS model does not predict a transition below 0.28 g of lateral acceleration, and as mentioned previously, the linear yaw plane model is unable to predict any variation of the handling behaviour of the vehicle with lateral acceleration.

From Table 3.4, it can be seen that the TBS model and the linear yaw plane model give similar predictions of the articulation angle. The Phase 4 model and the yaw/roll model however, generally give higher predictions of articulation angle than the other two models. It can be also noted that the roll angles of the sprung masses of the tractor and semitrailer predicted by the Phase 4 model are generally higher than those predicted by the yaw/roll model, when the front-wheel steering angle is greater than 2 degrees.

### 3.5 Vehicle Configuration 5

It can be seen from Table 2.1, the basic difference between this vehicle and Vehicle Configuration 4 is the location of the fifth wheel. In this vehicle, the fifth wheel is situated at the centre of the rear suspension of the tractor, whereas in Vehicle Configuration 4, the fifth wheel is located ahead of the tractor rear suspension centre. The rearward shift of the fifth wheel location generally reduces the understeer level of the tractor.

The steady-state lateral accelerations and yaw rates as functions of front-wheel steering angle of the tractor predicted using the four computer simulation models are shown in Figs. 3.14 and 3.15 respectively. The handling characteristics of the vehicle predicted by the computer simulation models are shown in Fig. 3.16. The predicted values are also listed in Table 3.5.

As can be seen from the predicted results shown in Figs. 3.14 and 3.15 and in Table 3.5, all four simulation models give essentially the same predictions of lateral acceleration and yaw rate up to front-wheel steering angle of 1.5 degrees (or lateral acceleration of 0.15 g approximately). Beyond that the differences between the predicted values obtained using the four models increase noticeably. For instance, at front-wheel steering angle of 2.0 degrees, the differences between the predicted lateral acceleration using the Phase 4 model and those using the yaw/roll model, the TBS model and the linear yaw plane model are 6.8%, 19.6%, and 25.2% respectively.

### 5-Axle Tractor-Semi trailer (Conf. 5)

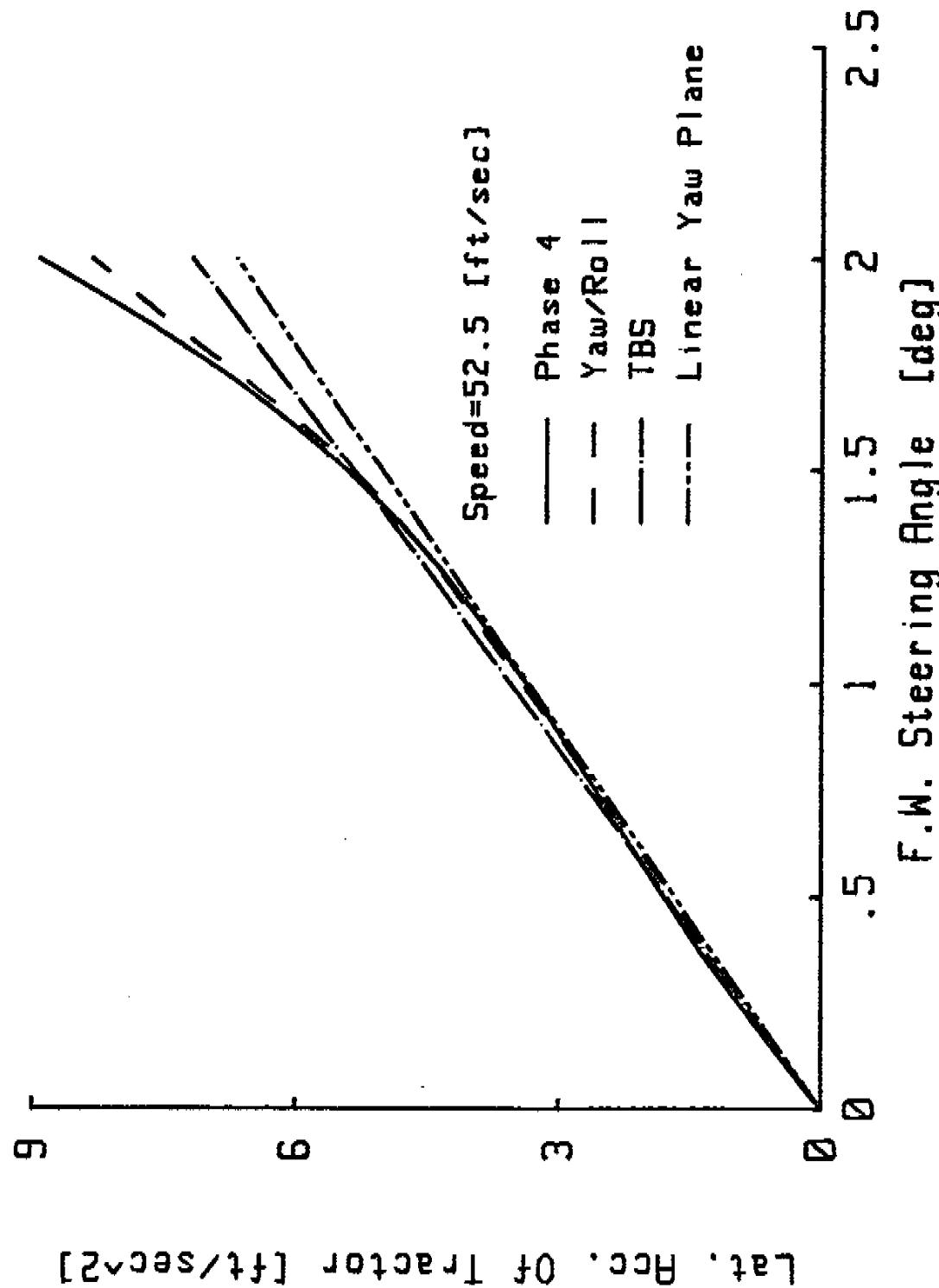


Fig. 3.14 Steady-state lateral acceleration response to steering input of Vehicle Configuration 5 predicted by various models

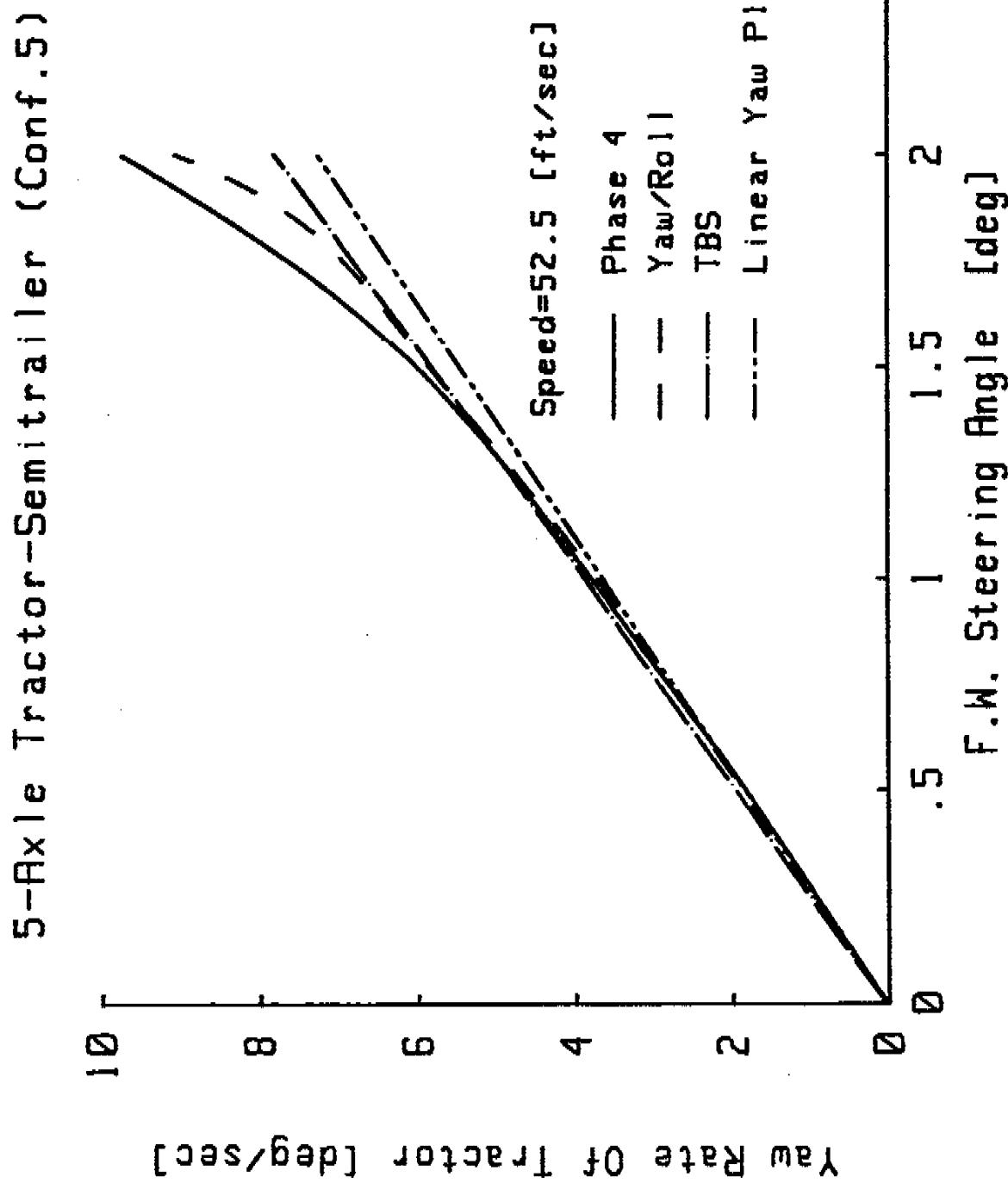


Fig. 3.15 Steady-state yaw rate response to steering input of Vehicle Configuration 5  
Predicted by various models

5-Axle Tractor-Semitrailer (Conf. 5)

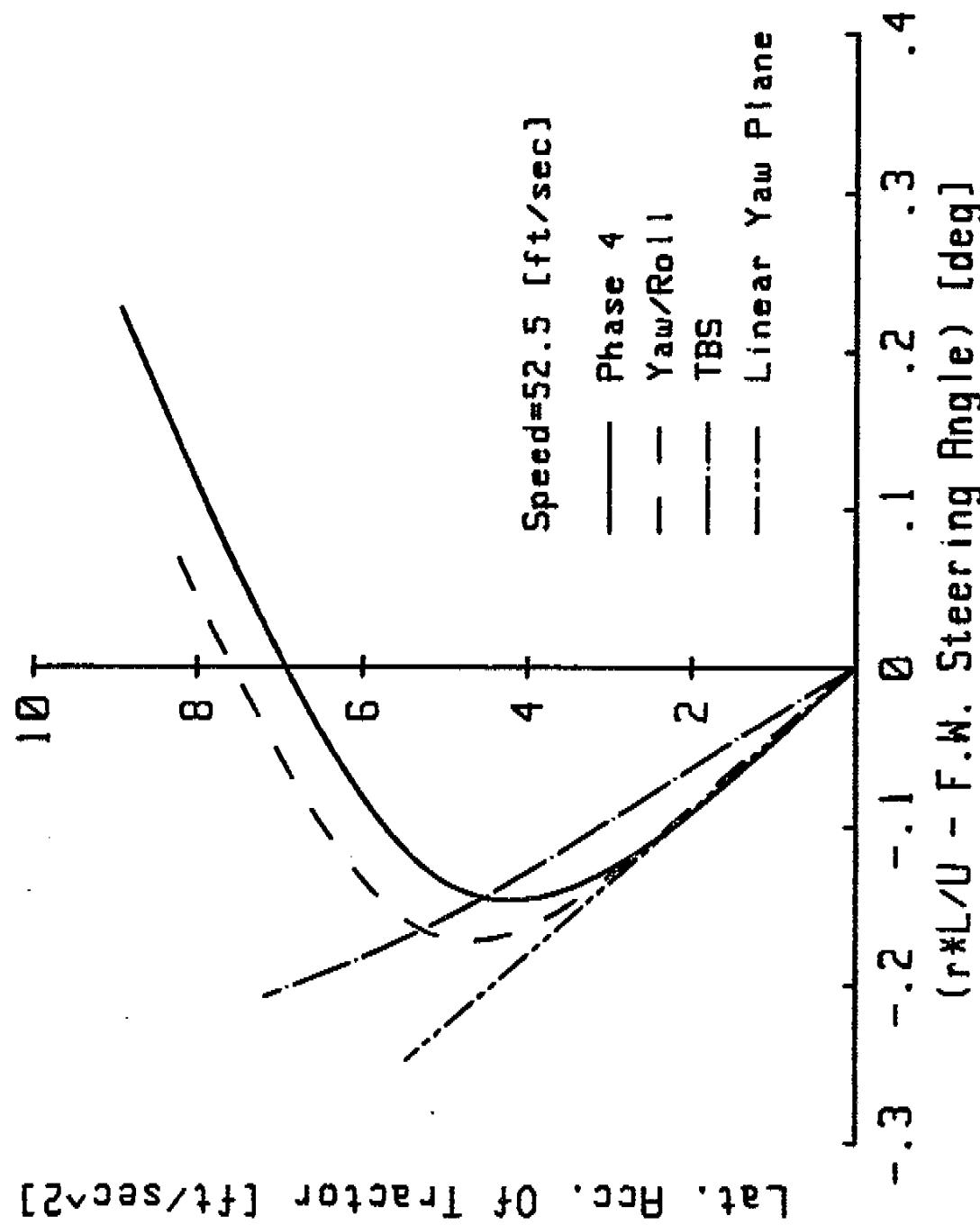


Fig. 3.16 Handling characteristics of Vehicle Configuration 5 predicted by various models

TABLE 3.5  
VEHICLE CONFIGURATION 5

A. Phase 4 Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.82	1.80	-0.084	-0.92	-0.73	1.27
1.0	52.71	3.79	3.37	-0.137	-1.81	-1.49	2.62
1.5	52.51	6.01	5.42	-0.126	-2.73	-2.40	4.23
1.75	52.5	7.64	6.93	-0.0037	-3.34	-2.99	5.31
2.0	52.52	9.75	8.94	+0.228	-4.51	-4.02	6.82

B. Yaw/Roll Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg)	$A_y$ (ft/sec <sup>2</sup> )	$r^* \ell / U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.86	1.70	-0.075	-0.64	-0.70	1.29
1.0	52.5	3.73	3.41	-0.147	-1.3	-1.42	2.59
1.5	52.5	5.85	5.35	-0.163	-2.04	-2.23	4.07
1.75	52.5	6.95	6.80	-0.161	-2.55	-2.85	5.1
2.0	52.5	9.10	8.33	+0.08	-3.32	-3.64	6.36

C. TBS Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.94	1.77	- 0.057	1.30
1.0	52.5	3.87	3.55	- 0.115	2.59
1.5	52.51	5.83	5.35	- 0.168	3.90
1.75	52.5	6.83	6.26	- 0.189	4.53
2.0	52.49	7.84	7.19	- 0.208	5.15

D. Linear Yaw Plane Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)
0.5	52.5	1.82	1.67	- 0.084	1.22
1.0	52.5	3.65	3.34	- 0.166	2.44
2.0	52.5	7.3	6.69	- 0.331	4.89

It can be seen from Fig. 3.16 that up to lateral acceleration of 0.1 g, the Phase 4 model, the yaw/roll model and the linear yaw plane model give practically the same prediction of understeer level, whereas the TBS model predicts lower level of understeer. The Phase 4 model predicts a transition from understeer to oversteer at a lateral acceleration of approximately 0.125 g, while the yaw/roll model predicts the transition occurring at a slightly higher lateral acceleration. The TBS model does not predict a transition below 0.25 g of lateral acceleration, while the linear yaw plane model is unable to predict any changes in the handling behaviour of the vehicle.

From Table 3.5, it can be noted that there are noticeable differences between the articulation angles predicted using the four models. The Phase 4 model gives the highest prediction, while the linear yaw plane model gives the lowest prediction. It can also be seen that the roll angles of the sprung masses of the tractor and semitrailer predicted using the Phase 4 model are generally higher than those predicted using the yaw/roll model.

### 3.6 Vehicle Configuration 6

This vehicle is a tractor-semitrailer-full trailer combination. The steady-state steering response of this vehicle configuration was simulated using the Phase 4 model, the yaw/roll model and the linear yaw plane model at a forward speed of 21.5 mph (31.5 ft/sec). The TBS model was not used in the simulation, as it is not designed for simulating this type of vehicle combination in its present form.

The steady-state lateral accelerations and yaw rates as functions of front-wheel steering angle of the tractor predicted using the three computer simulation models are shown in Figs. 3.17 and 3.18, respectively. The handling characteristics of the vehicle predicted using the various models are illustrated in Fig. 3.19. The predicted values are also listed in Table 3.6.

As can be seen from Figs. 3.17 and 3.18 and from Table 3.6, all three simulation models give practically the same predictions of lateral acceleration and yaw rate of the tractor up to front-wheel steering angle of 4.5 degrees (or lateral acceleration of approximately 0.25 g). The differences between the lateral acceleration predicted by the Phase 4 model and those by the yaw/roll model and the linear yaw plane model at front-wheel steering angle of 4.5 degrees are only 0.5 % and 4.7 % respectively.

5-Axle Double/27-Ft Trailers (Conf. 6)

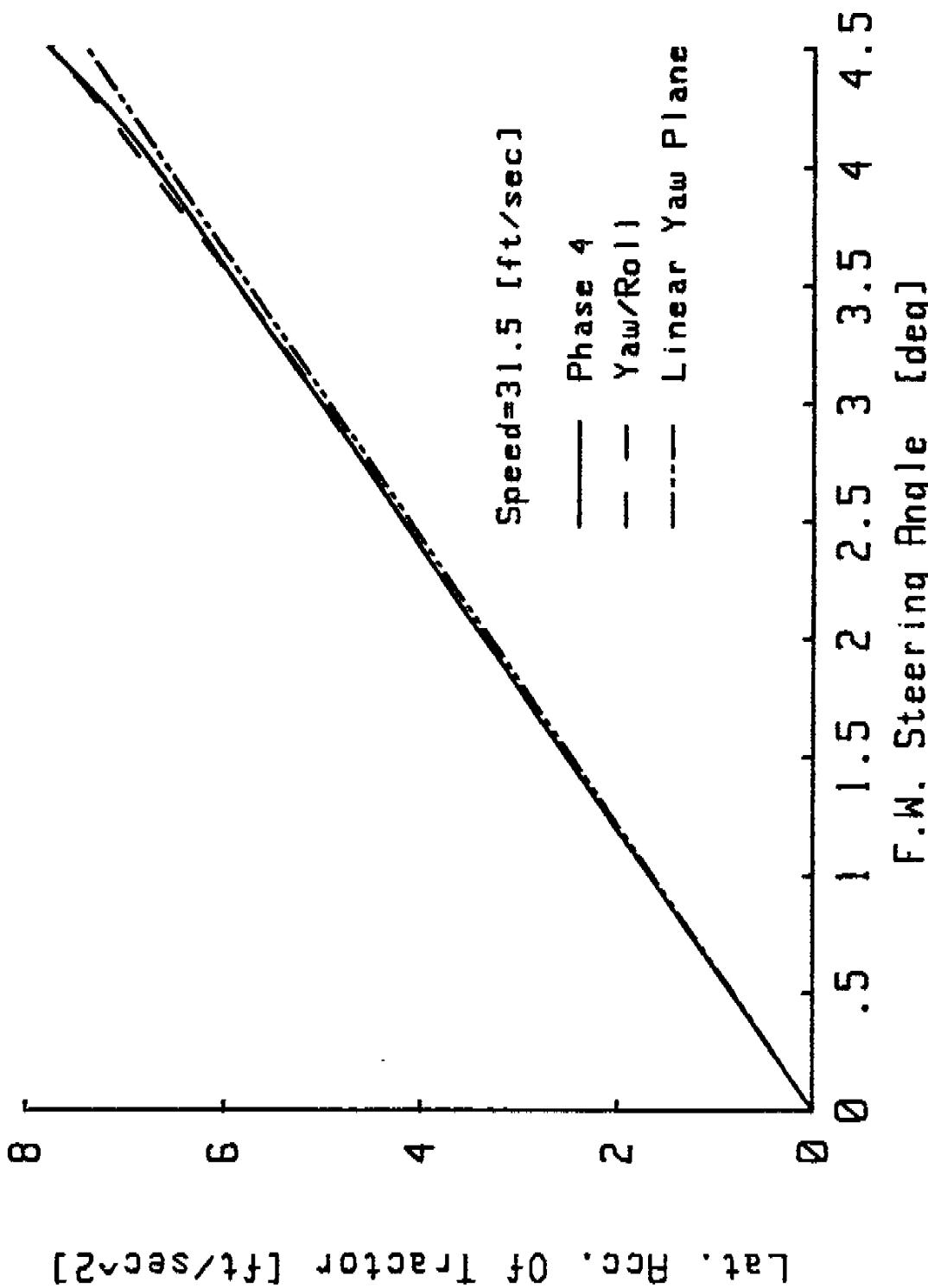


Fig. 3.17 Steady-state lateral acceleration response to steering input of Vehicle Configuration 6 predicted by various models

5-Axle Double/27-Ft Trailers (Conf. 6)

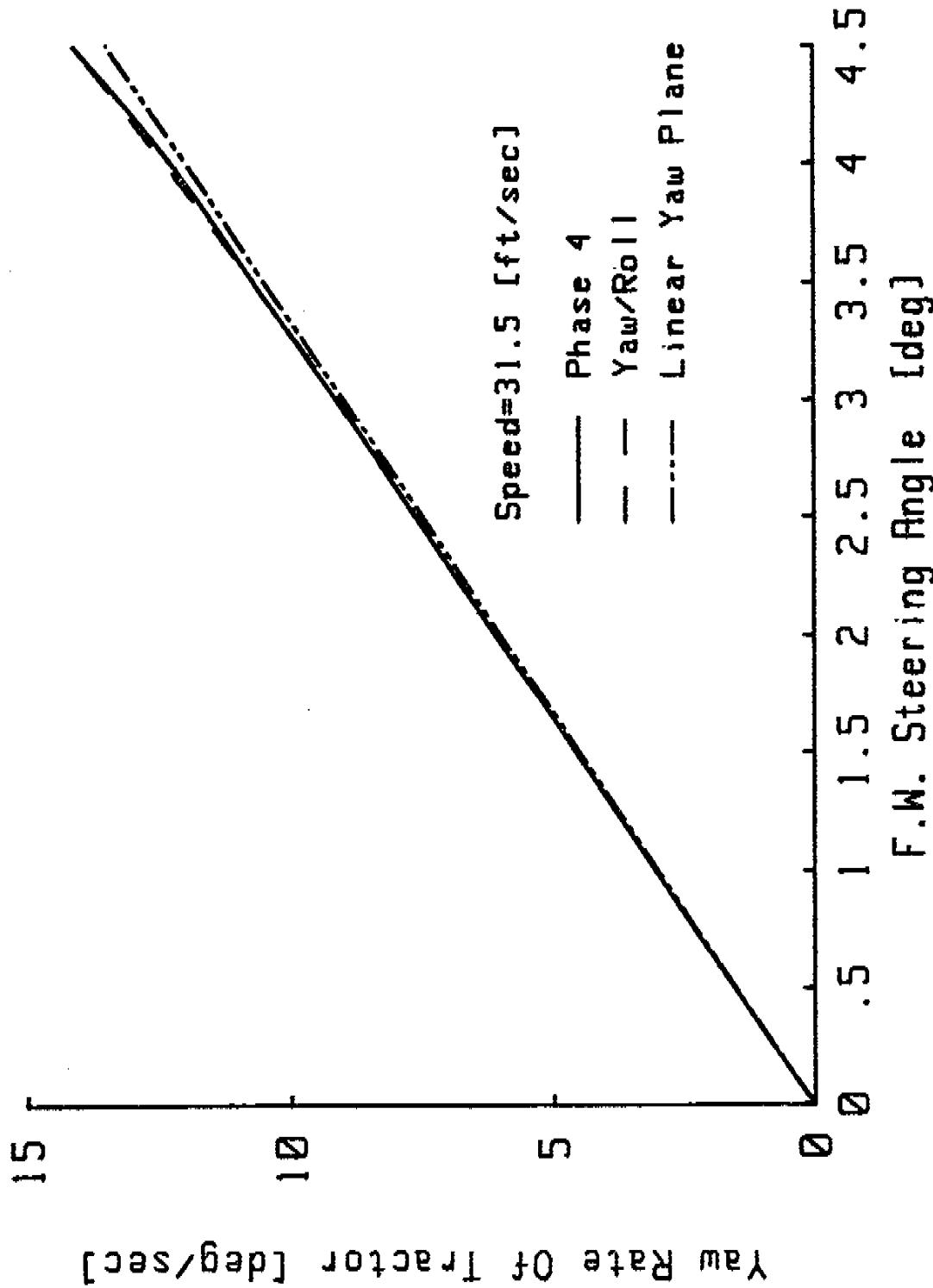


Fig. 3.18 Steady-state yaw rate response to steering input of Vehicle Configuration 6 predicted by various models

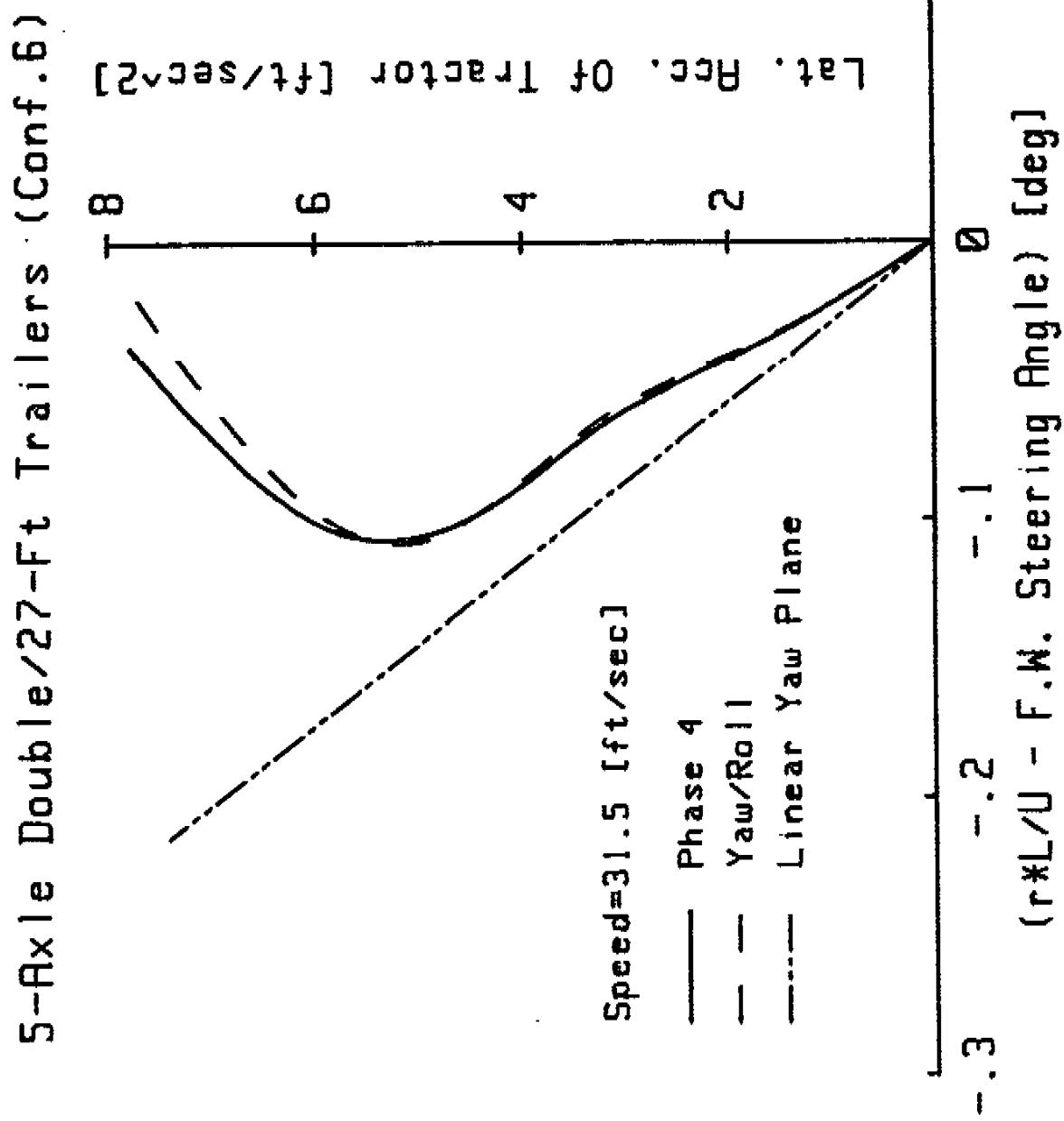


Fig. 3.19 Handling characteristics of Vehicle Configuration 6 predicted by various models

TABLE 3.6  
VEHICLE CONFIGURATION 6

A. Phase 4 Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec $^2$ )	$r^*l/U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\phi_3$ (deg)	$\Gamma_1$ (deg)	$\Gamma_2$ (deg)
1.0	31.53	3.04	4.67	- 0.036	- 0.51	- 0.43	- 0.35	1.93	2.89
2.0	31.55	6.09	3.36	- 0.069	- 1.25	- 1.09	- 0.88	3.880	5.83
2.5	31.7	7.63	4.19	- 0.093	- 1.63	- 1.42	- 1.20	4.88	7.47
3.0	31.51	9.14	5.03	- 0.098	- 2.07	- 1.79	- 1.48	5.82	8.99
3.5	31.5	10.7	5.88	- 0.103	- 2.45	- 2.15	- 1.79	6.79	10.56
4.0	31.5	12.28	6.72	- 0.101	- 2.94	- 2.61	- 2.10	7.82	14.04
4.5	31.68	14.14	7.80	- 0.037	- 3.96	- 3.14	- 2.60	8.90	

B. Yaw/Roll Model

$\delta_{av}$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec $^2$ )	$r^*l/U - \delta_{av}$ (deg)	$\phi_1$ (deg)	$\phi_2$ (deg)	$\phi_3$ (deg)	$\Gamma_1$ (deg)	$\Gamma_2$ (deg)
1.0	31.5	3.04	1.67	- 0.035	- 0.65	- 0.68	- 0.55	1.95	2.9
2.0	31.5	6.09	3.35	- 0.067	- 1.31	- 1.36	- 1.12	3.9	5.82
2.5	31.5	7.62	4.19	- 0.081	- 1.64	- 1.71	- 1.41	4.87	7.30
3.0	31.5	9.11	5.01	- 0.108	- 1.97	- 2.05	- 1.68	5.81	8.75
3.5	31.5	10.71	5.88	- 0.1	- 2.34	- 2.44	- 2.06	6.80	10.58
4.0	31.5	12.39	6.81	- 0.067	- 2.76	- 2.87	- 2.37	7.82	12.29
4.5	31.5	14.12	7.76	- 0.0175	- 3.18	- 3.32	- 2.74	8.88	14.07

### C. Linear Yaw Plane Model

$\delta$ (deg)	U (ft/sec)	r (deg/sec)	$A_y$ (ft/sec <sup>2</sup> )	$r^*l/U - \delta_{av}$ (deg)	$\Gamma_1$ (deg)	$\Gamma_2$ (deg)
1.0	31.5	3.0	1.65	- 0.048	1.91	2.91
2.0	31.5	6.0	3.3	- 0.095	3.82	5.84
3.0	31.5	8.99	4.95	- 0.146	5.74	8.86
4.5	31.5	13.5	7.43	- 0.214	8.59	13.09

NOTE:  $\phi_1$  - roll angle of the tractor

$\phi_2$  - roll angle of the semitrailer

$\phi_3$  - roll angle of the full-trailer

$\Gamma_1$  - articulation angle between the tractor and the semitrailer

$\Gamma_2$  - articulation angle between the semitrailer and the full-trailer

As can be seen from Fig. 3.19, the Phase 4 model and the yaw/roll model give essentially the same prediction of the handling characteristics of the vehicle. Both models predict a transition from understeer to oversteer at a lateral acceleration of approximately 0.15 g. The linear yaw plane model predicts a higher level of understeer than the other two models. As mentioned previously, the linear yaw plane model is unable to predict changes in the handling behaviour of the vehicle.

From Table 3.6, it can be seen that all three models give similar predictions of the articulation angle between the tractor and semitrailer and that between the semitrailer and full-trailer. It can also be noted that the Phase 4 model and the yaw/roll model give similar predictions of the roll angles of the sprung masses of the tractor, semitrailer and full-trailer.

#### 4. A COMPARISON OF VARIOUS COMPUTER SIMULATION MODELS FOR PREDICTING STEERING RESPONSES IN LANE-CHANGE TYPE TRANSIENT MANOEUVRES

This section examines the abilities of various computer simulation models in predicting the directional response to steering in lane-change (obstacle-avoidance) type transient manoeuvres. The lateral acceleration, yaw rate and articulation angle of Vehicle Configuration 1 in a moderate lane-change type transient manoeuvre were predicted using the Phase 4 model, the yaw/roll model, the TBS model and the linear yaw plane model. The simulated results were compared with the measured data reported in reference (7). In addition, the directional responses to steering of Vehicle Configurations 3 and 6 in a more severe lane-change manoeuvre were also simulated using the four computer simulation models. The results of this study provide quantitative information for evaluation the abilities of various computer models for simulating the transient directional behaviour of articulated vehicles.

#### 4.1 Steering Response of Vehicle Configuration 1 in a Lane-Change Manoeuvre

The transient directional response to steering input of Vehicle Configuration 1 was predicted using the four computer simulation models. In the simulations, the left and right front-wheel angles as functions of time measured during a test, shown in Fig. 4.1, were used as inputs (7). The tractor lateral acceleration, semitrailer lateral acceleration, tractor yaw rate, semitrailer yaw rate, and articulation angle of Vehicle Configuration 1 as functions of time during the lane-change manoeuvre, at a forward speed of 43 mph (63.04 ft/sec), were predicted using the four models, and are shown in Figs. 4.2, 4.3, 4.4, 4.5 and 4.6, respectively. For comparison, the measured data reported in reference (7) were also shown in the figures.

It can be seen that the responses of the tractor and semitrailer predicted by the four models generally follow the same trend as that measured. However, there are differences between the predicted peak values and the measured ones. For instance, the differences between the measured peak value of tractor lateral acceleration and those predicted using the Phase 4 model, the yaw/roll model, the TBS model and the linear yaw plane model are approximately 16%, 10%, 33% and 46%, respectively, as shown in Fig. 4.2. The agreement between the measured peak value of semitrailer lateral acceleration and those predicted is better than that for the tractor lateral acceleration. The differences between the measured peak value of semitrailer lateral acceleration and those predicted using the Phase 4 model, the yaw/roll model, the TBS model and the linear yaw plane model are approximately 20%, 8%, 12% and 20%, respectively, as can be seen from Fig. 4.3.

## Measured Front Wheel Steering Inputs

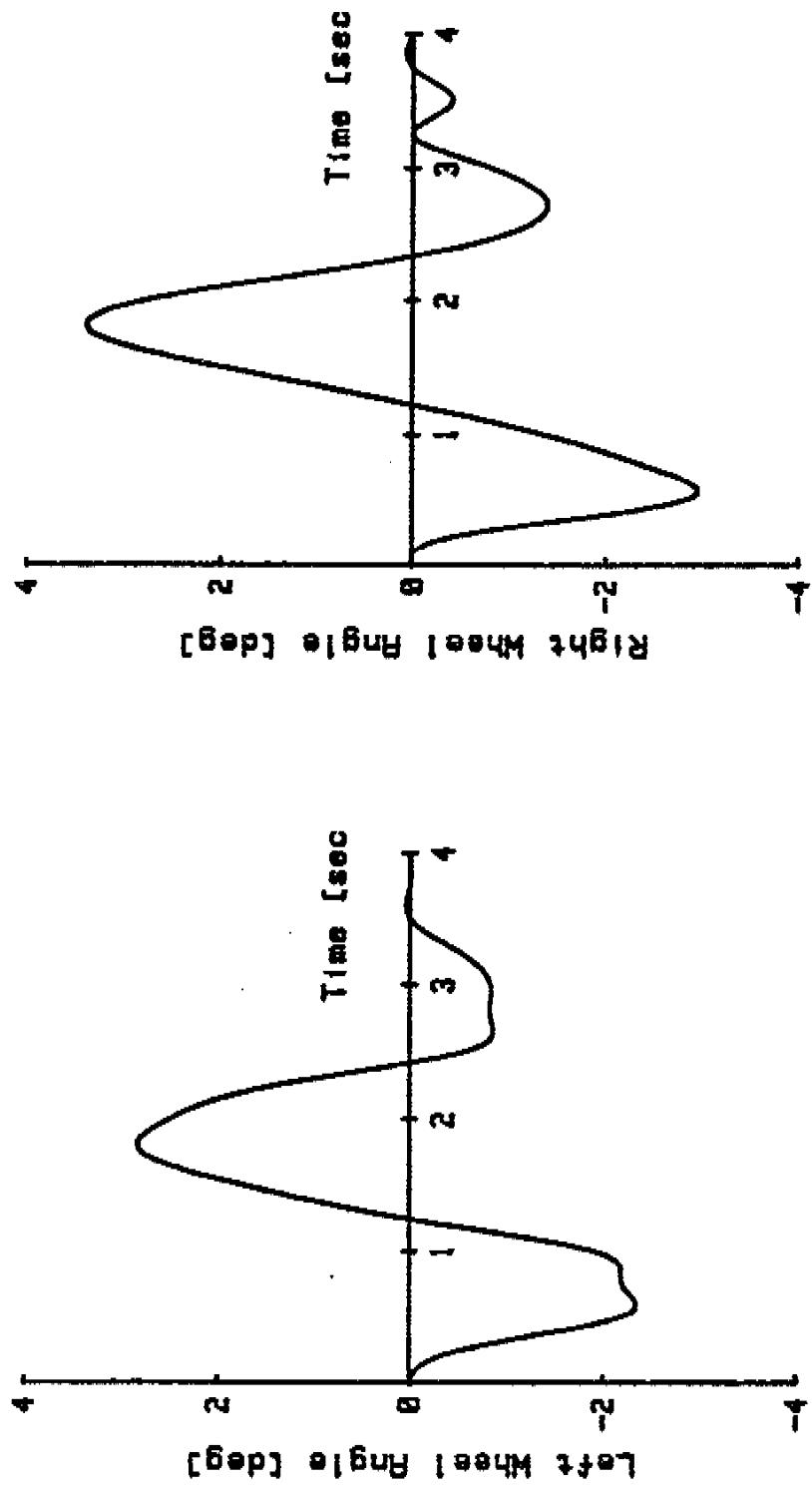


Fig. 4.1 Front-wheel steering inputs to the simulation of moderate lane-change manoeuvres

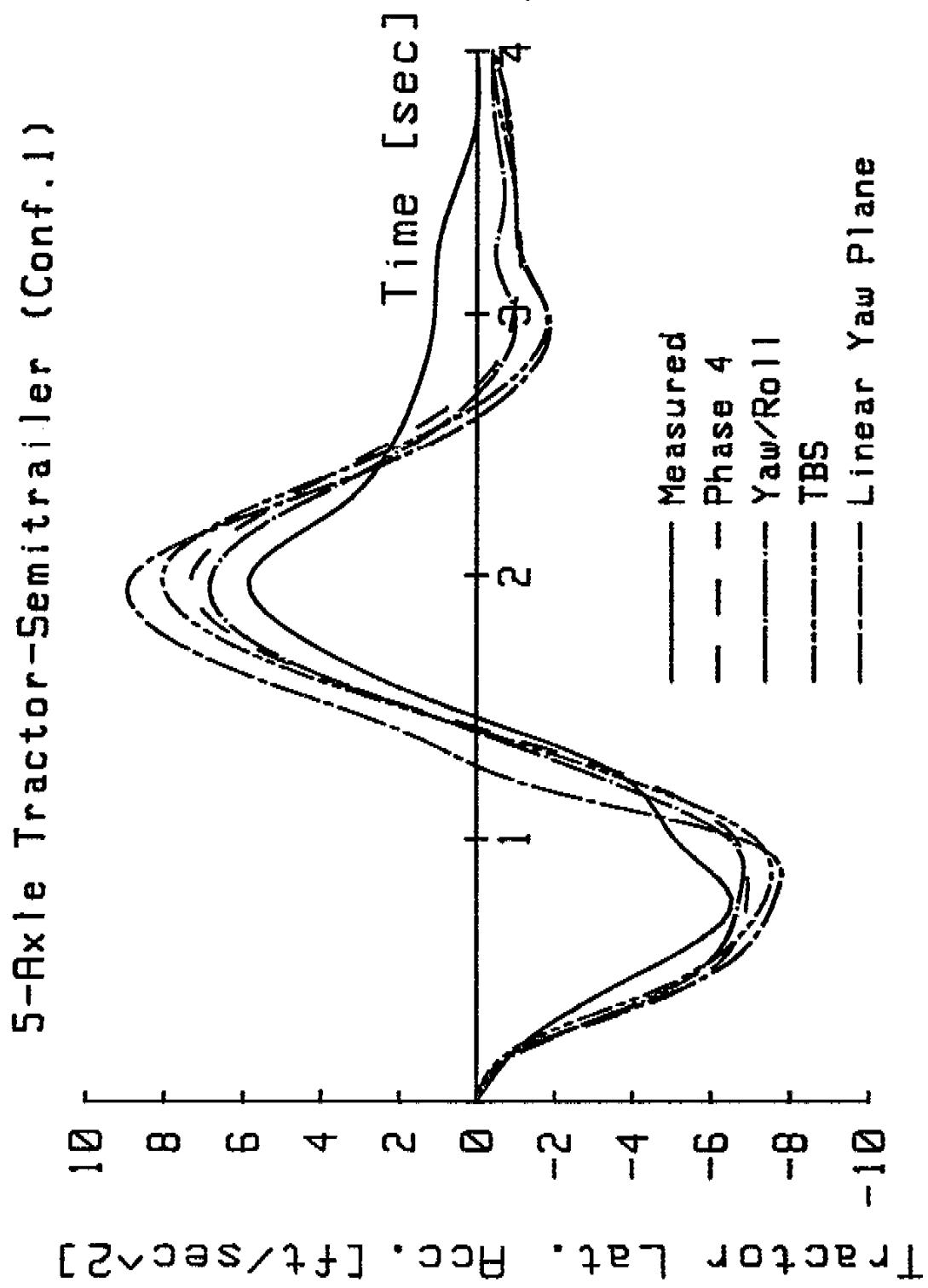


Fig. 4.2 Variation of tractor lateral acceleration with time of Vehicle Configuration 1 in a lane-change manoeuvre predicted by various models

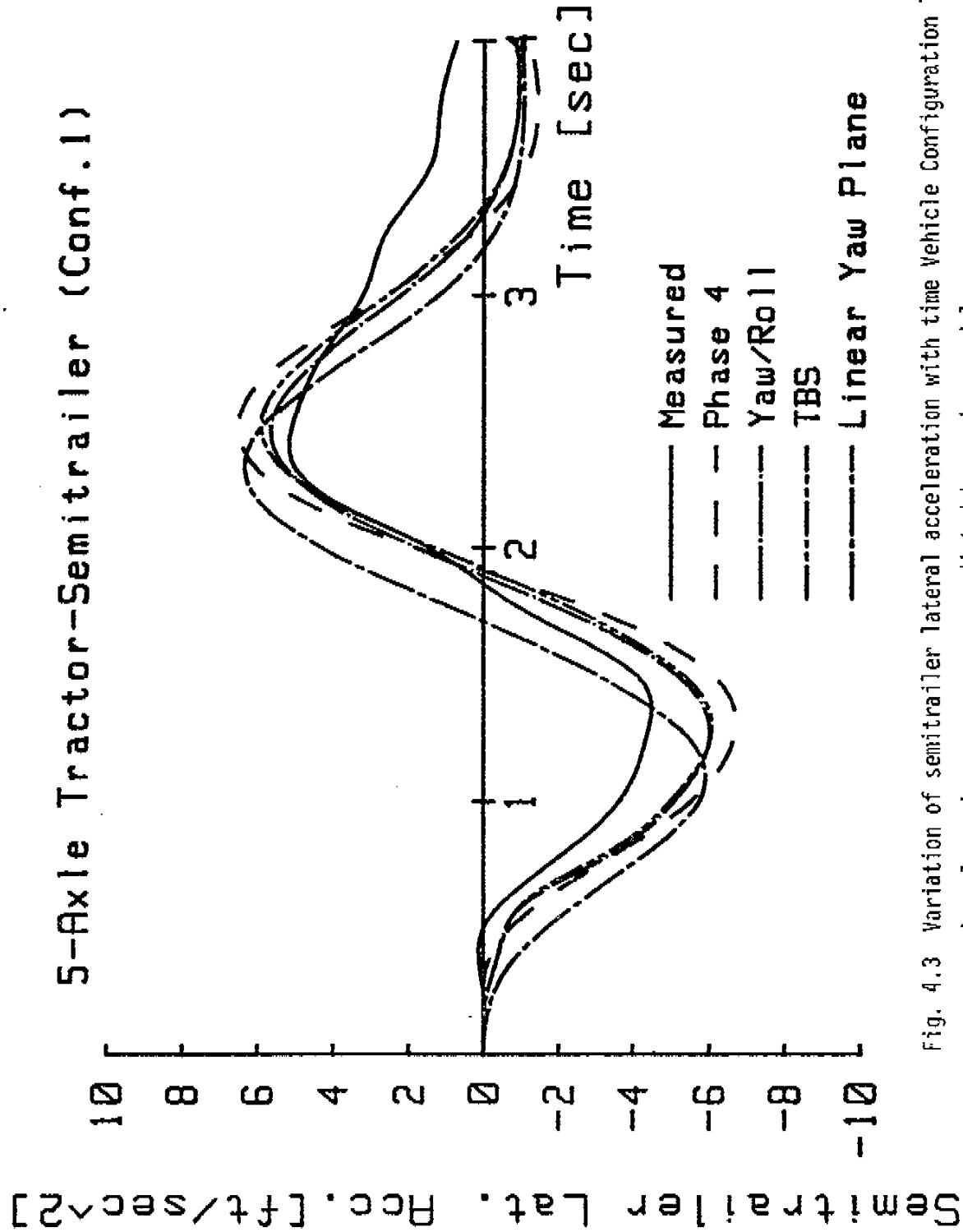


Fig. 4.3 Variation of semitrailer lateral acceleration with time Vehicle Configuration 1  
in a lane-change manoeuvre predicted by various models

### 5-Axle Tractor-Semitrailer (Conf. 1)

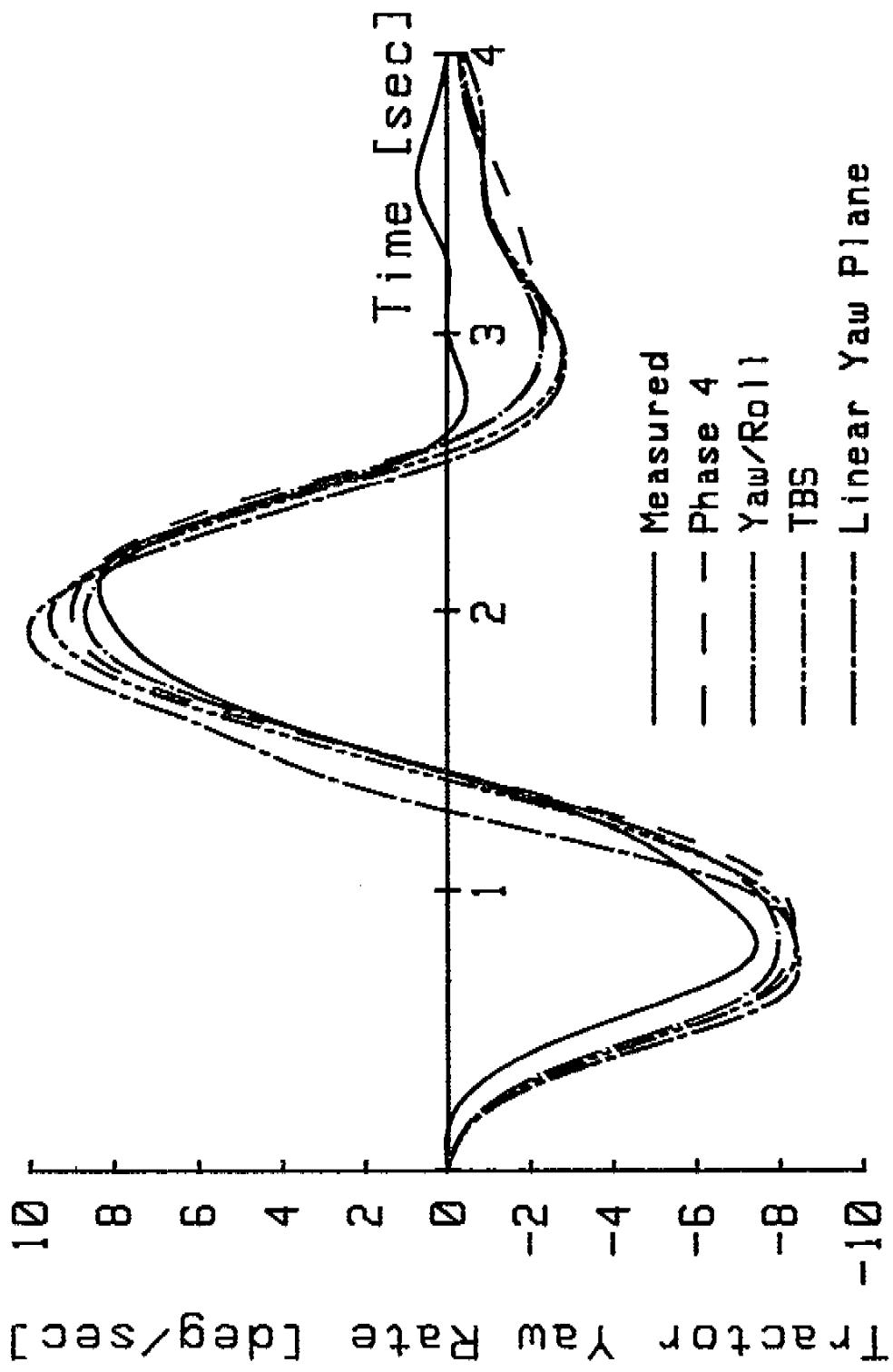


Fig. 4.4 Variation of tractor yaw rate with time of Vehicle Configuration 1 in a lane-change manoeuvre predicted by various models

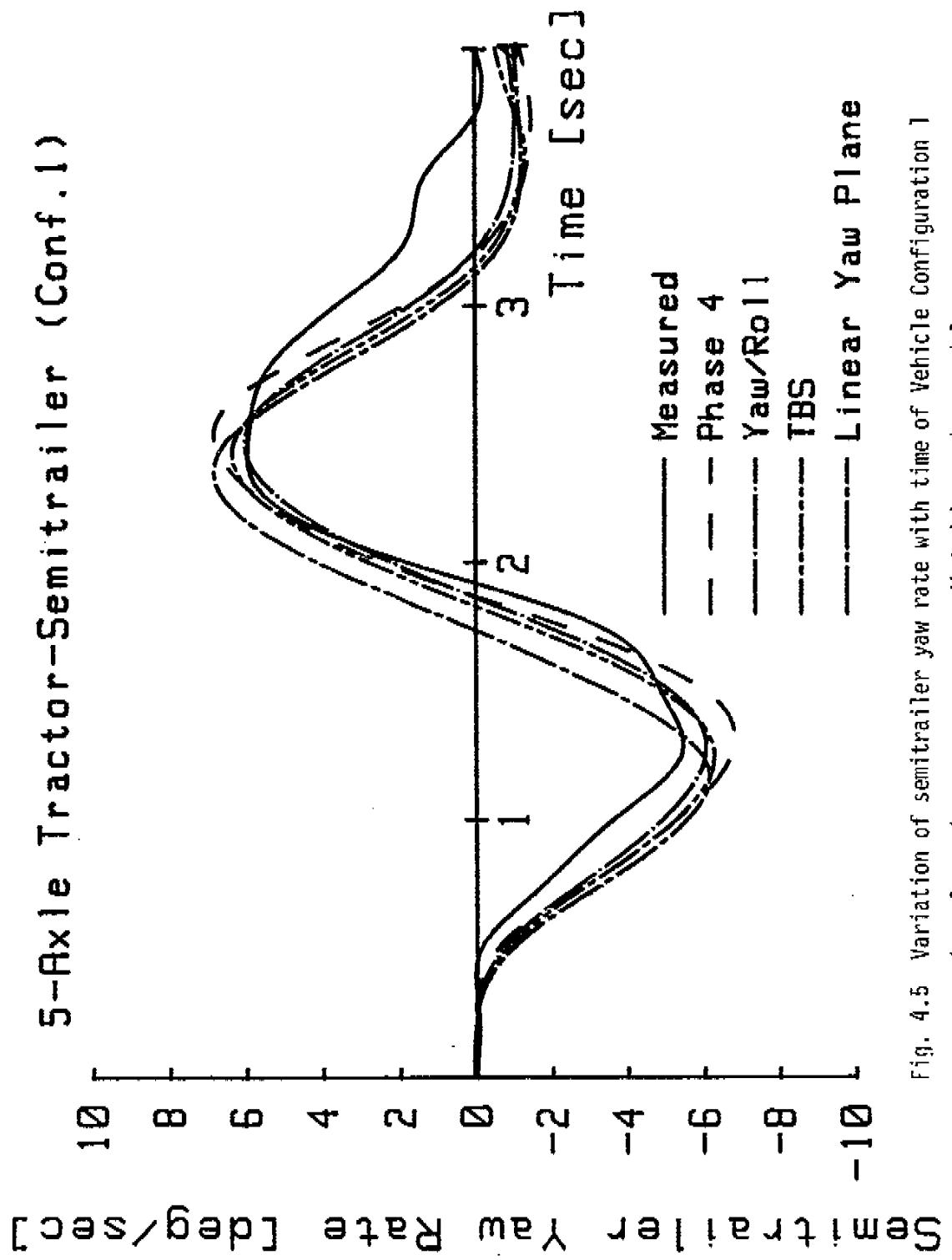


Fig. 4.5 Variation of semitrailer yaw rate with time of Vehicle Configuration 1 in a lane-change manoeuvre predicted by various models

### 5-Axle Tractor-Semitrailer (Conf. 1)

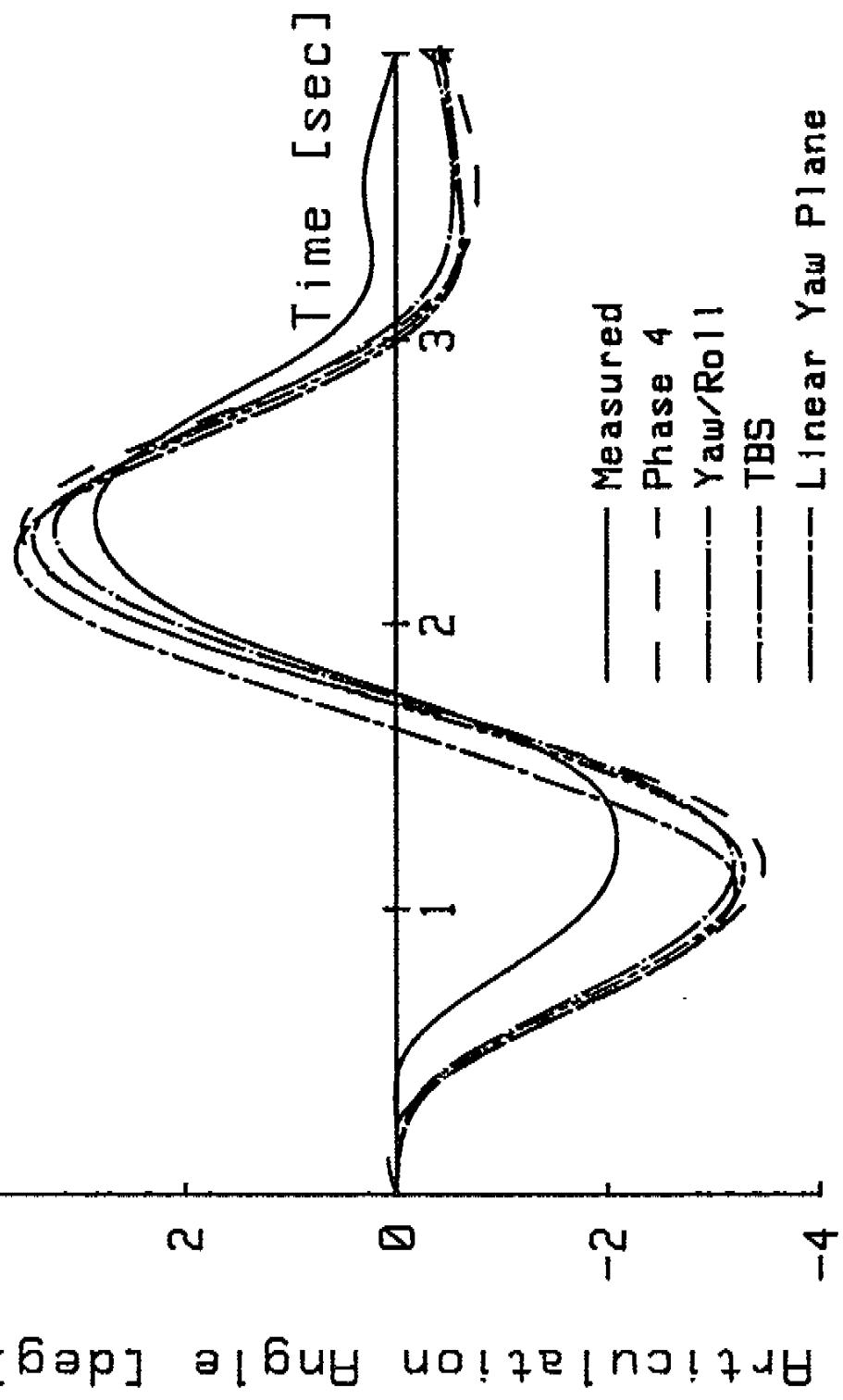


Fig. 4.6 Variation of articulation angle with time of Vehicle Configuration 1 in a lane-change manoeuvre predicted by various models

The agreement between the measured tractor yaw rate response and those predicted using the four models appears to be reasonable. The differences between the measured peak value of tractor yaw rate and those predicted using the Phase 4 model, the yaw/roll model, the TBS model, and the linear yaw plane model are approximately 9.8%, 4.9%, 15.8% and 21.9%, respectively, as can be seen from Fig. 4.4. The measured semitrailer yaw rate response and those predicted using the four models again show reasonable agreement. The differences between the measured peak value of semitrailer yaw rate and those predicted using the Phase 4 model, the yaw/roll model, the TBS model and the linear yaw plane model are approximately 13.3 %, 0 %, 3.3 %, and 13.3 %, respectively, as shown in Fig. 4.5. The articulation angle responses predicted using the four models are reasonably close. The differences between the peak values of articulation angle predicted using the four models are within 10 %. However, there is a noticeable difference between the measured and predicted peak values of articulation angle. For instance, the difference between the measured and the predicted peak value of articulated angle using the Phase 4 model is approximately 22 %, as can be seen from Fig. 4.6.

It should also be noted from the figures that there is a phase shift between the measured and predicted responses, and that there is a significant difference between the measured and predicted responses during the period from 2.5 to 4 seconds.

#### 4.2 Steering Response of Vehicle Configuration 3 in a Severe Lane-Change Manoeuvre

The transient directional response to steering input of Vehicle Configuration 3 (a 5-axle tractor-semitrailer) in a severe lane-change (obstacle-avoidance) type manoeuvre at a forward speed of 35.8 mph (52.5 ft/sec) was predicted using the four computer simulation models. In the simulations, the time history of the average front-wheel steering angle shown in Fig. 4.7 was used as input to the computer simulation models. It can be noted from Fig. 4.7 that the amplitude of the front-wheel steering angle is 6 degrees and that the steering input in the lane-change manoeuvre is completed in 2 seconds. This represents a severe (rapid) lane-change type manoeuvre, as compared with that described in Section 4.1.

The tractor yaw rate, semitrailer yaw rate, tractor lateral acceleration and semitrailer lateral acceleration of Vehicle Configuration 3 predicted using various computer simulation models are shown in Figs. 4.8, 4.9, 4.10 and 4.11, respectively. There is no measured data available for comparison. The information shown in the figures, therefore, only serves the purposes of illustrating the difference in the predictions made by various models.

It can be seen from Figs. 4.8 and 4.9 that the yaw rate responses of the tractor and semitrailer predicted using the four computer simulation models are essentially the same. The differences in the peak values of yaw rate of the tractor predicted using the four models are less than 9.3 %, while those of the semitrailer are within the range of 5.7 %.

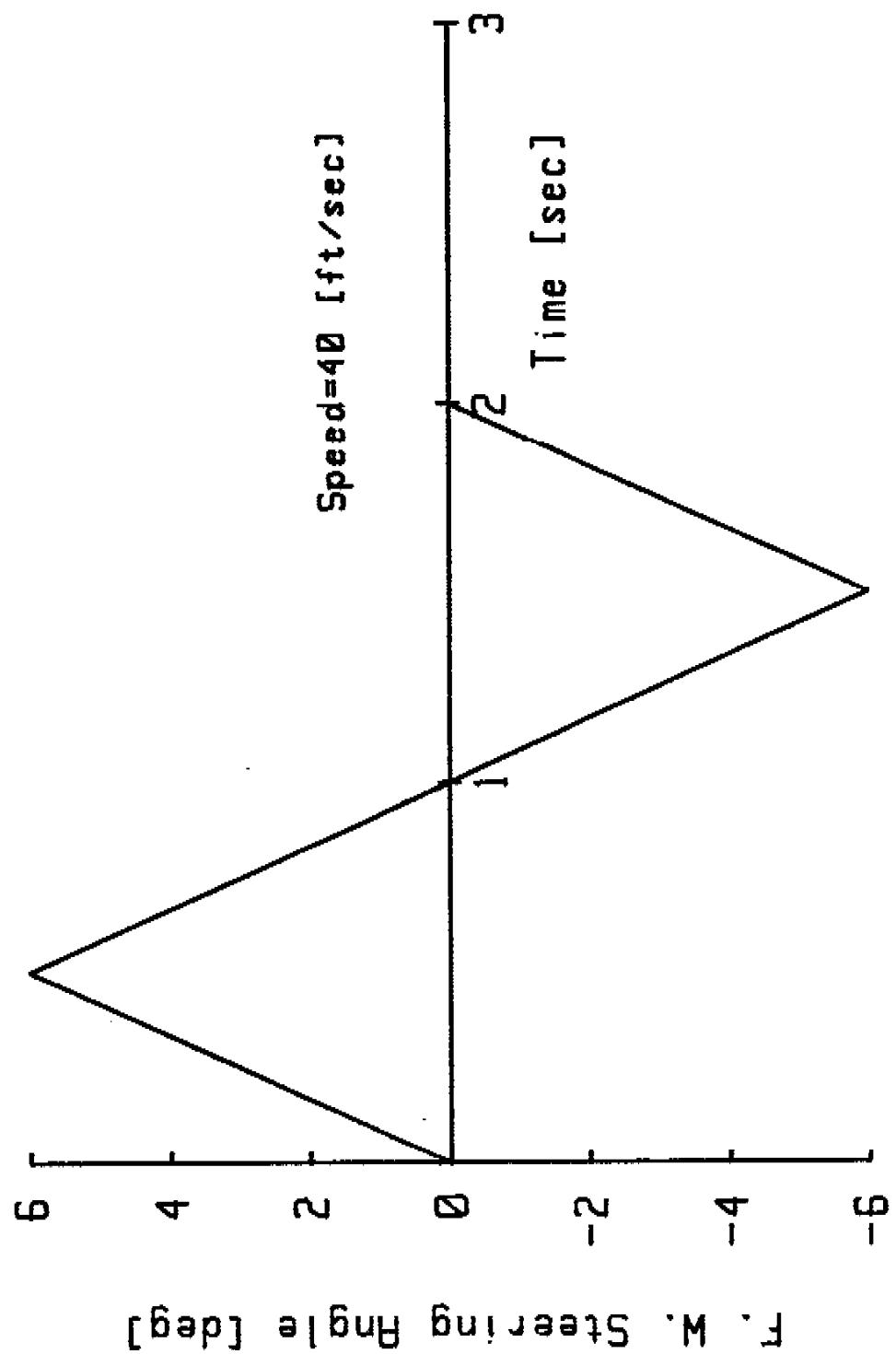


Fig. 4.7 Front-wheel steering inputs to the simulation of severe (rapid) lane-change manoeuvres

### 5-Axle Tractor-Semitrailer (Conf. 3)

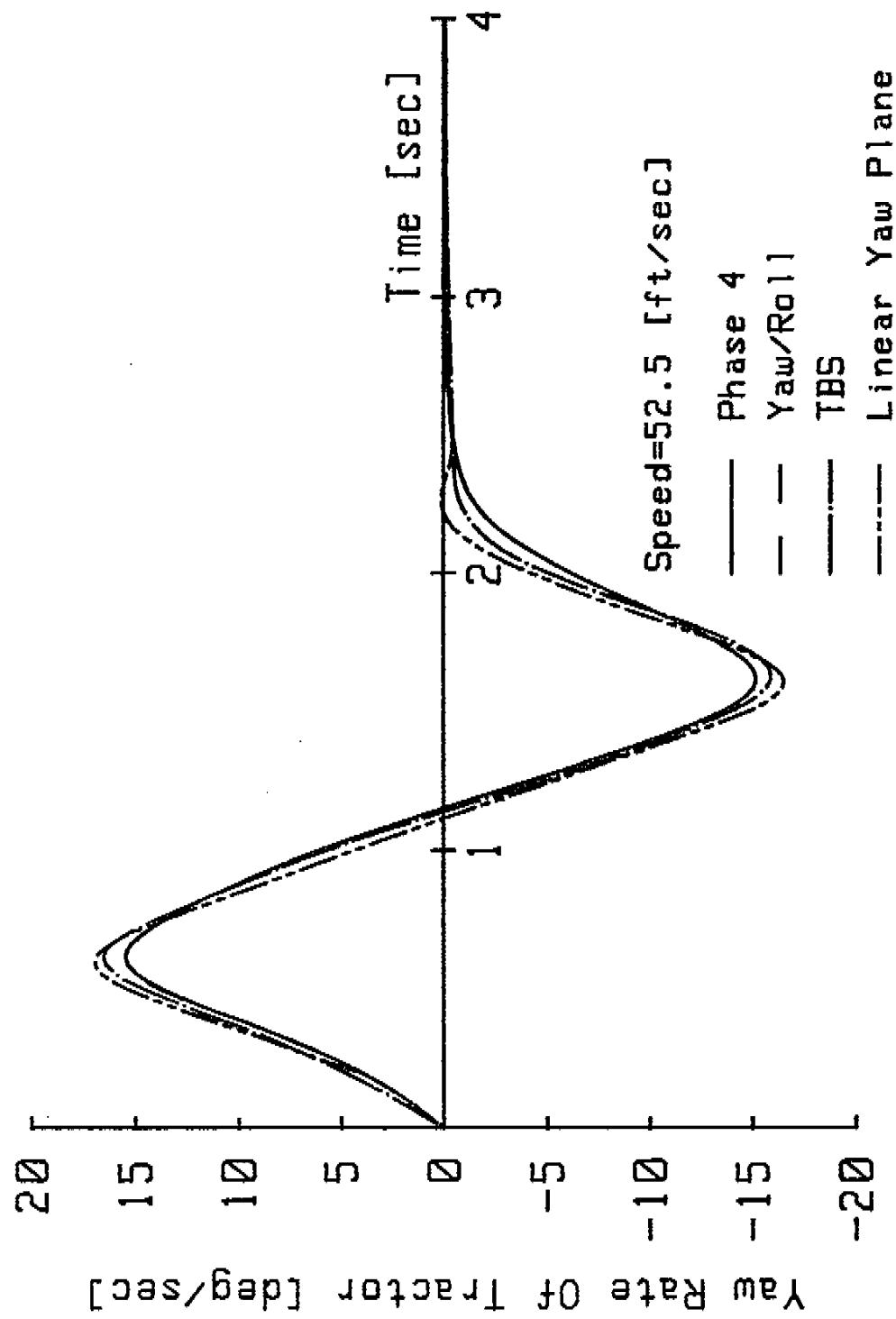


Fig. 4.8 Variation of tractor yaw rate with time of Vehicle Configuration 3 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Tractor-Semitrailer (Conf. 3)

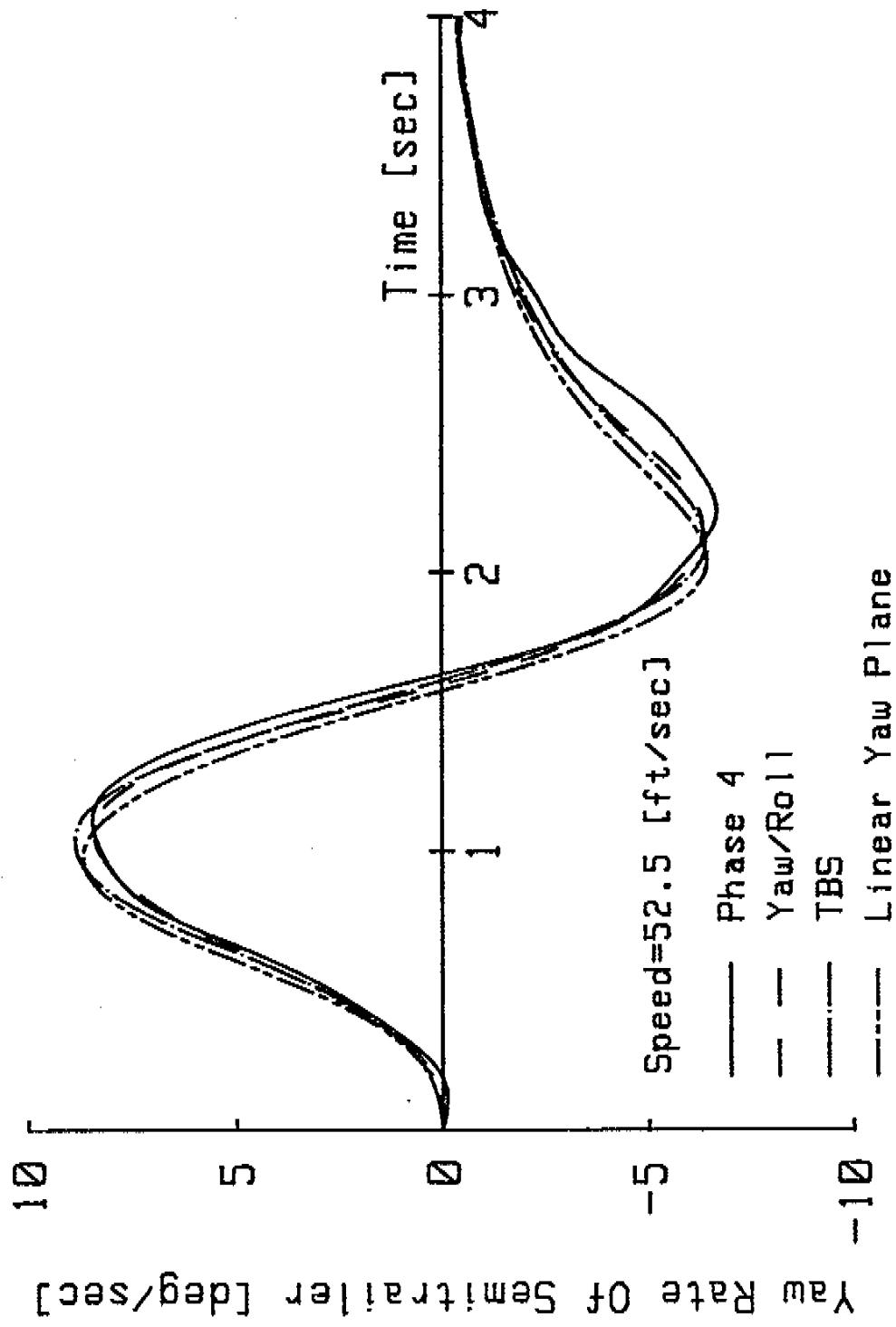


Fig. 4.9 Variation of semitrailer yaw rate with time of Vehicle Configuration 3 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Tractor-Semi trailer (Conf. 3)

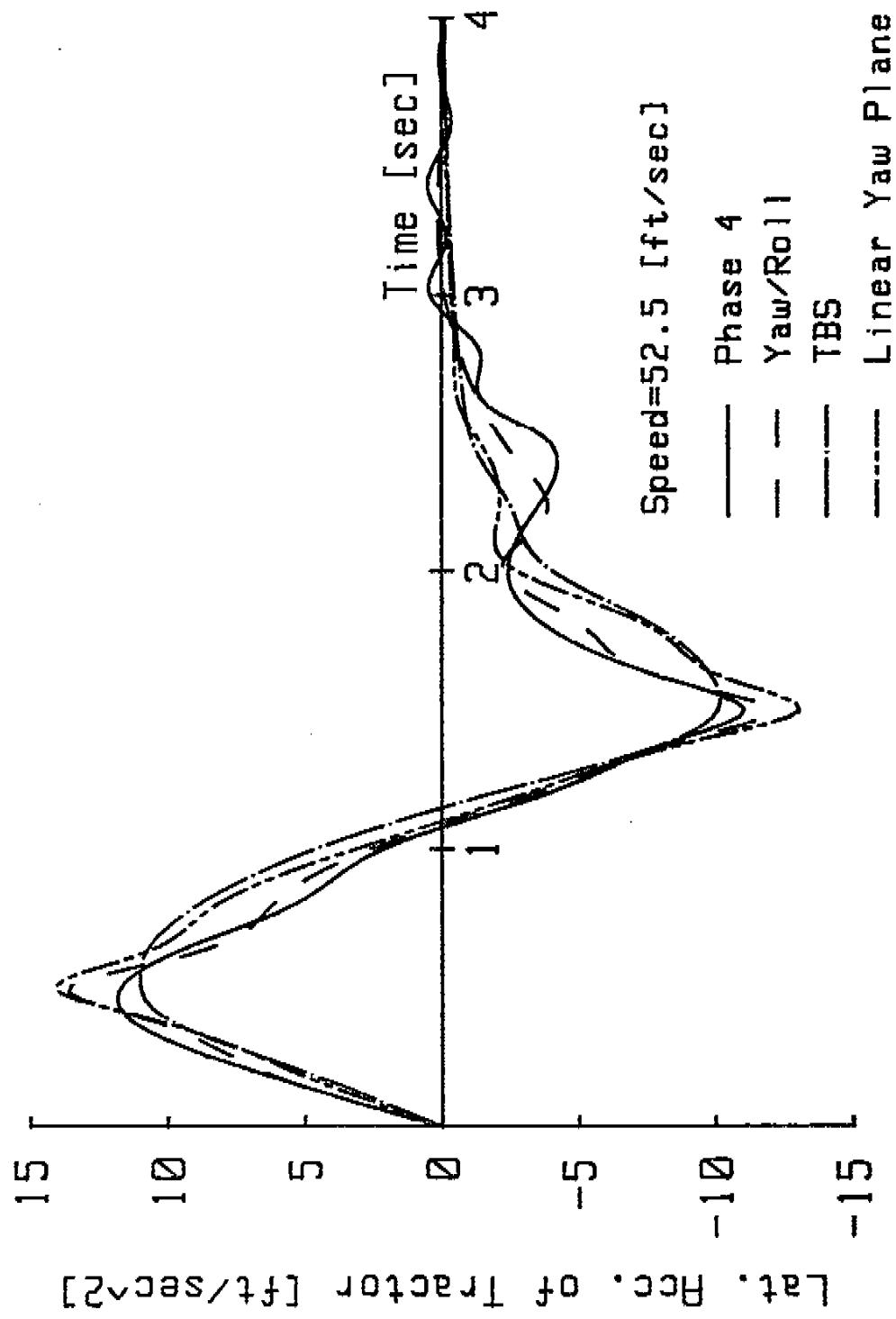


Fig. 4.10 Variation of tractor lateral acceleration with time of Vehicle Configuration 3 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Tractor-Semitrailer (Conf. 3)

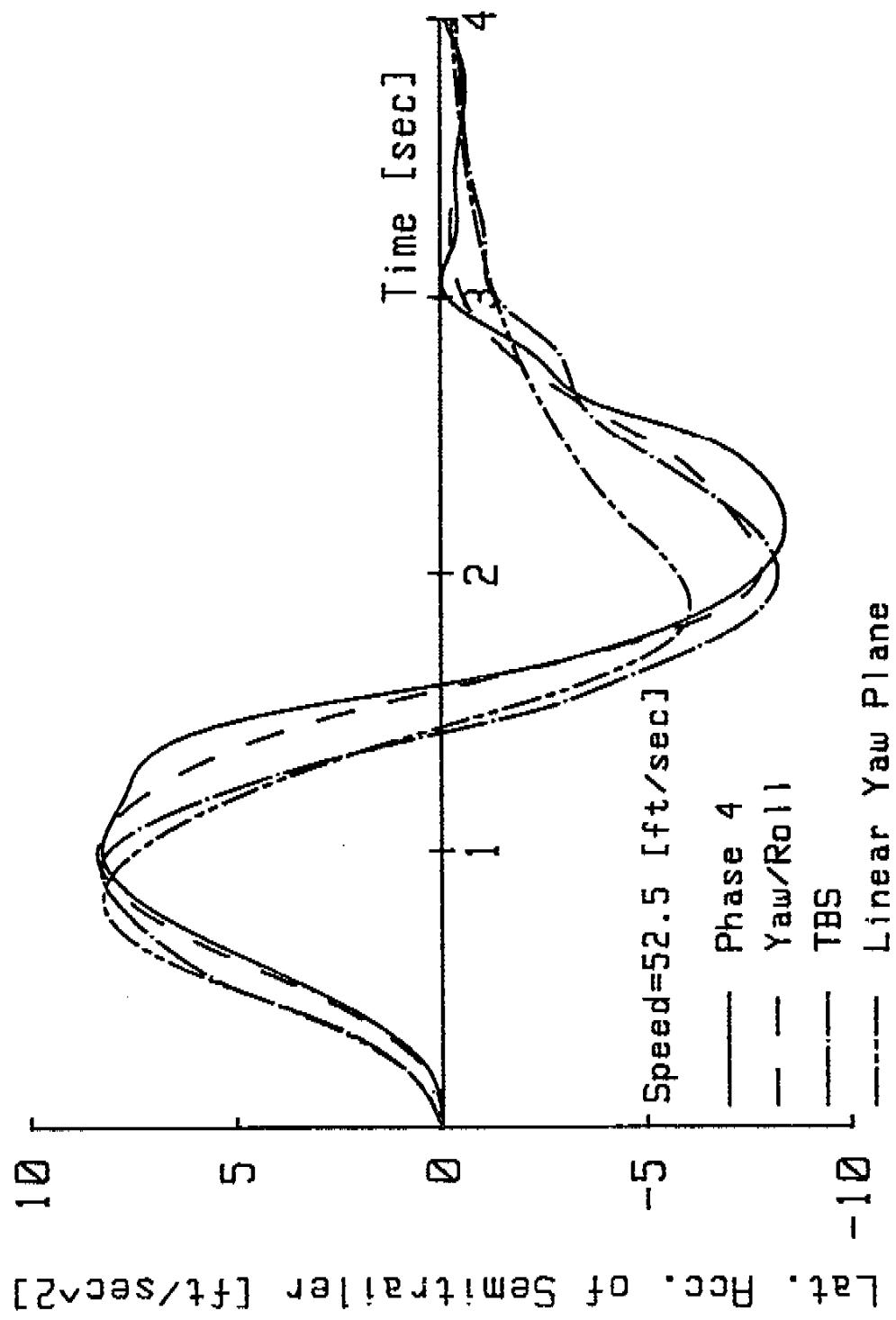


Fig. 4.11 Variation of semitrailer lateral acceleration with time of Vehicle Configuration 3 in a severe lane-change manoeuvre predicted by various models

As can be seen from Figs. 4.10 and 4.11, the characteristics of the lateral acceleration response of the tractor and semitrailer predicted using the four computer simulation models show noticeable differences in phase and in peak value. For instance, the differences in the peak value of tractor lateral acceleration predicted using the Phase 4 model and those predicted using the yaw/roll model, the TBS model and the linear yaw plane model are 15.2 %, 7.1 % and 19.2 %, respectively.

#### 4.3 Steering Response of Vehicle Configuration 6 in a Severe Lane-Change Manoeuvre

The transient directional response to steering input of Vehicle Configuration 6 (a tractor-semitrailer-full trailer) in a severe lane-change manoeuvre at a forward speed of 27.3 mph (40 ft/sec), was predicted using the Phase 4 model, the yaw/roll model, and the linear yaw plane model. The TBS model was not used in the simulation, as it is not designed, in its present form, for simulating this type of vehicle combination. The steering input to the various simulation models is the same as that shown in Fig. 4.7.

The yaw rates of the tractor, semitrailer and full trailer (pup trailer) and the lateral accelerations of the tractor, semitrailer and full trailer of Vehicle Configuration 6 predicted using the three computer simulation models are shown in Figs. 4.12, 4.13, 4.14, 4.15, 4.16 and 4.17, respectively.

It can be seen from Figs. 4.12 and 4.13 that the yaw rate responses of the tractor and semitrailer predicted using the three computer simulation models are quite close. The differences in the peak values of yaw rate of the tractor predicted using the three models are less than 9.8 %, while those of the semitrailer are less than 2.8 %. The yaw rate responses of the full trailer (pup trailer) predicted using the Phase 4 model and the yaw/roll model are essentially identical, as can be seen from Fig. 4.14. However, there is a significant difference in the yaw rate response characteristics predicted using the linear yaw plane model and those predicted using the Phase 4 model and the yaw/roll model, and the difference in the peak value of yaw rate is approximately 15 %.

### 5-Axle Double/27-Ft Trailers (Conf. 6)

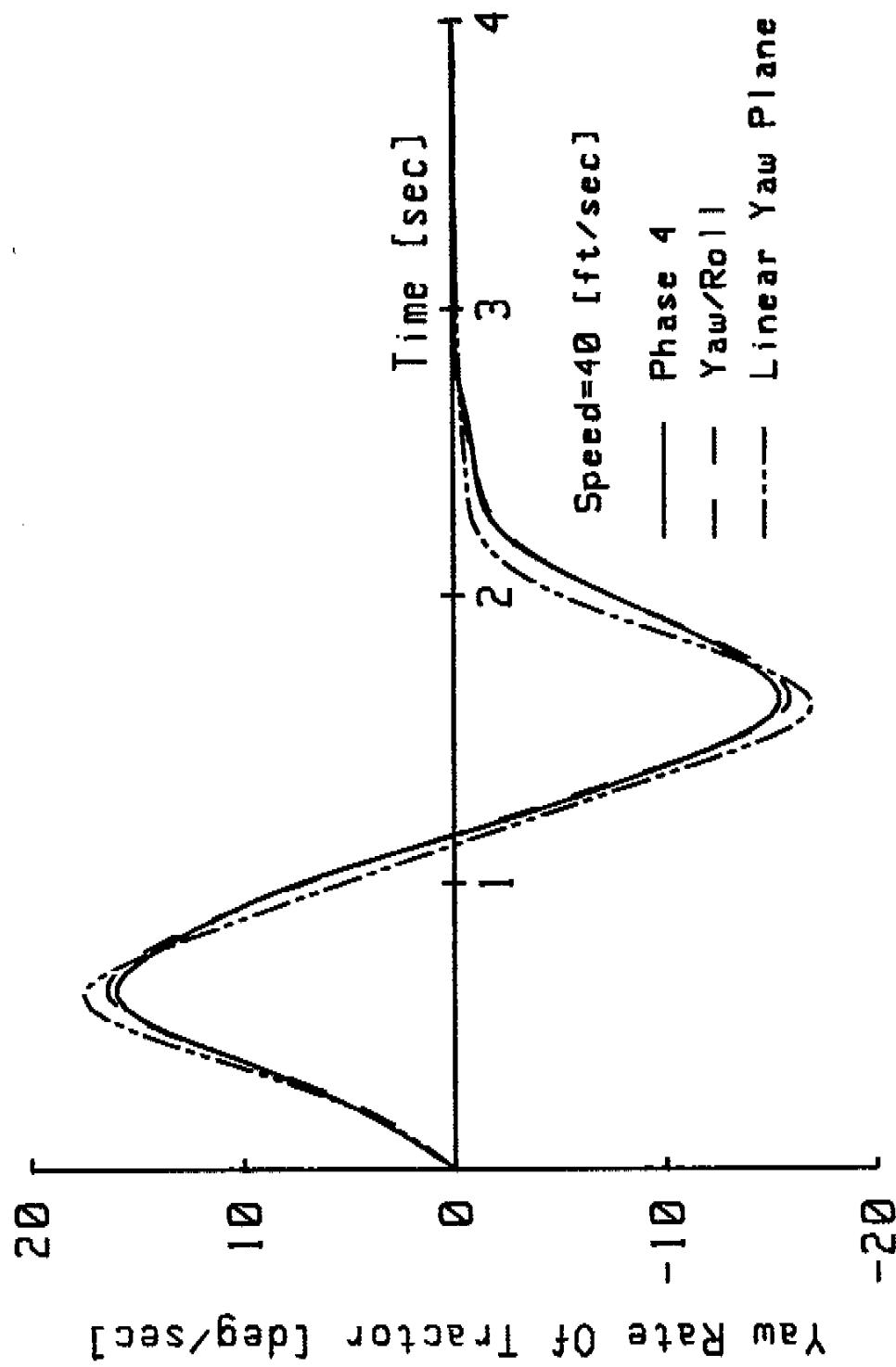


Fig. 4.12 Variation of tractor yaw rate with time of Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

5-Axle Double/27-Ft Trailers (Conf. 6)

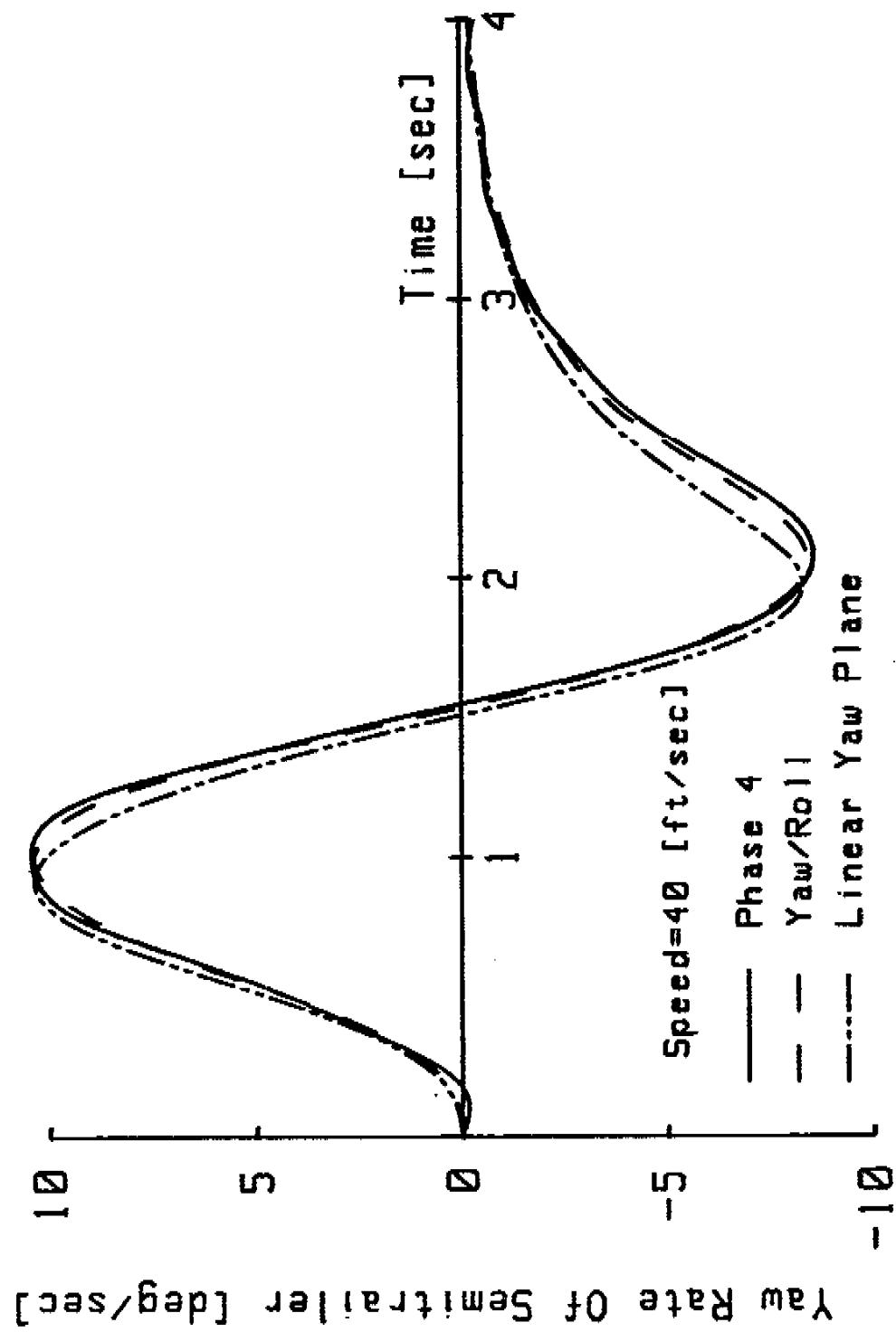


Fig. 4.13 Variation of semitrailer yaw rate with time of Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Double/27-ft Trailers (Conf. 6)

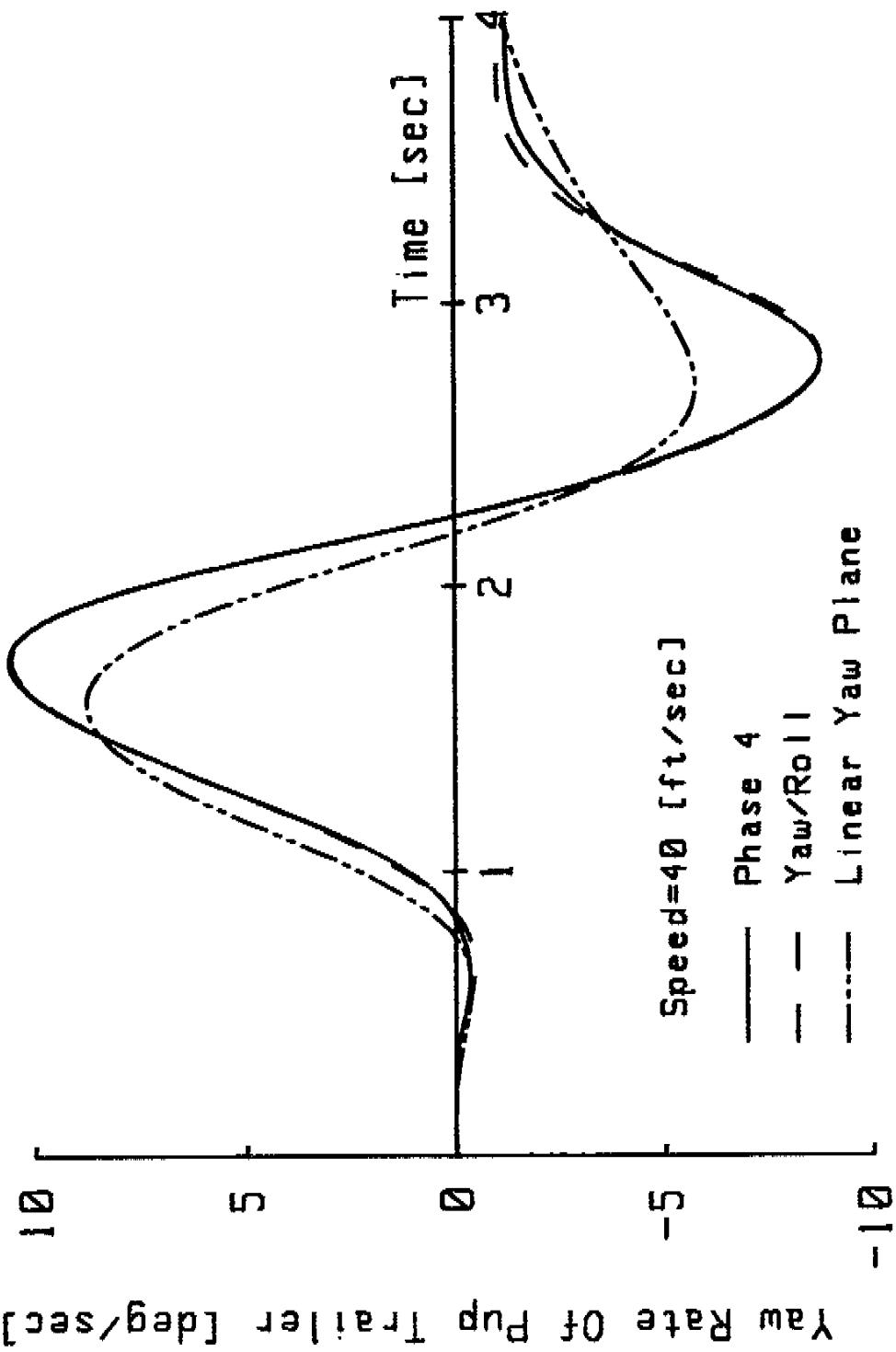


Fig. 4.14 Variation of pup trailer yaw rate with time of Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Double/27-Ft Trailers (Conf. 6)

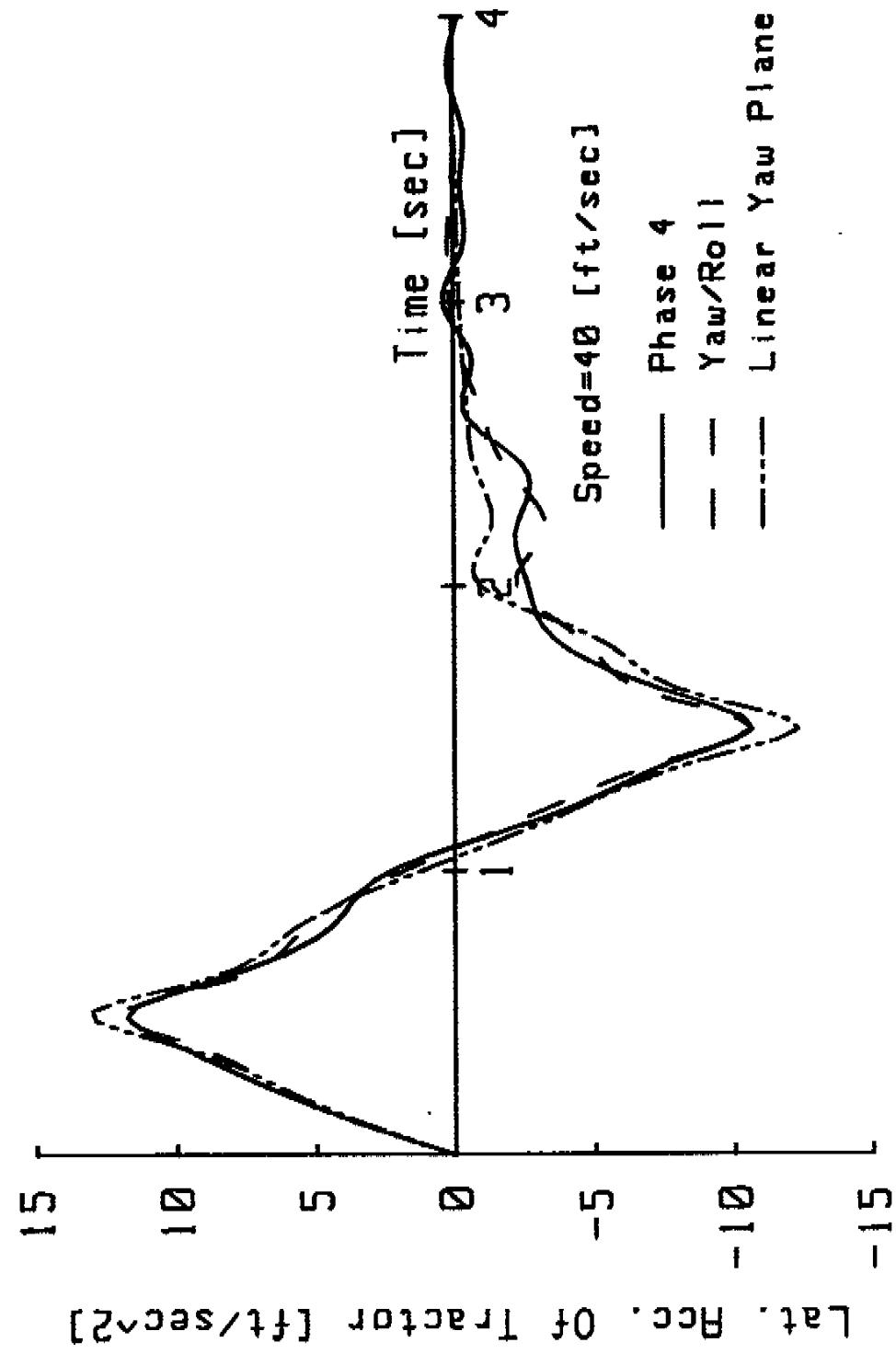


Fig. 4.15 Variation of tractor lateral acceleration with time of Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Double/27-Ft Trailers (Conf. 6)

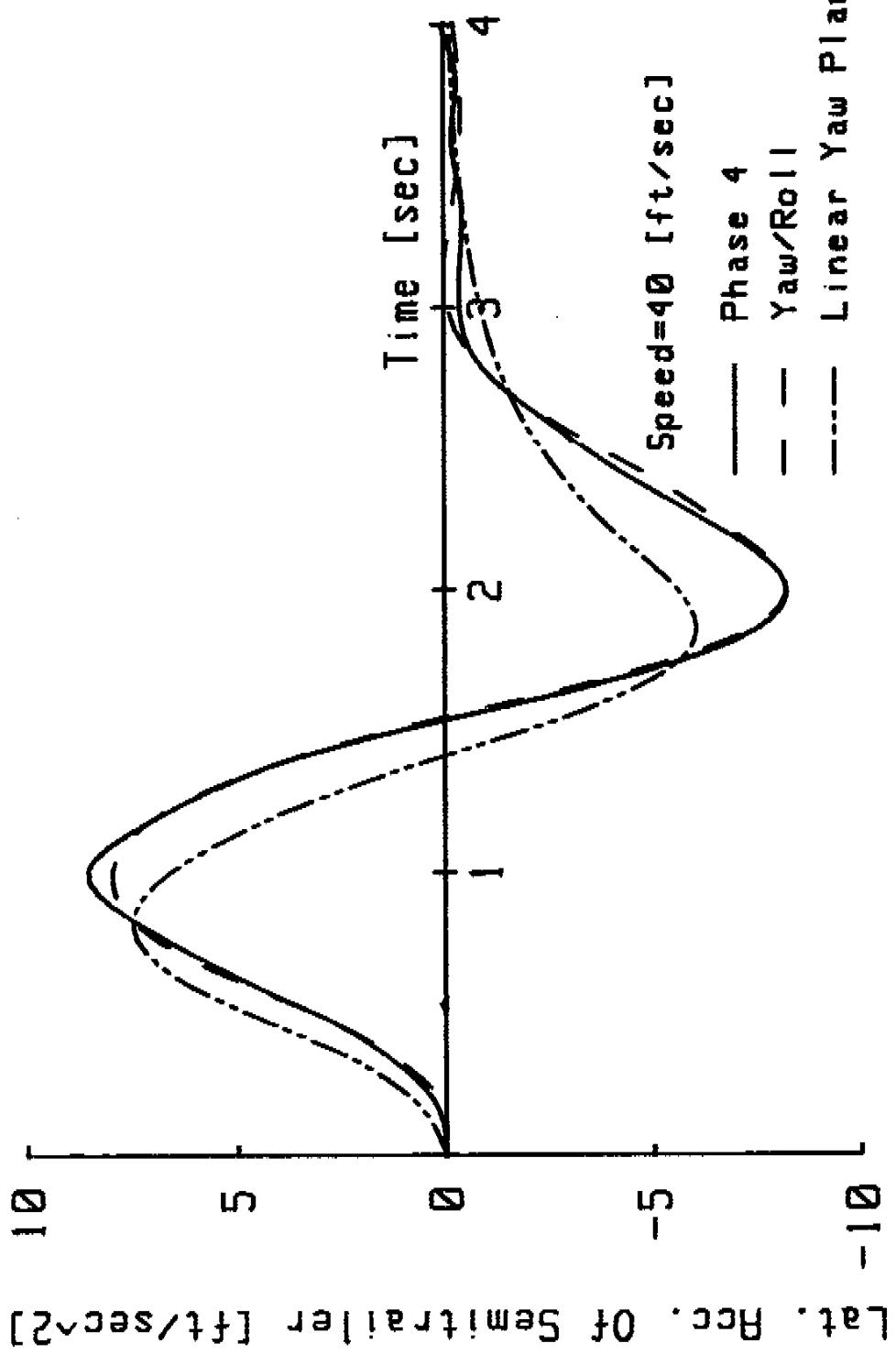


Fig. 4.16 Variation of semitrailer lateral acceleration with time of Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

### 5-Axle Double/27-ft Trailers (Conf. 6)

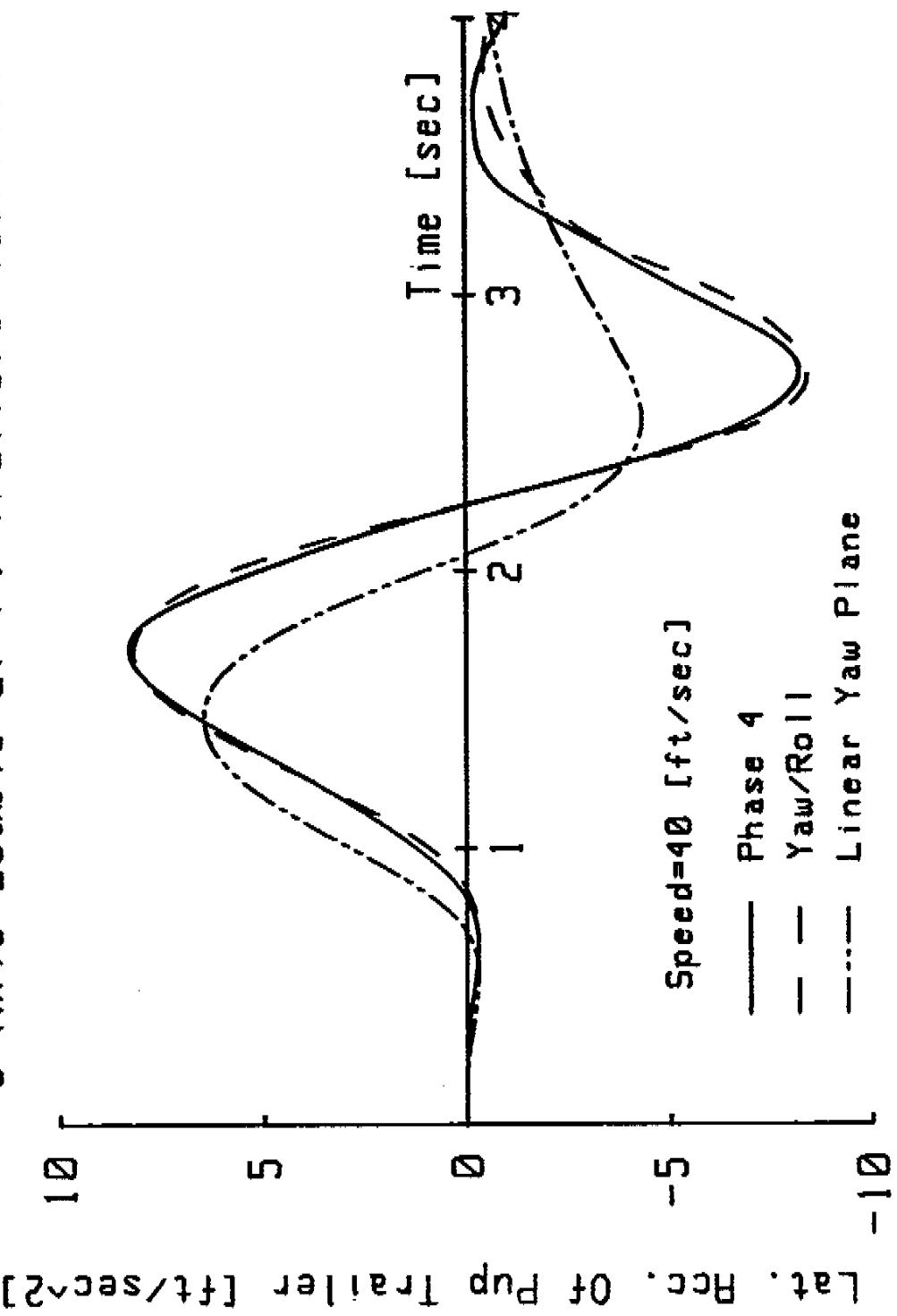


Fig. 4.17 Variation of pup trailer lateral acceleration with time for Vehicle Configuration 6 in a severe lane-change manoeuvre predicted by various models

From Fig. 4.15, it can be seen that the lateral acceleration responses of the tractor predicted using the three computer simulation models are similar, and that the differences in the peak value of lateral acceleration are within 9.9 %. The lateral acceleration responses of the semitrailer and of the full trailer(pup trailer)predicted using the Phase 4 model and the yaw/roll model are essentially the same, as can be seen from Figs. 4.16 and 4.17. However, there is a noticeable difference between the characteristics of the lateral acceleration responses of the semitrailer and the full trailer predicted using the linear yaw plane model and those predicted using the other two models. For instance, the difference between the peak value of lateral acceleration of the semitrailer predicted using the linear yaw plane model and that predicted using the other two models is approximately 13 %, while that for the full trailer is approximately 22.6 %. Furthermore, there is a significant phase shift between the responses predicted using the linear yaw plane model and those predicted using the Phase 4 model and the yaw/roll model.

## 5. A PARAMETRIC SENSITIVITY STUDY OF THE STATIC ROLL MODEL

As mentioned previously in Section 1.5, the static roll model was developed for the purpose of determining the rollover threshold of articulated vehicles during steady-state turning manoeuvres. This model is based on a number of simplifying assumptions. However, it requires a considerable amount of vehicle input data, some of which can only be obtained using special apparatus. For instance, the model requires, among others, the following vehicle parameters as inputs:

- A. Tractor frame torsional stiffness;
- B. Coulomb friction present in the tractor frame;
- C. Equivalent trailer structural and fifth wheel compliance;
- D. Separation of fifth wheel plates (Lash)

If no existing data are available for the vehicle to be simulated, then these parameters have to be measured. For some of the parameters, such as the equivalent tractor frame torsional stiffness, even if the equipment is available, there appears to be a lack of generally accepted procedures for measuring it, as the value of the equivalent torsional stiffness depends on the loading conditions as well as the type of constraint used in the tests. It appears, therefore, that while the model represents a simplification of the actual vehicle system, the acquisition of appropriate input data to the model could be quite involved. Thus, it seems useful to conduct an evaluation of the significance of certain input parameters to the simulated results. If it is found that the simulated results are not sensitive to certain input parameters, then the considerable effort that may be required to measure them may not be justified. This would also provide guidance

for the revision of the original model, so as to make it more useful to the practising engineer.

In this section, the sensitivity of the simulated results to the four input parameters mentioned above is evaluated. In the evaluation, a 5-axle dirt truck (from File 25 of a UMTRI magnetic tape dated November 25, 1981) was used. The basic parameters of this vehicle are given in Appendix G. In the study, the values of the four parameters are varied within a certain range and the corresponding simulated results are obtained. The results of the sensitivity study are described below.

### 5.1 Tractor Frame Torsional Stiffness

The effects of the equivalent torsional stiffness of the tractor frame on the rollover threshold, expressed in terms of the maximum lateral acceleration ( $a_y \text{ max}$ ) allowed without causing rollover, were examined using the static roll model. Three values of the equivalent torsional stiffness: 40,000 (measured value for the 5-axle, dirt truck), 500,000 and 9,000,000 in-lb/deg (equivalent to a very rigid tractor frame) were used in the simulation. The results are shown in Table 5.1.

Table 5.1  
Effects of Tractor Frame Torsional Stiffness

Tractor frame torsional stiffness, in-lb/deg	Rollover threshold, $a_y \text{ max}$ , g	Roll angle of the first sprung mass, $\phi_1$ , deg	Roll angle of second sprung mass, $\phi_2$ , deg	Roll angle of third sprung mass, $\phi_3$ , deg
40,000	0.35	6.01	5.40	5.78
500,000	0.35	5.5	5.42	5.80
9,000,000	0.35	5.47	5.42	5.80

It can be seen from Table 5.1 that within the range of 40,000 to 9,000,000 in-lb/deg, the torsional stiffness of the tractor frame has no effect whatsoever on the rollover threshold of the vehicle simulated. It has a minor effect on the simulated roll angle of the first sprung mass. However, the difference is less than 9 % when the torsional stiffness varies from 40,000 to 9,000,000 in-lb/deg. Table 5.1 also indicates that

the torsional stiffness of the tractor frame has essentially no effect on the simulated roll angles of the second and third sprung masses.

## 5.2 Coulomb Friction Present in the Tractor Frame

The effects of the Coulomb type friction present in the tractor frame on the rollover threshold of the vehicle were evaluated using the static roll model. Three values of Coulomb friction: 11,000 (measured value for the 5-axle, dirt truck), 5,500 and 0 lb, were used in the simulations. The results are given in Table 5.2.

Table 5.2  
Effects of Coulomb Friction Present in the Tractor Frame

Coulomb friction, lb	Rollover threshold, $a_y$ max, g	Roll angle of first sprung mass, $\phi_1$ , deg	Roll angle of second sprung mass, $\phi_2$ , deg	Roll angle of third sprung mass, $\phi_3$ , deg
11,000	0.35	6.01	5.4	5.78
5,500	0.35	6.11	5.4	5.78
0	0.35	6.22	5.4	5.78

As can be seen from Table 5.2, within the range of 0 to 11,000 lb, the Coulomb friction present in the tractor frame has no effect at all on the rollover threshold, and the roll angles of the second and third sprung masses of the vehicle simulated. It has a very slight effect on the simulated roll angle of the first sprung mass. However, the difference

is less than 3.5 %, when the Coulomb friction varies from 0 to 11,000 lb.

### 5.3 Equivalent Trailer Structural and Fifth Wheel Compliance

The effects of the equivalent trailer structural and fifth wheel compliance on the rollover threshold of the vehicle were assessed using the static roll model. Three values of the equivalent torsional stiffness of the trailer structure and fifth wheel: 1,000,000 (measured value for the 5-axle dirt truck), 500,000, and 9,000,000 in-lb/deg (equivalent to rigid), were used in the simulations. The results are summarized in Table 5.3.

Table 5.3  
Effects of Equivalent Trailer Structural and Fifth Wheel Compliance

Equivalent torsional stiffness of trailer structure & 5th wheel, in-lb/deg	Rollover threshold, $a_y \text{ max } g$	Roll angle of the 1st sprung mass, $\phi_1$ , deg.	Roll angle of the 2nd sprung mass, $\phi_2$ , deg	Roll angle of the 3rd sprung mass, $\phi_3$ , deg
1,000,000	0.35	6.01	5.4	5.78
500,000	0.35	5.72	5.02	5.78
9,000,000	0.35	6.26	5.74	5.78

As shown in Table 5.3, the equivalent trailer structural and fifth wheel compliance has no effect whatsoever on the rollover threshold and the roll angle of the third sprung mass. It has only a slight effect on

the simulated roll angles of the first and the second sprung masses,  $\phi_1$  and  $\phi_2$ . The difference in the simulated roll angle  $\phi_1$  is approximately 4.8 %, when the equivalent torsional stiffness is reduced from 1,000,000 to 500,000 in-lb/deg, while the difference is 4.2 %, when the stiffness is increased from 1,000,000 to 9,000,000 in-lb/deg. For the simulated roll angle  $\phi_2$ , the difference is approximately 7 %, when the torsional stiffness changes from 1,000,000 to 500,000 in-lb/deg, whereas the difference is 6.3 % when the stiffness varies from 1,000,000 to 9,000,000 in-lb/deg.

#### 5.4 Separation of Fifth Wheel Plates (Lash)

The effects of the separation of fifth wheel plates (lash) on the roll behaviour of the vehicle were evaluated using the static roll model. In the simulation, the lash was varied from 0 (measured value for the 5-axle, dirt truck) to 2 in. It is found that within this range, the lash has practically no effect on the rollover threshold and the roll angles of the first, second, and third sprung masses.

The effects of the overturning stiffness of the tires on the roll behaviour of the vehicle were also examined. The measured values of the overturning stiffness of the tires on the first axle, second axle and third axle of the 5-axle, dirt truck, were 1,000, 2,000, and 2,000 in-lb/deg, respectively. No effect on the rollover threshold of the vehicle has been found by ignoring the overturning stiffness of the tires.

After the individual effects of the tractor frame torsional stiffness, Coulomb friction present in the tractor frame and trailer structural and fifth wheel compliance and separation of the fifth wheel plates had been

examined, their combined effects on the roll behaviour of the 5-axle dirt truck were evaluated. For the evaluation, certain parameters of the baseline vehicle were modified. The modified values of these parameters are: tractor frame torsional stiffness of 9,000,000 in-lb/deg (essentially a rigid frame), zero Coulomb friction, and equivalent torsional stiffness of trailer structure and fifth wheel of 9,000,000 in-lb/deg (essentially a rigid connection), no lash in the fifth wheel plates and zero overturning stiffness of the tires. The rollover threshold and the corresponding roll angles of the sprung masses of the modified vehicle were predicted using the static roll model and compared with that of the baseline vehicle with measured values shown in Tables 1, 2 and 3. The results are given in Table 5.4.

Table 5.4  
A Comparison of the Roll Behaviour of  
The Modified and The Baseline Vehicle

Vehicle Type	Rollover Threshold, $a_y \text{ max, g}$	Roll angle of first sprung mass, $\phi_1$ , deg	Roll angle of second sprung mass, $\phi_2$ , deg	Roll angle of third sprung mass, $\phi_3$ , deg
Baseline Vehicle	0.35	6.01	5.40	5.78
Modified Vehicle	0.35	5.76	5.72	5.78

From Table 5.4, it can be seen that the changing of the tractor frame torsional stiffness, Coulomb friction present in the tractor frame, and trailer structural and fifth wheel compliance, lash in the fifth wheel plates and overturning stiffness of tires in the ranges shown, has no effect on the rollover threshold and the corresponding roll angle of the third sprung mass. Their combined effects on the roll angles of the first and second sprung masses are slight. For instance, the difference in the simulated roll angle of the first sprung mass between the modified and the baseline vehicle is only 4.2 %, and the difference in the simulated roll angle of the second sprung mass between the two vehicles is approximately 6 %.

## 6. DISCUSSIONS

Based on the results of this study, the following observations are made:

A. In comparison with the measured data available, the steady-state steering responses of tractor-semitrailers with different design features predicted using the Phase 4 model, the yaw/roll model, the TBS model, and the linear yaw plane model all have varying degrees of error (see Figs. 3.2, 3.3, 3.5 and 3.6.). It appears that a more sophisticated simulation model, such as the Phase 4 model, does not necessarily give a more accurate prediction than a simpler model, such as the TBS model or the linear yaw plane model, under certain circumstances. For instance, for Vehicle Configuration 2, of which the tractor is highly understeer, the simple model (i.e., the linear yaw plane model) appears to give the best overall prediction in the lateral acceleration range up to approximately 0.4 g (see Fig. 3.5).

For the six vehicle configurations examined in this study, it appears that there are no significant differences in the steady-state steering responses predicted using the four simulation models in the lateral acceleration range up to approximately 0.25 g.

B. There are significant differences in handling characteristics predicted using the four simulation models in most cases. Since the linear yaw plane model does not take into account the effects of load transfer and uses a linear tire model, it is not capable of predicting changes in handling behaviour with lateral acceleration. On the other hand, the Phase 4 model, the yaw/roll model, and the TBS model take into

account the effects of load transfer and the non-linear behaviour of tires to varying degrees. Consequently, these three models can predict changes in handling behaviour with lateral acceleration (see Figs. 3.4 and 3.7). It should be noted, however, that the predictions made by these three models are still noticeably different from the measured data available.

It should be pointed out that the parameter  $(rl/U - \delta_{av})$  used to characterize vehicle handling behaviour is very sensitive to the errors in the values of yaw rate  $r$ , forward speed  $U$ , and the front wheel steering angle  $\delta_{av}$ . A small error in the values of  $r$ ,  $U$ , and  $\delta_{av}$  will result in a significant error in the value of the parameter  $(rl/U - \delta_{av})$ . For example, if the wheelbase  $l$  of a vehicle is 142 in., and the nominal values of  $r$ ,  $U$  and  $\delta_{av}$  are 10 deg/s, 63.07 ft/s, and 1.7 deg., respectively, then a  $\pm 5\%$  error in these values will result in an error in the value of  $(rl/U - \delta_{av})$  ranging from  $-150\%$  to  $+160\%$ . Even a  $\pm 1\%$  error in these values will result in an error of  $\pm 31\%$  in the value of  $(rl/U - \delta_{av})$ . This would cause problems in using this parameter to characterize the handling behaviour in practice.

C. The transient steering responses of tractor-semitrailers predicted using the four simulation models are qualitatively similar in many cases. However, in comparison with the measured data available, all the predictions have varying degrees of error, as can be seen from Figs. 4.2 to 4.6. It appears that a more sophisticated model, such as the Phase 4 model, is not necessarily better than a simpler model, such as the TBS model or the linear yaw plane model, in predicting the transient responses of tractor-semitrailers like Vehicle Configuration 1.

For tractor semitrailers in severe lane-change manoeuvres, the transient steering responses, particularly the yaw rate responses, predicted using various models are quite close, as can be seen from Figs. 4.8 and 4.9.

For doubles, such as Vehicle Configuration 6, in severe lane-change manoeuvres, the yaw rate responses of the tractor and semitrailers predicted

using the various models are similar (see Figs. 4.12 and 4.13). However, there is a noticeable difference between the yaw rate responses of the full trailer predicted using the linear yaw plane model and those predicted using the Phase 4 model and the yaw/roll model (see Fig. 4.14).

D. The results of a parametric sensitivity study of the static roll model indicate that some of the parameters, such as the tractor frame torsional stiffness, Coulomb friction in the tractor frame, equivalent trailer structural and fifth wheel compliance, separation of fifth wheel plates, and the overturning stiffness of tires, have little, if any, effects on the rollover threshold of the articulated vehicle examined in this study. This may indicate that the static roll model could be simplified, at least for certain applications. This would greatly reduce the number of input data required and hence the effort needed to obtain them.

E. To aid engineering practitioners in industry to utilize the various simulation models in a cost-effective manner, defining the appropriate areas of application of different models is of significance. This will provide guiding principles for the practitioner to select the appropriate model for a particular application. To achieve this objective, it appears that further work on the experimental evaluation of various models will be required.

F. Based on the results of this study, it appears that a sophisticated simulation model does not necessarily guarantee a more accurate prediction than a simpler one. On the other hand, a sophisticated model generally requires a large number of input data. Many of them can only be obtained using special measuring equipment. Quite often, even if the equipment is available, the cost of obtaining the required data is high. It appears that a comprehensive parametric sensitivity study of these sophisticated models may be useful. The objective is to evaluate the relative significance of various parameters to vehicle dynamic behaviour. If some of the parameters

are judged to have an insignificant effect, then they may be excluded. This will make it easier and less costly for the user of the models to obtain input data.

G. Some of the simulation models evaluated in this study have primarily been developed by researchers and for researchers. They are not structured in a "user-friendly" manner. The procedure for inputting data to these models is not particularly convenient to an ordinary user and considerable effort may be required to interpret the outputs. To make the simulation techniques more useful to engineering practitioners, further work on streamlining the structure of the models, particularly the input and output, is recommended.

H. Based on the results of this study and on the current state of the simulation techniques for the lateral dynamics of articulated vehicles, a more realistic goal, for the foreseeable future, for the development of computer simulation models is to provide a common basis for evaluating the dynamic performance of different vehicles on a relative basis, rather than to accurately reproduce their actual dynamic behaviour in minute detail.

## REFERENCES

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6. J. Y. Wong and J. Woodroffe, Course Notes for the "Introductory Workshop on Computer Simulation of the Handling and Braking of Heavy Vehicle Combinations", Roads and Transportation Association of Canada, July 1985.

7. P. S. Fancher, Jr., C. Mallikarjunarao, and R. L. Nisonger, Simulation of the directional response characteristics of tractor-semitrailer vehicles, Report No. UM-HSR1-79-9, PB80-189632, U. S. Department of Commerce, National Technical Information Service, March 1979.

**APPENDIX A**

**Vehicle Configuration 1**  
**Input Data Files**

```

VAX/VMS RTAC P4CONF1 21-APR-1985 11 34 DISK4VAXUSER1 [USER1..RTACJP4CONF1] DAT 1 VAX/VMS
VAX/VMS RTAC P4CONF1 21-APR-1985 11 34 DISK4VAXUSER1 [USER1..RTACJP4CONF1] DAT 1 VAX/VMS
VAX/VMS RTAC P4CONF1 21-APR-1985 11 34 DISK4VAXUSER1 [USER1..RTACJP4CONF1] DAT 1 VAX/VMS

```

	RARRR	TTTTT	AAA	CCCC
H	R	T	A	C
R	R	T	A	C
R	R	T	A	C
R	R	T	AAGAA	G
R	R	T	A	C
R	R	T	A	C
R	R	T	A	C

VAX/VMS RTAC

```
D15K$VAXUSER1 :USER1 RTAC JP4CONF1 DAT 1 VAX/VMS  
D15K$VAXUSER1 :USER1 RTAC JP4CONF1 DAT 1 VAX/VMS  
D15K$VAXUSER1 :USER1 RTAC JP4CONF1 DAT 1 VAX/VMS
```

```
DISK$VAXUSER1  [USER1..RTACJP4CONF1..DATI..1  
DISK$VAXUSER1  [USER1..RTACJP4CONF1..DATI..1  
DISK$VAXUSER1  [USER1..RTACJP4CONF1..DATI..1  
DISK$VAXUSER1  [USER1..RTACJP4CONF1..DATI..1
```

3-AXLE TRACTOR-SEMI TRAILER (CONF. 1)		10 , 1 , 032
240	01	6700
300	63.07	6700
400	016	6000
500	0 , 0	6700
600	0 , 25,-35,-55	7000
600	0 , 5,-2,-2,-77	7100
602	0 , 75,-2,-2,-5	7300
604	1 , 0,-1,94,-1,39	7350
605	1 , 25,0,0,0,2	7400
609	1 , 5,1,7,2,0	7500
610	1 , 75,2,0,3,33	7600
612	2 , 0,2,5,2,77	7700
614	2 , 25,1,4,0,64	7800
616	2 , 5,-0,55,-0,77	8000
618	2 , 75,-0,83,-1,37	8050
620	3 , 0,-0,83,-0,83	8100
622	3 , 25,-0,5,0,0	8200
624	3 , 5,0,0,-0,4	8400
626	3 , 75,0,0,0,0	8700
628	0 , 0	9700
900	0 , 0	9000
1000	10 , 0.	13800
1200	4 , 0	13900
1300	0 , 1	14100
1400	0,0	14200
1500	111,00000	14300
1600	142,	14400
1700	BBP7, 93	14500
1700	7110, 07	14600
1700	39, 7	14700
2000	1B1164, 6	14800
2100	37753,	14900
2200	69955,	15000
2300	0 , 0	17300
2400	0 , 0	17400
2500	48, 5	17500
2600	20000,	17600
2700	36	17700
2800	1012, 5	17800
4000	10	17900
4100	300,	18000
4200	3719,	18100
4300	24, 35	18200
4400	0 , 00	18300
4500	0 , 0	18400
4600	32, 6	18500
4700	80, 5	18600
4800	2000, , 6000, , 7000,	18700
5100	01	18800
5300	61,	18900
6000	06,	19000
6100	0 , 0	19100
6140	1 , 1, 210	19200
6360	3 , 4, 5355	19300
6400	4 , 4, 6535	19400
6440	7 , 1, 6977	19500

CAB		10 , 1 , 032
27912	0	20000
27914	3 , 1 , 460	20100
27916	4 , 1 , 586	20200
27920	7 , 1 , 83	20300
27922	10 , 1 , 966	20400
27924	06	20500
27926	0 , 0 , 0	20600
27928	1 , 1 , 114	20700
27930	3 , 1 , 310	20800
27932	4 , 1 , 394	20900
27936	7 , 1 , 595	21000
27938	10 , 1 , 735	21100
27940	06	21200
27942	0 , 0 , 0	21300
27944	1 , 1 , 082 ,	21400
27946	3 , 1 , 229	21500
27948	4 , 1 , 294	21900
27952	7 , 1 , 459	22000
27954	10 , 1 , 584	22100
27956	01	22200
27958	0	22300
27970	0	22500
27974	01	22600
27976	0	22700
27978	0	22800
27980	0	22900
27982	0	23000
27984	0	23100
27986	0	23200
27988	0	23300
27990	0	23400
27992	0	23500
27994	0	23600
27996	0	23700
27997	1	23800
27998	28000	23900
28000	0	24000
28002	0	24100
28004	0	24200
28006	0	24300
28008	0	24400
28010	1200,	24500
28200	5200,	24600
28300	19, 5	24700
28400	115,	24800
28500	13,	249, 5
28600	-2,	250,
28700	20000,	25100
28800	0	25200
28900	1200,	25300
29000	3300,	25400
29100	19, 5	25500
29200	115,	25600
29300	0, 175	25700
29400	0, 25	25800
29500	0	25900
29600	0, 175	26000
29700	0, 25	26100
29800	0, 0	26200
29900	-1	26300
30000	0	26400
30000	38	26500
30000	72	26600
30000	1520,	26700
30000	9587, 5	26800
30000	0, 1	26900
30000	0	27000
30000	0	27100
30000	13	27200
30000	0, 1	27300
30000	0	27400
30000	38,	27500
30000	72	27600
30000	1500,	27700
30000	0, 1	27800
30000	0, 0	27900
30000	0	28000
30000	0	28100
30000	0	28200
30000	0	28300
30000	0	28400
30000	0	28500
30000	0	28600
30000	0	28700
30000	0	28800
30000	0	28900
30000	0	29000
30000	0	29100
30000	0	29200
30000	0	29300
30000	0	29400
30000	0	29500
30000	0	29600
30000	0	29700
30000	0	29800
30000	0	29900
30000	0	30000

VAX/VMS

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

VAX/VMS  
VAX/VMS  
VAX/VMS

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

## 5-AXLE TRACTOR-SEMITRAILERS (CDN: 1)

160	0205	63 07. 4. , 1. , 02	6400	2000 , 44 , 72 , 84 , 85 , 78 , 59
300	63 07. 4. , 1. , 02	6500	4000 , 103 , 235 , 235 , 245 , 187.	
400	312	6600	4000 , 153 , 341 , 376 , 431 , 435 , 333.	
500	16316 , 34001	6700	6000 , 205 , 472 , 350 , 622 , 400 , 470.	
600	18165 , 4111054. 06	6800	7000 , 227 , 537 , 637 , 716 , 673 , 550.	
700	47853 , 2517342. 06	6900	6097	
000	59755 , 2517038	7000	0 , 1 , 3 , 4 , 9 , 7 , 10,	
700	39 7. 65 671	7100	2000 , 31 , 44 , 46 , 56 , 61 , 39,	
1000	8892. 929 15781. 397 15611. 397 16873 137. 17823 137	7200	4000 , 60 , 143 , 157 , 189 , 194 , 138	
1100	1190. 2340. , 2770. 1520.	7300	6000 , 130 , 261 , 297 , 324 , 373 , 327.	
1200	3719. , 4458. , 4458. , 4100. , 4100	7400	8000 , 179 , 367 , 439 , 532 , 562 , 473.	
1300	35. 9. -81. 1. -131. 1. -156. 6. -206. 1	7500	9000 , 200 , 452 , 533 , 624 , 630 , 562.	
1400	20. 3. 26. 3. 20. 3. 17. 5. 19. 5	7600	1	
1500	24. 55. 32. , 22. , 75. , 25. 6	7700	2	
1600	16. 3. 17. 5. 17. 3. 19. , 19	7800	0 , 0 , 0	
1700	40. 25. 36. , 36. , 36. , 36	7900	5. , -1. 5. , -1. 5	
1800	0. 0. 13. , 13. , 13. , 13. ,			
1700	5700. , 5700. , 5700. , 5300. , 5300.			
2000	0 , 1. , 1. , 1. , 1			
2100	0 , 0 , 78000 , 0 , 0			
2200	500. , 250. , 250. , 950. , 950			
2300	10 , 10 , 10 , 10 , 10			
2500	-106. 1. 229			
2600	40. 5			
2700	777499999.			
2800	01			
2900	03			
3000	0102020303			
3100	03			
3200	-10125. , -10			
3300	0 , 0 , 0 , 0,			
3400	10125. , 10.			
3500	03			
3600	-15000. , -10			
3700	0 , 0 , 0 , 0,			
3800	15000. , 10.			
3700	03			
4000	-75875. , -10.			
4100	0 , 0 , 0 , 0,			
4200	75875. , 10.			
4400	02			
4500	0101010202			
4600	0603			
4700	0 , 0 , 1 , 0 , 3 , 4 , 5 , 7 , 10.			
4800	2000. , 436. , 1063. , 1311. , 1506. , 1794. , 2044			
4900	4000. , 633. , 1657. , 2072. , 2430. , 2957. , 3042.			
5000	6000. , 764. , 2021. , 2539. , 3019. , 3704. , 4022.			
5100	8000. , 831. , 2225. , 2827. , 3384. , 4190. , 5004.			
5200	7000. , 830. , 2284. , 2709. , 3487. , 4341. , 5319.			
5300	0607			
5400	0 , 0 , 1 , 0 , 3 , 4 , 5 , 7 , 10.			
5500	2000. , 365. , 936. , 1371. , 1660. , 1935.			
5600	1000. , 556. , 1503. , 2283. , 2814. , 3400.			
5700	6000. , 616. , 1850. , 2363. , 2821. , 3570. , 4409.			
5800	8000. , 737. , 2025. , 25976. , 3135. , 4012. , 5030.			
5900	9000. , 742. , 27050. , 3446. , 3213. , 4131. , 5255.			
6000	0.			
6100	0101010207			
6200	0607			
6300	0 , 0 , 1 , 0 , 3 , 4 , 5 , 7 , 10			

5/5

WAK / WMS RTAC  
WAN / WMS RTAC  
WAN / WMS RTAC

WAX LUMS

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DISK5\XUSER1\T1USER1-RTAC2TBSCNF1-DAT1
DISK4\XUSER1\T1USER1-RTAC2TBSCNF1-DAT1
DISK3\XUSER1\T1USER1-RTAC2TBSCNF1-DAT1
DISK2\XUSER1\T1USER1-RTAC2TBSCNF1-DAT1
DISK1\XUSER1\T1USER1-RTAC2TBSCNF1-DAT1

```

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VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1  
LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1  
LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1

RRRR

TTTT

AAAA

CCCC

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1  
LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1  
LYPCONF1 21-APR-1985 12:43 LPBO: 21-APR-1985 12:48 DISK\$VAXUSER1:1USER1.RTACJLYPCONF1.DAT1

VAX/VMS

VAX/VMS

VAX/VMS

	05-AXLE TERRACITOR SEMI TRAILER (CUMF 132)
200	16016 0 57951 0
210	192915 0 2775432 0
300	63 11 53 89 103 89 147 0B 196 3D 0 0 0 0 0 0 0 0 0
400	63 11 53 89 103 89 147 0B 196 3D 0 0 0 0 0 0 0 0 0
500	7B B9 23D 17
600	1460 0 2760 0 3760 0 2460 0 2460 0 0 0
650	0 0 0 0 0 0
700	63 07 0 1 10 0 1 0
800	220 400 400 400 400 400 0 0
900	0 0 0 0 0 0
950	20000 20000 20000 20000 0 0 0 0
1000	0 0 0 0 0 0
1000	13 0 13 0 13 0 13 0 0 0 0 0 0 0 0 0 0 0 0 0
1100	6 0 0 0 1 1
1200	0 0
1300	0 0 0 0
1400	0 5 -2 0
1500	6 0 -2 0

**APPENDIX B**

**Vehicle Configuration 2**

**Input Data Files**

VAX/VMS

		AAA	CCCC
RRRR	TTTT	A A	C C
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	TTTT	A A	C C
R R	T T	AAAAA	C
R R	T T	A A	C C
R R	T T	A A	C C

TRAC

	PACONF 2	PACONF 2	PACONF 2	PACONF 2	PACONF 2	PACONF 2
21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983
21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983
21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983
21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983	11:35	21-APR-1983

DISK\$VAXUSER1:[USER1].RTACJP4CONF2.DAT1  
DISK\$VAXUSER1:[USER1].RTACJP4CONF2.DAT1  
DISK\$VAXUSER1:[USER1].RTACJP4CONF2.DAT1

DISK4VAXUSER1:[USER1] RTACJP4CONF2 DAT11  
DISK4VAXUSER1:[USER1] RTACJP4CONF2 DAT1  
DISK4VAXUSER1:[USER1] RTACJP4CONF2 DAT1

## 5-AXLE TRACTOR-SEMITRAILER (CONF 2)

100	01	21500	0	27948	4	294
200	01	21900	410	27952	7	459
300	05	22000	6252.29	27954	10	.384
400	003	22100	11061.71	27956	01	
500	010	22200	69	27957	0	
600	010	22300	71000	27958	01	
700	010	22400	70916.9	27959	0	
800	010	22500	78906.4	27960	28000.	
900	010	22600	4040.0	27961	0	
1000	10.0	22700	162.	27962	00	
1200	6.0	22800	64.5	27963	00	
1300	01	22900	37500	27964	00	
1400	00	23000	1727000	27965	19.3	
1500	11.00000	23100	1727000	27966	1.14.	
1600	14.9	23200	01	27967	00	
1700	3097.93	23300	49.3	27968	-2	
1700	7110.07	23400	50.	27969	28000.	
1800	39.7	23500	0.0	27970	0	
1900	18154.6	23600	9587.5	27971	0	
2000	120000.	23700	250.	27972	0	
2100	69933	23800	4450	27973	0	
2200	69933	23900	22.	27974	0	
2300	00	24000	50.	27975	0	
2400	00	24100	0.1	27976	0	
2500	40.3	24200	0.0	27977	0	
2600	120000.	24300	35.	27978	0	
2700	36.	24400	72.	27979	0	
2800	1012.5	24500	2340.	27980	0	
2900	10.	24600	1500.	27981	0	
3000	500	24700	10.	27982	0	
3100	371.9	24800	250.	27983	0	
3200	24.53	24900	22.	27984	0	
3300	0.00	25000	0.1	27985	0	
3400	109385.	25100	78000.	27986	0	
3500	-1	25200	35.	27987	0	
3600	32.6	25300	0.1	27988	0	
3700	2000.4000.9000	25400	27200.	27989	12	
3800	01	25500	1200.	27990	03	
3900	43	25600	5700.	27992	0000.9000.	
4000	06	25700	20.3	27993	0	
4100	010	25800	115	27994	0	
4200	1.1.218	25900	13.	27995	0	
4300	3.1.5355	26000	115	27996	0	
4400	4.1.4553	26100	19800	27997	0	
4500	7.1.897.	26200	19900	27998	1.1.183	
4600	10.1.1.022	26300	28000	27999	3.1.469	
4700	01	26400	0.	279914	3.1.469	
4800	1190	26500	0.	279916	4.1.596	
4900	0	26600	1200.	279920	7.1.83	
5000	-1	26700	5700.	279922	10.1.968	
5100	03	26800	20.3	279924	06	
5200	19100	26900	0.0	279926	0.0.	
5300	26000	27000	115	279928	10.1.735	
5400	19200	27100	0.05	279930	06	
5500	06	27200	0.25	279932	1.1.114	
5600	0.0.	27300	0.	279934	3.1.310	
5700	20400	27400	0.	279936	4.1.394	
5800	01	27500	0.0	279938	10.1.229	
5900	01	27600	0.05	279940	06	
6000	06	27700	0.25	279942	0.0	
6100	010	27800	0.	279944	1.1.002	
6200	1.1.218	27900	0.	279946	3.1.229	
6300	3.1.5355	28000	115			
6400	4.1.4553	28100	19900			
6500	7.1.897.	28200	28000			
6600	10.1.1.022	28300	0.			
6700	06	28400	1200.			
6800	0.0.	28500	5700.			
6900	1.1.127	28600	20.3			
7000	3.1.337	28700	0.0			
7100	4.1.423	28800	0.25			
7200	7.1.617	28900	0.			
7300	10.1.737	29000	0.			
7400	06	29100	0.			
7500	0.0.	29200	0.05			
7600	1.1.092	29300	0.25			
7700	3.1.254	29400	0.			
7800	4.1.321	29500	0.			
7900	10.1.402	29600	0.05			
8000	7.1.402	29700	0.			

WAX / VMS  
VAX / VMS  
VAX / VMS

VAX/VMS	VAX/VMS	VAX/VMS
YRCNCONF2 21-APR-1983 11:41 LPBO: 21-APR-1983 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]	YRCNCONF2 21-APR-1983 11:41 LPBO: 21-APR-1983 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]	YRCNCONF2 21-APR-1983 11:41 LPBO: 21-APR-1983 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]
YRCNCONF2 21-APR-1985 11:41 LPBO: 21-APR-1985 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]	YRCNCONF2 21-APR-1985 11:41 LPBO: 21-APR-1985 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]	YRCNCONF2 21-APR-1985 11:41 LPBO: 21-APR-1985 11:41 DISK&VAXUSER!:[USER1 RTACJYRCNCONF2. DAT]

YRC0NF2 21-APR-1983 11:41 LPB0: 21-APR-1983 11:41 DISK&VAXUSER:[USER1] RTAC)YRC0NF2.DAT:1  
YRC0NF2 21-APR-1983 11:41 LPB0: 21-APR-1983 11:41 DISK&VAXUSER:[USER1] RTAC)YRC0NF2.DAT:1  
YRC0NF2 21-APR-1983 11:41 LPB0: 21-APR-1983 11:41 DISK&VAXUSER:[USER1] RTAC)YRC0NF2.DAT:1

NAME	TYPE	AAA	CCCC
R R	T T T T	A A	C
R R	T T	A A	C
R R	T	A A	C
R R	T	A A	C
R R	T T T T	A A A A	C C C C
R R	T T T	A A A	C C C
R R	T T	A A	C C
R R	T	A A	C C

MATE  
RATAC

```

YRCONF2 21-APR-1983 11:41 LPO0: 21-APR-1983 11:41 DISK4VAXUSER1:[USER].RTACJYRCONE2.DAT1

```



	VAX/VMS	VAX/VMS	VAX/VMS	VAX/VMS
TAC	RTAC	RTAC	RTAC	RTAC
TBSCONF2	21-APR-1985	15.22	LP00:	21-APR-1985
TDSCONF2	21-APR-1985	15.22	LP00:	21-APR-1985
TDSCONF2	21-APR-1985	15.22	LP00:	21-APR-1985
TDSCONF2	21-APR-1985	15.22	LP00:	21-APR-1985

	CCCC	AAAA	TTTT	RRRR	GGGG
R	A	A	T	R	H
H	C	C	I	H	H
H	C	C	I	H	H
RRRR	C	C	T	RRRR	RRRR
RRR	C	C	T	RRR	RRR
RR	C	C	T	RR	RR
R	C	C	T	R	R

RDRR	TTTT	AAA	CCCC
R R	T T	A A	C
R R	T T	A A	C
R R	T T	A A	C
R R	T T	AAAAA	C
R R	T T	A A	C
R R	T T	A A	CCCC

VAX/VMS	RTAC	TDSCONF2	21-APR-1983	15:22	LPDO	21-APR-1983	15:22	DISK\VAUXUSER1\TUSER1.RTACJTBSCONF2.DAT1
VAX/VMS	RTAC	TDSCONF2	21-APR-1983	15:22	LPBO	21-APR-1983	15:22	DISK\VAUXUSER1\TUSER1.RTACJTBSCONF2.DAT1
VAX/VMS	RTAC	TDSCONF2	21-APR-1983	15:22	LPBO	21-APR-1983	19:22	DISK\VAUXUSER1\TUSER1.RTACJTBSCONF2.DAT1



VAX/VMS

LYPCONF2	21-APR-1985	12:43	LPDO.	21-APR-1985	12:46	DISK4WAVUSER1.	LUSER1.	RTAC1LYPCONF2.DAT,1
LYPCONF2	21-APR-1985	12:43	LPDO.	21-APR-1985	12:46	DISK4WAVUSER1.	LUSER1.	RTAC1LYPCONF2.DAT,1
LYPCONF2	21-APR-1985	12:43	LPDO.	21-APR-1985	12:46	DISK4WAVUSER1.	LUSER1.	RTAC1LYPCONF2.DAT,1

RATAC

RRRR	TTTT	AAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	T T	A A	C C
R R	T T	AAA	CCC
R R	T T	A A	C C
R R	T T	A A	CCCC

RRRR	TTTT	AAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
R R	T T	A A	C C
R R R R	T T T T	A A A A	C C C C
R R	T T	A A	C C
R R	T T	A A	C C
R R	T T	A A	C C

VAX/VMS

```
D1 DISK$ VAXUSER1:[USER1 RTAC JL YPCONF2.DAT] 1  
D1 DISK$ VAXUSER1:[USER1 RTAC JL YPCONF2.DAT] 1  
D1 DISK$ VAXUSER1:[USER1 RTAC JL YPCONF2.DAT] 1
```

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RRRR	TTTT	AAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	T T	A A	C C
R R	T T	AAA	CCC
R R	T T	A A	C C
R R	T T	A A	CCCC

```
DISH#VAXUSER1:[USER1] RTACJLYPCONF2.DAT1:1  
DISH#VAXUSER1:[USER1] RTACJLYPCONF2.DAT1:1  
DISH#VAXUSER1:[USER1] RTACJLYPCONF2.DAT1:1
```

LYPCONF2.DAT;1

RRRR	TTTT	AAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	T T	A A	C C
R R	T T	AAA	CCC
R R	T T	A A	C C
R R	T T	A A	CCCC

NN	FFFFFF	222222
NN	FFFFF	222222
NN	FF	22
NN	TF	22
NN	FF	22
NN	TF	22
NN	NN	22
NN	FFF	22
NN	FFFF	22
NN	FFFFF	22
NN	NNNN	22
NN	FF	22

```
DISK\WAVUSER1\USER1.R1AC1LYPCONF2.DAT1  
DISK\WAVUSER1\USER1.R1AC1LYPCONF2.DAT1  
DISK\WAVUSER1\USER1.R1AC1LYPCONF2.DAT1  
PR-1963 12.40  
PR-1963 12.40  
PR-1963 12.46  
PR-1965 12.46
```



APPENDIX C  
Vehicle Configuration 3  
Input Data Files

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

```

P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1
P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1
P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1

```

RHRHRH	TITTTT	AAA	CCCC
R R R	T T T	A A A	C C C
DPRRPR	T T T	A A A	C C C
R R R	T T T	A A A	C C C
R R R	T T T	A A A	C C C

PRINC  
H1AC

```

PACONF3 2J-APR-1985 11:35 LP00 2J-APR-1985 11:36 [USER1 RTACJP4CONF3. DAT1
PACONF3 2J-APR-1985 11:35 LP00 2J-APR-1985 11:36 DISK$VAXUSER1: [USER1 RTACJP4CONF3. DAT1
PACONF3 2J-APR-1985 11:35 LP00 2J-APR-1985 11:36 DISK$VAXUSER1: [USER1 RTACJP4CONF3. DAT1
PACONF3 2J-APR-1985 11:35 LP00 2J-APR-1985 11:36 DISK$VAXUSER1: [USER1 RTACJP4CONF3. DAT1

```

```

P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1
P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1
P4CONF 3 21-APR-1983 11:35 LPBD 21-APR-1983 11:36 DISK%VAXUSER1:[USER1.RTAGJP4CONF3.DAT]1

```





SYNTHETIC POLY(URIDYLIC ACID) ANALOGUE 31

5-AXLE TRACOR-SEMITRAILER (CONF. 3)	
100	0:05
200	0:05
300	52 5. 6 . 1 . 02
400	7700 , 61500,
500	7700 , 61500,
600	15000 , 20440,
700	25000 , 3012523,
900	75000 , 3050003,
700	45 , 81.3
1000	12000 , 17000 , 17000 , 17000 , 17000,
1100	1200 , 2300 , 2300 , 1500 , 1500,
1200	3719 , 4458 , 4458 , 4100 , 4100,
1300	20. B. -91.2. -139. 2. -190. 2. -238. 2
1400	19. 5. 19. 5. 19. 5. 19. 5. 19. 5.
1500	23. 27. 27. 27. 27. 27.
1600	16. 19. 19. 19. 19.
1700	40. 27. 5. 27. 5. 29. 5. 27. 5
1800	0. 13. 13. 13. 13.
1900	4500 , 4500 , 4500 , 4500 , 4500.
2000	0. 0. 0. 0. 0.
2100	1500 , 6000 , 6000 , 10000 , 10000,
2200	300 , 1000 , 1000 , 1000 , 1000,
2300	0. 0. 0. 0. 0.
2400	-100. 05. 217. 8
2500	48
2600	1000000.
2700	01
2800	03
2900	01020203030
3000	04
3100	-20000 , -20
3200	0. , 0.
3300	0645 , 7.2
3400	25000 , 7.5
3500	09
3600	-22500 , -41.
3700	0. , -1.
3800	0. , 0.
3900	3500 , 1.
4000	3750 , 1. 5
4100	8750 , 2.
4200	12250 , 2. 5
4300	16250 , 3.
4400	45000 , 4.
4500	09
4600	-32500 , -11.
4700	0. , -1. 5
4800	0. , 0.
4900	2531 , 0. 5
5000	6469 , 1.
5100	10749 , 1. 5
5200	16031 , 2.
5300	21635 , 2. 5
5400	50625 , 3
5500	01
5600	010101010101
5700	04000
5800	0. , 1. 2. 0. 4. 0. 4. 0. 12. 0
5900	3000 , 540 , 970 , 1710 , 2130 , 2470,
6000	5000 , 1040 , 1580 , 2070 , 2400 , 4140
6100	7000 , 1990 , 2710 , 3420 , 4680 , 6210,



A scatter plot showing the relationship between NYY and NYN. The x-axis is labeled 'NYY' and the y-axis is labeled 'NYN'. Both axes range from 0.0 to 0.1. Data points are plotted as small circles. A regression line is drawn through the points, showing a positive correlation. The points are clustered around the line, with a few outliers at higher values of NYY.

NYY	NYN
0.00	0.00
0.01	0.01
0.02	0.02
0.03	0.03
0.04	0.04
0.05	0.05
0.06	0.06
0.07	0.07
0.08	0.08
0.09	0.09
0.10	0.10
0.11	0.11
0.12	0.12
0.13	0.13
0.14	0.14
0.15	0.15
0.16	0.16
0.17	0.17
0.18	0.18
0.19	0.19
0.20	0.20
0.21	0.21
0.22	0.22
0.23	0.23
0.24	0.24
0.25	0.25
0.26	0.26
0.27	0.27
0.28	0.28
0.29	0.29
0.30	0.30
0.31	0.31
0.32	0.32
0.33	0.33
0.34	0.34
0.35	0.35
0.36	0.36
0.37	0.37
0.38	0.38
0.39	0.39
0.40	0.40
0.41	0.41
0.42	0.42
0.43	0.43
0.44	0.44
0.45	0.45
0.46	0.46
0.47	0.47
0.48	0.48
0.49	0.49
0.50	0.50
0.51	0.51
0.52	0.52
0.53	0.53
0.54	0.54
0.55	0.55
0.56	0.56
0.57	0.57
0.58	0.58
0.59	0.59
0.60	0.60
0.61	0.61
0.62	0.62
0.63	0.63
0.64	0.64
0.65	0.65
0.66	0.66
0.67	0.67
0.68	0.68
0.69	0.69
0.70	0.70
0.71	0.71
0.72	0.72
0.73	0.73
0.74	0.74
0.75	0.75
0.76	0.76
0.77	0.77
0.78	0.78
0.79	0.79
0.80	0.80
0.81	0.81
0.82	0.82
0.83	0.83
0.84	0.84
0.85	0.85
0.86	0.86
0.87	0.87
0.88	0.88
0.89	0.89
0.90	0.90
0.91	0.91
0.92	0.92
0.93	0.93
0.94	0.94
0.95	0.95
0.96	0.96
0.97	0.97
0.98	0.98
0.99	0.99
1.00	1.00

P7 AM

LPCONFIG	DATE	TIME	DISK&VAXUSER1	VAX/VMS
21-APR-1983	12:58	LPBO	RTAC1LYPCONF3, DAT1	VAX/VMS
21-APR-1983	12:58	LPBO	RTAC1LYPCONF3, DAT1	VAX/VMS
21-APR-1983	12:58	LPBO	RTAC1LYPCONF3, DAT1	VAX/VMS
21-APR-1983	12:58	LPBO	RTAC1LYPCONF3, DAT1	VAX/VMS

प्राप्ति	प्राप्ति	प्राप्ति	प्राप्ति	प्राप्ति
स	स	स	स	स
स	स	स	स	स
स	स	स	स	स
स	स	स	स	स

PRIMER	T/T/T/T	A/A/A/A	C/C/C/C
R R	T T	A A	C
R R	T T	A A	C
R/R/R	T T	A A	C
R/R/R	T T	A A	C
R R	T T	A A	C
R R	T T	A A	C

RTAC  
RTAC  
RTAC  
RTAC

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NN	FFFFFFFFFF	3303333
NN	FFFFFFFFFF	3333333
NN	FF	33
NN	NN	33
NNNN	FF	33
NNNN	FF	33
NN	FF	33
NN	FF	33
NN	NN	333333
NN	NN	333333

D1D1SH4VAXUSER1: CUSER1, RTAC2L YPCONFIG3, DATA1  
D1D1SH4VAXUSER1: IUSER1, RTAC2L YPCONFIG3, DATA1  
D1D1SH4VAXUSER1: MUSER1, RTAC2L YPCONFIG3, DATA1  
D1D1SH4VAXUSER1: PUSER1, RTAC2L YPCONFIG3, DATA1  
VAX/VI  
VAX/VI  
VAX/VI  
VAX/VI

05-AXLE TRACTOR-SEMITRAILER (CONF. 3 & 4 )

100	15000.	64500.
200	15000.	64500.
300	214217.	4202700.
400	60.76.59.24.107.24.180.20.228.28.0.0.0.0.0.0.	
500	68.89.227.723	
600	1600.2480.2480.2400.2400.0.	
700	0.0.0.0.0.0.	
800	52.5.3.10.11.05	
900	267.400.400.400.400.0	
1000	0.0.0.0.0.0.	
1100	28000.28000.28000.28000.0.0.	
1200	0.0.0.0.0.	
1300	13.13.13.13.0.0.0.0.0.0.	
1400	8.001.1	
1500	03	
1600	0.0	
1700	1.5.2.6	
1800	10.2.6	

APPENDIX D  
Vehicle Configuration 4  
Input Data Files

VAN DER VENNS  
RITA C.  
RITA C.

```

PACONF4 21-APR-1985 11:35 LPDO: 21-APR-1985 11:37 DISK4-VAXUSER1:[USER1.RTACJP4CONF4.DATI.1
PACONF4 21-APR-1985 11:35 LPDO: 21-APR-1985 11:37 DISK4-VAXUSER1:[USER1.RTACJP4CONF4.DATI.1
PACONF4 21-APR-1985 11:35 LPDO: 21-APR-1985 11:37 DISK4-VAXUSER1:[USER1.RTACJP4CONF4.DATI.1

```

RRRR	TTTT	AAAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	T T	A A	C C
R R	T T	AAAAA	C CCCC
R R	R R	T	A A
R R	R R	T	A A

R1ACG

P4CONF4 21-APR-1985 11:35 LPBO: 21-APR-1985 11:37  
P4CONF4 21-APR-1985 11:35 LPBO: 21-APR-1985 11:37  
P4CONF4 21-APR-1985 11:35 LPBO: 21-APR-1985 11:37

DISKSWAXUSER12

## 3-AXLE TRACTOR-SEMITRAILER (CONF. 4)

100	01	6700	12., ., 833	05	18900	2300.
200	34, 5	6800	06	12700	19000	13.
300	003	6900	0., 0.	13000	19100	-1.
400	0., 0.	7000	1., ., 143	13100	19200	-2.
500	0., 0.	7100	2., ., 252	13200	19300	0.
600	0, 5, 2, 3, 2, 5	7200	4., ., 438	13300	19400	45000, ., 3.
700	10, ., 2, 5, 2, 5	7300	6., ., 501	13400	19500	0, 02
1000	04	7400	12., ., 693	13500	19600	19, 1
900	0, 0	7500	06	13600	19700	113.
1000	2, 5, 0,	7600	0., 0.	13700	19800	13.
1100	2, 6, 0,	7700	1., ., 107	13800	19900	-1.
1200	10, ., 0,	7800	2., ., 194	13900	20000	-2.
1300	0	7900	4., ., 300	14000	20100	0.
1400	0, 1	8000	6., ., 310	14100	20200	1200.
1500	00	8100	12., ., 603	14200	20300	4100.
1600	111000000	8200	05	14300	20400	19, 3
1700	144,	8300	0	14400	20500	113.
1000	8760,	8400	04	14500	20600	0.
1900	6540,	8500	004	14600	20700	0, 05
2000	44,	8500	1	14700	20800	0, 23
2100	15000,	8600	15	14800	20900	1000.
2200	75000,	8700	1.	14900	21000	0.
2300	75000,	8800	03	15000	21100	0.
2400	0, 0	8900	0.	15100	21200	0.
2500	14, 35	9000	4,	15200	21300	1500.
2600	10, 0	9100	0,	15300	21400	0, 073
2700	50000,	9200	12,	15400	21500	0, 25
2800	36,	9300	16,	15500	21600	0, 073
2900	-119,	9400	1., 1., ., 9., ., 3., 1	15600	21700	1500.
3000	04	9500	1., 1., ., 9., ., 3., 1	15700	21800	432.
3100	-20000, ., -20,	9600	1., 1., ., 9., ., 33, 13	15800	21900	4500.
3200	0, 0,	9700	1., 1., ., 9., ., 42, 17	15900	22000	60, 0.
3300	9250, ., 7, 2	9800	1., 1., ., 9., ., 40, 22	16000	22100	20100, 19, 5
3400	25900, ., 7, 5	9900	-2,	16100	22200	20200, 113.
3500	0, 03	10000	03	16200	22300	13.
3600	-20000, ., -20,	10100	30000, ., 6000, ., 7000,	16300	22400	28400, -1,
3700	0, , 0	10200	01	16400	22500	28500, -2,
3800	0040, ., 7, 2	10300	66,	16500	22600	28600, 0.
3900	25900, ., 7, 5	10400	03	16600	22700	28700, 1200.
4000	0, 00	10500	0., 0.	16700	22800	28800, 4500.
4100	0,	10600	11, ., 43	16800	22900	3050000.
4200	0,	10700	2., ., 30	16900	23000	3100000.
4300	3719,	10800	3, ., 77	17000	23100	01
4400	23,	10900	1, 0., ., 55	17100	23200	50.
4500	0, 00	11000	09	17200	23300	0.
4600	0, 0	11100	0., 0.	17300	23400	0, 175
4700	32,	11200	1., ., 39	17400	23500	-122.
4800	80, 0	11300	12., ., 75	17500	23600	09
4900	1260,	11400	13., ., 73	17600	23700	-30000, ., -11.
5000	5500	11500	1., ., 50	17700	23800	0., -1, 5
5600	-1	11600	05	17800	23900	0, 0.
5700	03	11700	0., 0.	17900	24000	3373, ., 5
5900	3000, ., 6000, ., 7000,	11800	11, ., 44	18000	24100	11032, ., 1, 3
5700	01	11900	12., ., 70	18100	24200	16073, ., 2
6000	66,	12000	13., ., 67	18200	24300	22500, ., 3, 5
6100	06,	12100	1., ., 43	18300	24400	34230, ., 3
6300	0, 0,	12200	03	18400	24500	0, 02
6400	1, 1, ., 103	12300	0	18500	24600	-35000, ., -11.
6500	2, 1, ., 333	12400	04	18600	24700	0, ., -1, 3
6600	4, ., 575	12500	1	18700	24800	0, ., 0, 2
6600	6, ., ., 708	12600	5	18800	24900	14117, ., 1
		12700	1,	18900	25000	

VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC  
 VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC  
 VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC

YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCUNFA.DAT1  
 YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCDNFA.DAT1  
 YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCNF4.DAT1

RRRR	TTTT	AAA	CCCC
R R	T	A A	C
R R	T	A A	C
R R	T	AAA	C
R R	T	A A	C
R R	T	A A	CCCC

YY	YY	ARRARRAR	CCCCCCCC	000000	NN	NN	FFFFFFFFFF	44	44
YY	YY	RRRRRRRR	CCCCCC	000000	NN	NN	FFFFFFFFFF	44	44
YY	YY	RR	CC	00	NN	NN	FF	44	44
YY	YY	RR	CC	00	NN	NN	FF	44	44
YY	YY	RR	CC	00	NNNN	NN	FF	44	44
YY	YY	RR	CC	00	NNNN	NN	FF	44	44
YY	YY	RRRRRRRA	CC	00	NN	NN	FFFFFFFFFF	4444444444	444444444444
YY	YY	RRRRRRRA	CC	00	NN	NN	FFFFFFFFFF	444444444444	444444444444
YY	YY	RR	CC	00	NN	NN	FF	44	44
YY	YY	RR	CC	00	NN	NN	FF	44	44
YY	YY	RR	CC	00	NN	NN	FF	44	44
YY	YY	RR	CC	000000	NN	NN	FF	44	44
YY	YY	RR	CC	000000	NN	NN	FF	44	44

DDDDDDDD	DDDDDDDD	AAAAAA	TTTTTTTT	1111	11	11	1111	1111	1111
DDDDDDDD	DDDDDDDD	AAA	TTTTTTTT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DD	DD	AA	TT	1111	11	11	1111	1111	1111
DDDDDDDD	DDDDDDDD	AA	TT	1111	1111	1111	1111	1111	1111
DDDDDDDD	DDDDDDDD	AA	TT	1111	1111	1111	1111	1111	1111

RRRR	TTTT	AAA	CCCC
R R	T	A A	C
R R	T	A A	C
RRRR	T	A A	C
R R	T	AAA	C
R R	T	A A	C
R R	T	A A	CCCC

VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC  
 VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC  
 VAX/VMS RTAC VAX/VMS RTAC VAX/VMS RTAC

YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCUNFA.DAT1  
 YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCDNFA.DAT1  
 YACONF4 21-APR-1985 11:57 LPBO: 21-APR-1985 11:57 DISK\$VAXUSER1: [USER1.RTAC]YRCNF4.DAT1

VAX/VMS  
VAX/VMS  
VAX/VMS

## 5-AXLE TRACER-SEMITRAILER (CONF 4)

100	52.3, 6.	1..02	
200	0205		
300	52.3, 6.	1..02	
400	0302		
500	7700., 61500.		
600	15000., 204440		
700	75000., 3812523		
800	75000., 3850003		
900	44., 81.3		
1000	12000., 17000.	17000., 17000.	
1100	1200., 2300.	1500., 1500.	
1200	3719., 4458.	4458., 4100.	
1300	28.8., -91.2., -137.2., -190.2., -238.2		
1400	19.5., 19.5., 19.5., 19.5., 19.5.		
1500	23., 29., 29., 29., 29.		
1600	16., 19., 19., 19., 19.		
1700	40., 29., 5., 29., 5., 27., 5., 27., 5.		
1800	0., 13., 13., 13., 13.		
1900	4500., 4500., 4500., 4500., 4500.		
2000	0., 0., 0., 0., 0.		
2100	0., 0., 78000., 0., 0.		
2200	300., 1000., 1000., 1000., 1000.		
2300	0., 0., 0., 0., 0.		
2400	-100., 05., 217.8		
2500	4B.		
2600	1000000.		
2700	01		
2800	03		
2900	0102020303		
3000	04		
3100	-20000., -70		
3200	0., 0.		
3300	0.645., 7.2		
3400	25000., 7.5		
3500	01,		
3600	-22500., -11.		
3700	0., -1.		
3800	0., 0.		
3900	3500., 1.		
4000	5750., 1.5		
4100	0750., 2.		
4200	12250., 2.5		
4300	16250., 3		
4400	45000., 4.		
4500	09		
4600	-32500., -11		
4700	0., -1.5		
4800	0., 0.		
4900	2531., 0.5		
5000	6.461., 1		
5100	10967., 1.5		
5200	16031., 2		
5300	21656., 2.5		
5400	50625., 3		
5500	01		
5600	0101010101		
5700	9406		
5800	0., 1., 0.2., 0.4., 0.6., 0.12., 0		
5900	3000., 540., 940., 1710., 2130., 2490		
6000	6100., 1040., 1560., 2260., 3100., 4110		
6100	7100., 970., 1710., 3120., 4210., 6310		

VAX/VMS	RTAC	TBSCONF4	30-APR-1985	12:54	LPBO:	30-APR-1985	12:55	TBSCONF4 DAT11
VAX/VMS	RTAC	TBSCONF4	30-APR-1985	12:54	LPBO:	30-APR-1985	12:55	TBSCONF4 DAT11
VAX/VMS	RTAC	TBSCONF4	30-APR-1985	12:54	LPBO:	30-APR-1985	12:55	TBSCONF4 DAT11

VAX/VU

	RRRR	TTTT	AAA	CCCC
R	R	T	A	C
R	R	T	A	C
R	R	T	A	C
R	R	T	AAAA	C
R	R	T	A	C
R	R	T	A	C

WAX / VMS

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LPBC: 30-APR-1985 12:55 TESCONF4 DAT11  
 LPBC: 30-APR-1985 12:55 TESCONF4 DAT11  
 LPBC: 30-APR-1985 12:55 TESCONF4 DAT11

DISCONF4 DAT:1

DISCONF4, DAT1

09:55 APR 22 '85 TBSCNFF 4.FBBELGIP

N.Y.Y. 05000

VAX/VMS	RTAC	LYPCONF4	21-APR-1985	13:00	LPBO.	21-APR-1985	13:00	DISK\$VAXUSER1:[USER1..RTAC]LYPCONF4..DAT1..1	VAX/VMS
VAX/VMS	RTAC	LYPCONF4	21-APR-1985	13:00	LPBO.	21-APR-1985	13:00	DISK\$VAXUSER1:[USER1..RTAC]LYPCONF4..DAT1..1	VAX/VMS
VAX/VMS	RTAC	LYPCONF4	21-APR-1985	13:00	LPBO.	21-APR-1985	13:00	DISK\$VAXUSER1:[USER1..RTAC]LYPCONF4..DAT1..1	VAX/VMS

	RARRA	TTTTT	AAA	CCCCC
R	R	T	A	C
R	R	T	A	C
R	R	T	A	C
R	R	T	AAA	C
R	R	T	A	C
R	R	T	A	C

```

VAX/VMS RTAC LYPCONF4 21-APR-1985 13:00 LPBO: 21-APR-1985 13:00 DISK4\MAXUSER1\1USER1.RTAC LYPCONF4.DAT1
VAX/VMS RTAC LYPCONF4 21-APR-1985 13:00 LPBO: 21-APR-1985 13:00 DISK4\MAXUSER1\1USER1.RTAC LYPCONF4.DAT1
VAX/VMS RTAC LYPCONF4 21-APR-1985 13:00 LPBO: 21-APR-1985 13:00 DISK4\MAXUSER1\1USER1.RTAC LYPCONF4.DAT1

```

54

174

05-AXLE TRACTOR-SEMI TRAILER (CONF 3 & 4 )

100	05500	,641000
200	214217	,4203700
300	60,76,59,24,107	24,180,28,228,28,0,0,0,0,0,0,
400	6B,89,227,723	
500	1600	,2480, ,3100, ,2480, ,2480, ,0,
600	0,0,0,0,0	
700	52,5,1,10,1,05	
800	267,400,400	,400, ,400, ,0
900	0,0,0,0,0	
1000	28000,29000	,28000, ,28000, ,0,0,
1100	0,0,0,0,0	
1200	13,13,13,13	,13,0,0,0,0,0,0,
1300	0,001,1	
1400	03	
1500	0,0	
1600	5,2,b	
1700	10,7,6	
1800		

**APPENDIX E**

**Vehicle Configuration 5**

**Input Data Files**

WAX UMS  
WAX SWMS  
HTAC

P4CONF3 21-APR-1983 14:19 LPBO: 21-APR-1983 14:19  
 P4CONF5 21-APR-1983 14:19 LPBO: 21-APR-1983 14:19  
 P4CONF3 21-APR-1983 14:19 LPBO: 21-APR-1983 14:19

DISK>VAXUSER1:[USER1]ATACJP4CONF5.DAT,1  
DISK>VALUSER1:[USER1]ATACJP4CONF5.DAT,1  
DISK>VAXUSER1:[USER1]ATACJP4CONF5.DAT,1

RRRR	TTTT	AAAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
RRRR	TTTT	A A A A	C C C C
R R	T T	A A	C C
R R	T T	A A	C C

	CCCC	AAA	TTTT	RRRR	WWWW
A	C	A	T	R	R
R	C	A	T	R	R
A	C	A	T	R	R
R	C	A	T	R	R
A	C	A	T	R	R
R	C	A	T	R	R
A	C	A	T	R	R
R	C	A	T	R	R

TRAC

PACONF3 21-APR-1983 14:19  
PACONF3 21-APR-1983 14:19  
PACONF3 21-APR-1983 14:19  
PACONF3 21-APR-1983 14:19

D:\DESK\>VAXUSER1:[USER1] ATACJP4CONF3.DAT[1]  
D:\DESK\>VAXUSER1:[USER1] ATACJP4CONF3.DAT[1]

100	5-AXLE TRACTOR-SEMITRAILER (CONF.	5.1	12000	05	18900	2300.	
200	01	6700	12., .033	12900	0	19000	13.
300	52.5	6800	06	13000	4	19100	-1
400	00.5	6900	0, 0.	13100	0	19200	-2
500	0.0.	7000	1., 143	13200	12.	19300	0.0
600	0.5, 1.5	7100	2., .252	13300	1., 1. 1. 1. 1. 1.	19400	1200.
700	10., 1.5, 1.5	7200	4., .490	13400	1., 1. 1. 1. 1. 1.	19500	45000. 1.3
800	04	7300	6., .501	13500	1., 1. 1. 1. 1. 1.	19600	19.3
900	0.0.	7400	12., .693	13600	75., .73, .75, .93, 1.	19700	115
1000	2.5. 0.	7500	06	13700	5., 5., 6., 9., 95, 1.	19800	13.
1100	2. 6. 0.	7600	0., 0.	13800	4., 4., 45., 85., 95	19900	-1.
1200	10., 0.	7700	1., .107	14000	-1600.	20000	-2
1300	0	7800	2., .194	14100	0., 0., 5. 0. 0. 0.	20100	0.
1400	0.1	7900	4., .380	14200	4500.	20200	1200.
1500	00	8000	6., .318	14300	19.3	20300	4500.
1600	11100000	8100	12., .683	14400	103	20400	19.5
1700	144.	8200	03	14500	01	20500	115
1800	0930.	8300	0	14600	48.	20600	0.03
1900	6540.	8400	.04	14700	50.	20700	0.29
2000	44.	8500	.1	14800	0.0	20800	1000.
2100	15000.	8600	.3	14900	-121.	20900	0
2200	75000.	8700	1.	15000	09	21000	0.075
2300	75000.	8800	03	15100	-20000. -11.	21100	0.25
2400	0.0	8900	0	15200	0., -1	21200	0.25
2500	0.	9000	4.	15300	0., 0.	21300	1500.
2600	40. 0.	9100	8.	15400	4000. 1.	21400	0.075
2700	50000.	9200	12.	15500	6500. 1. 5	21500	0.25
2800	36.	9300	16.	15600	9500. 2.	21600	1300.
2900	-119.	9400	1., 1., 9., 3., 1	15700	13000. 2. 5	21700	432.
3000	04	9500	1., 1., 9., 3., 1	15800	17000. 3	21800	4500.
31000	-20000. -20.	9600	1., 1., 9., 35., 13	15900	50000. 4. 0	21900	7300.
3200	0., 0.	9700	1., 1., 9., 42., 17	16000	0.02	22000	60.0
3300	9250., 7.2	9800	1., 1., 9., 48., 22	16100	-25000. -11.	22100	60000.
3400	25000., 7.5	9900	-2.	16200	0., 1. 0	22200	750000.
3500	0.08	10000	03	16300	0., 0. 2	22300	790000.
3600	-20000. -20.	10100	3000. , 6000. , 9000.	16400	3000. 1.	22400	32000.
3700	0., 0.	10200	01	16500	3000. 1. 5	22500	213. 94
3800	8040., 7.2	10300	66.	16600	0000. 2.	22600	85.0
3900	25000., 7.5	10400	05	16700	11300. 12. 5	22700	132000.
4000	0.08	10500	0., 0.	16800	15500. 1. 2	22800	3050000.
4100	0.	10600	1., 68	16900	40000. 1. 4	22900	3100000.
4200	0.	10700	2., .80	17000	0.02	23000	01.
4300	3719.	10800	3., .77	17100	0.0	23100	48. 0
4400	23.	10900	1. 0., 53	17200	0.0	23200	30.
4500	0.00	11000	05	17300	4450.	23300	0.0
4600	0.0	11100	0., 0.	17400	29.	23400	-122.
4700	32.	11200	1., .59	17500	0.0	23500	09.
4800	B0. 0	11300	1. 2., 73	17600	0.0	23600	-30000. -11.
4900	1200.	11400	1. 3., 73	17700	38.	23700	0., -1. 5
5000	01	11500	1., .50	17800	72.	23800	0.0
5100	0.0	11600	03	17900	2300.	23900	3375. , 0. 5
5200	-1.	11700	0., 0.	18000	-121.	24000	7313. , 1.
5300	03	11800	1., .44	18100	0	24100	11812. , 1. 3
5400	30000. , 6000. , 9000.	11900	1. 2., 70	18200	0	24200	16875. , 2.
5500	01	12000	1., .59	18300	4450.	24300	36250. , 3.
5600	06.	12100	1., .45	18400	27.	24400	0.02
5700	0., 0.	12200	05	18500	0.	24500	-35000. , -11.
5800	12300	0.	0.	18600	78000.	24600	0., -1. 3
5900	1., 183	12400	04	18700	38.	24700	24800
6000	2., .333	12500	1.	18800	72.	24800	0., 0. 2
6100	06.	12600	1.	18900	1.	24900	1687. , 0. 5
6200	0., 0.	12700	1.				

```

VAX/VMS RTAC YRCONF1 21-APR-1985 11:39 LPDO. 21-APR-1985 11:39 DISK4VAXUSER1:[USER1.RTAC]YRCONF1.DAT1]
VAX/VMS RTAC YRCONF1 21-APR-1985 11:39 LPDO. 21-APR-1985 11:39 DISK4VAXUSER1:[USER1.RTAC]YRCONF1.DAT2]
VAX/VMS RTAC YRCONF1 21-APR-1985 11:39 LPDO. 21-APR-1985 11:39 DISK4VAXUSER1:[USER1.RTAC]YRCONF1.DAT3]

```

	RRRR	TTTT	AAAA	CCCC
A	R	T	A	C
R	R	T	A	C
RRR	R	T	A	C
RRR	R	T	AAA	C
RRR	R	T	A	C
RRR	R	T	A	C

YACONF1 21-APR-1905 11:39 LPBO: 21-APR-1905 11:39  
YACONF1 21-APR-1905 11:39 LPBO: 21-APR-1905 11:39  
YACONF1 21-APR-1905 11:39 LPBO: 21-APR-1905 11:39

```
D1ISK%VAXUSER1:[USER1.RTAC.JYRCNF1.DAT]1 VAX/VMS  
D1ISK%VAXUSER1:[USER1.RTAC.JYRCNF1.DAT]1 VAX/VMS  
D1ISK%VAXUSER1:[USER1.RTAC.JYRCNF1.DAT]1 VAX/VMS
```

DIGIKA VAXUSER1: !USER1 RTACJYACON1: !DAT1  
DIGIKA VAXUSER1: !USER1 RTACJYACON1: !DAT1  
DIGIKA VAXUSER1: !USER1 RTACJYACON1: !DAT1



VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1  
TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1  
TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1

VAX/  
VAX/  
VAX/

RRRRH TTTTT AAAA CCCC  
R R T A A C  
R R T A A C  
RRRRH T A A C  
R R T AAAA C  
R R T A A C  
R R T A A CCCC

BBBBBBB BBBBDDDD  
TT TT DD DD  
TT BB DD DD  
TT BBBBDBB BBBBDBB  
TT BBBBDBB BBBBDBB  
TT DD BD BD  
TT DD BB BB  
TT JJ BB BB  
TT BBBBDBB BBBBDBB  
TT BBBBDBB BBBBDBB

SSSSSSS SSSSSSS  
SS SS CC CC  
SS CC CC  
SSS SSSS CC  
SSSSS CC  
SS CC  
SS CC  
SS CC  
SSS SSSS CC  
SSSSS CC

CCCCCCC CCCCCCC  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000  
0000000 0000000

HHHHH HHHHH  
DD DD AA AA  
DD DU AA AA  
DD DD AA AA  
DD DU AA AA  
DD DD AA AA  
DD DD AAAA  
DD DD AAAA  
DD AA AA  
DD DD AA AA  
DD DD AA AA  
DD DU AA AA

TTTTTTTTT TTTTTTTTT  
TT TT TT TT  
TT TT TT TT

1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111  
1111 1111

RRRRH TTTTT AAAA CCCC  
R R T A A C  
R R T A A C  
RRRRH T A A C  
R R T AAAA C  
R R T A A C  
R R T A A CCCC

TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1  
TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1  
TBSCONF5 21-APR-1985 15:25 LPBO: 21-APR-1985 15:28 TBSCONF5 DAT1

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VMS RTAC

09:55 APR 22 '85 T85CONF5.F08EELGIP

RTAC

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LYPCONFS 21-APR-1985 14:18 LPDO: 21-APR-1985 14:18 DISK&VAXUSER1:[USER1] LYACJLYPCONFS.DAT.1 VAX.
LYPCONFS 21-APR-1985 14:18 LPDO: 21-APR-1985 14:18 DISK&VAXUSER1:[USER1] LYACJLYPCONFS.DAT.1 VAX.
LYPCONFS 21-APR-1985 14:18 LPDO: 21-APR-1985 14:18 DISK&VAXUSER1:[USER1] LYACJLYPCONFS.DAT.1 VAX.

```

	RRRR	TTTT	AAA	CCCC
R	R	T	A	C
R	R	T	A	C
RRRR	TTTT	T	A	C
R	R	T	AAAAA	C
R	R	T	A	C
R	R	T	A	C

卷之三

DISKSWAKUSER1 : LUSER1 : RTACJLYCONF5 : DAT1  
DISKSWAKUSER1 : LUSER1 : RTAC2LYCONF5 : DAT1  
DISKSWAKUSER1 : LUSER1 : RTAC3LYCONF5 : DAT1

05-AXLE TRACTOR-SEMITRAILER (CONF. 5)	
100	15500 , 64500
200	15500 , 64500
300	214217 , 42027400
400	60 , 76 , 59 , 24 , 107
500	241 , 327 , 773
600	1400 , 2800 , 2800 , 2400
700	0 , 0 , 0 , 0 , 0
800	32 , 5 , 1 , 10 , 1 , 05
900	240 , 420 , 420 , 400 , 400
1000	0 , 0 , 0 , 0 , 0 , 0
11000	20000 , 280000 , 280000 , 200000 , 0 , 0
12000	0 , 0 , 0 , 0 , 0
13000	13 , 13 , 13 , 13 , 0 , 0 , 0 , 0 , 0 , 0
14000	B , 0001 , 1
15000	03
	16000
	17000
	18000

**APPENDIX F**

**Vehicle Configuration 6**

**Input Data Files**

VAX / VAX / VAX /

PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1	LUSER1..RTACJP4CONF6..DATI..1
PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1 <th>LUSER1..RTACJP4CONF6..DATI..1</th>	LUSER1..RTACJP4CONF6..DATI..1
PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1 <th>LUSER1..RTACJP4CONF6..DATI..1</th>	LUSER1..RTACJP4CONF6..DATI..1

WAX / VMS RTAC

			CCCC
		AAA	
	TTTT		
RRRR	R	T	A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A A A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A C

CCCCCCCC	000000	NN	NN	FFFFFFFFFF	66666666
CCCCCCCC	000000	NN	NN	FFFFFFFFFF	66666666
CC	00	00	NN	FF	bb
CC	00	00	NN	FF	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CCCCCCCC	00000000	NNNNNNNN	NNNNNNNN	CCCCCCCC	6666666666666666
CCCCCCCC	00000000	NNNNNNNN	NNNNNNNN	CCCCCCCC	6666666666666666

RRRR	TTTT	AAA	CCCC
R R	T T	A A	C C
R R	T T	A A	C C
R R	T T	A A	C C
R R	T T	AAAAA	C C
R R	T T	A A	C C
R R	T T	A A	C C

VAX/VMS RTAC  
VAX/VMS RTAC  
VAX/VME RTAC

VAX /

PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1	LUSER1..RTACJP4CONF6..DATI..1
PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1 <th>LUSER1..RTACJP4CONF6..DATI..1</th>	LUSER1..RTACJP4CONF6..DATI..1
PACONF	30-APR-1985	12:53	LP00	30-APR-1985	12:53	DISK4\WAXUSER1 <th>LUSER1..RTACJP4CONF6..DATI..1</th>	LUSER1..RTACJP4CONF6..DATI..1

WAX / VMS RTAC

			CCCC
		AAA	
	TTTT		
RRRR	R	T	A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A A A A C
R	R	T	A A C
R	R	T	A A C
R	R	T	A A C

CCCCCCCC	000000	NN	NN	FFFFFFFFFF	66666666
CCCCCCCC	000000	NN	NN	FFFFFFFFFF	66666666
CC	00	00	NN	FF	bb
CC	00	00	NN	FF	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CC	00	00	NNNN	NN	bb
CCCCCCCC	00000000	NNNNNNNN	NNNNNNNN	CCCCCCCC	6666666666666666
CCCCCCCC	00000000	NNNNNNNN	NNNNNNNN	CCCCCCCC	6666666666666666

VAX /  
VAX /  
VAX /

## **SEATBELT / 27-Ft TRAILERS (CONF. 61)**

250000	1500.	31100	1500.
251000	01	31200	-1
252000	80.	31300	00
253000	0.		
254000	49.		
255000	232.		
256000	4250.		
257000	4250.		
258000	60.0		
259000	346000.		
260000	1150000.		
261000	1150000.		
262000	26300.		
263000	126.		
264000	85.0		
265000	650000.		
266000	3750000.		
267000	3750000.		
268000	00		
269000	-132.		
270000	0.0		
271000	0.0		
272000	4100.		
273000	29.		
274000	0.		
275000	100000.		
276000	3B.		
277000	72.		
278000	1500.		
279000	13.		
280000	-1.		
281000	-2.		
282000	0.		
283000	1200.		
284000	4500.		
285000	19.3		
286000	113.		
287000	00.		
288000	-132.		
289000	0.0		
290000	0.0		
291000	4100.		
292000	29.		
293000	0.		
294000	100000.		
295000	30.		
296000	72.		
297000	1500.		
298000	13.		
299000	-1.		
300000	-2.		
301000	0.		
302000	1200.		
303000	4500.		
304000	19.3		
305000	115.		
306000	0.223		
307000	.23		
308000	1500.		
309000	0.273		
310000	.25		



## 5-AXLE DOUBLE / 27-FT TRAILERS (CDNF. 6 )

100	0405	40. , 8 . , 1. , 02	6200	02
200	0405	020. 0101	6300	0407
300	500	9700 , 30300 , 1000. , 31000.	6400	0407
400	600	15000 , 101204 , 1900. , 100375	6500	0. 0 , 1. , 0. 3 , 0. 4
500	700	75000 , 475638 , 2560 , 497375.	6600	0. 5 , 92 , 100 , 125 , 100.
600	800	75000 , 469434 , 2360 , 490000.	6700	6000 , 130 , 261 , 300 , 354 , 373 , 332.
700	900	44 , 81 , 44 , 80 , 7	6800	4800 , 200 , 452 , 533 , 623 , 650 , 565.
800	1000	10000 , 17500 , 17500 , 17500.	6900	0405
900	1100	1200 , 2300 , 1500 , 1500 , 1500.	7000	0 , 1 , 2 , 3 , 4.
1000	1200	3719 , 4458 , 4100 , 4100 , 4100.	7100	3000 , 29 , 42 , 44 , 38.
1100	1300	24 , -96 , -119 , 0 , -126.	7200	6000 , 92 , 150 , 157 , 146.
1200	1400	19 , 5 , 19 , 5 , 19 , 5 , 19 , 5	7300	9000 , 152 , 267 , 329 , 335.
1300	1500	23 , 29 , 29 , 29 , 29 ,	7400	1.
1400	1600	16 , 19 , 19 , 19 , 19 ,	7600	006.
1500	1700	40 , 29 , 3 , 29 , 5 , 29 , 5 , 29 , 5	7700	0 , 0.
1600	1800	0 , 13 , 13 , 13 , 13 ,	7800	0 , 5 , 6 , 6.
1700	1900	4500 , 4500 , 4500 , 4500 , 4500.	7900	1 , 0 , 0.
1800	2000	0 , 0 , 0 , 0 , 0	8000	1 , 5 , -6 , -6.
1900	2100	1500 , 8600 , 10000 , 100000 , 100000	8100	2 , 0 , 0.
2000	2200	300 , 1000 , 1000 , 1000 , 1000	8200	4 , 0 , 0.
2100	2300	0 , 0 , 0 , 0 , 0		
2200	2400	-B7 , 27 , 133 , 07 , -134 , 93 , B0 , , 0 , , 126.		
2300	2500	4B , 44 , 4B		
2400	2600	1000000 , 0 , 1000000		
2500	2700	010304		
2600	2800	03		
2700	2900	0102030303		
2800	3000	04		
2900	3100	-20000 , -20.		
3000	3200	0 , 0.		
3100	3300	B645 , 7 , 2		
3200	3400	25000 , 7 , 5		
3300	3500	07		
3400	3600	-29000 , -11.		
3500	3700	0 , -1		
3600	3800	0 , 0.		
3700	3900	4650 , 1.		
3800	4000	7650 , 1 , 3		
3900	4100	11650 , 2 ,		
4000	4200	16300 , 2 , 5		
4100	4300	21600 , 3		
4200	4400	59500 , 4,		
4300	4500	09		
4400	4600	-40500 , -11.		
4500	4700	0 , -1 , 5		
4600	4800	0 , 0.		
4700	4900	2812 , 0 , 5		
4800	5000	7100 , 1 ,		
4900	5100	12175 , 1 , 5		
5000	5200	17213 , 1 , 2		
5100	5300	24063 , 1 , 2 , 5		
5200	5400	56000 , 1 , 3 ,		
5300	5500	01		
5400	5600	0101010101		
5500	5700	0106		
5600	5800	0 , 1 , 0 , 2 , 0 , 4 , 0 , 6 , 0 , 12 , 0		
5700	5900	3000 , 340 , 1710 , 2130 , 2490,		
5800	6000	6000 , 840 , 1500 , 2760 , 3480 , 4140,		
5900	6100	9000 , 1710 , 3420 , 4680 , 6210,		

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    AX/VMS RTAC LYPCDNF6 30-APR-1985 12:53 LPBO: 30-APR-1985 12:55
    AX/VMS RTAC LYPCDNF6 30-APR-1985 12:53 LPBO: 30-APR-1985 12:55
    AX/VMS RTAC LYPCDNF6 30-APR-1985 12:53 LPBO: 30-APR-1985 12:55

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10

				CCCC
RRRR	TTTT	AAA	A	C
R R	T T	A A	A	C
R R	T T	A A	A	C
R R R R	T T T T	AAAAA	A	C
R R R R	T T T T	A A A A	A	C
R R R R	T T T T	A A A A	A	C

DDDDDDDD	AAAAAA	TTTTTTTT	JJJJ	11
DDDDDDDD	AAAAAA	TTTTTTTT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DD	AA	TT	JJJJ	11
DDDDDDDD	AAAAAA	TTTTTTTT	JJJJ	11
DDDDDDDD	AAAAAA	TTTTTTTT	JJJJ	11

	RRRR	TTTT	AAA	CCCC
R	R	T	A	C
R	R	T	A	C
R	R	T	A	C
R	R	T	AAAA	C
R	R	T	A	C
R	R	T	A	CCCC

## 3-AXLE DOUBLE / 27-Ft TRAILER (CONF. 6 )

100	
200	13200, .47800, .2500, .32500,
300	132646, 64, 525908 56, 2560, .621588 25
400	3B 543 B1 435, 0, .113, 323, 0, 0,
500	0, 0, 126, , 0, , 0
600	72, 725, 138, 677, 149, 323, 80, , 0, , 126,
700	1500, .2800, , 0, .2800, , 0, , 0,
800	2800, , 0, , 2800, , 0, , 0,
900	31, 5, 0, 10, , 1, 05
1000	266, 7, 400, , 0, , 400, , 0, , 0
1100	400, , 0, , 400, , 0, , 0
1200	28000, , 0, , 28000, , 0, , 0, , 28000,
1300	0, , 28000, , 0, , 0,
1400	13, , 0, , 13, , 0, , 0, , 13, , 0, , 0, , 0,
1500	9, , 001, , 1
1600	03
1700	0, , 0,
1800	5, 2,
1900	10, , 2

## APPENDIX G

Basic Parameters of a 5-axle, Dirt Truck

WU1	= 1500	(lb)	Weight of first unsprung mass
WU2	= 3000	(lb)	Weight of second unsprung mass
WU3	= 4000	(lb)	Weight of third unsprung mass
WAX1	= 18,000	(lb)	Vertical load on the first axle
WAX2	= 26,000	(lb)	Vertical load on the second axle
WAX3	= 26,000	(lb)	Vertical load on the third axle
T1	= 40.25	(in.)	Half track width of the inner tires on the first axle
T2	= 29	(in.)	Half track width of the inner tires on the second axle
T3	= 29	(in.)	Half track width of the inner tires on the third axle
A1	= 0	(in.)	Dual tire spacing on the first axle
A2	= 13	(in.)	Dual tire spacing on the second axle
A3	= 13	(in.)	Dual tire spacing on the third axle
S1	= 16	(in.)	Half spring spacing at the first axle
S2	= 19	(in.)	Half spring spacing at the second axle
S3	= 19	(in.)	Half spring spacing at the third axle
ZS1	= 44	(in.)	Height of c.g. of the first sprung mass, above ground level
ZS2	= 44	(in.)	Height of c.g. of the second sprung mass, above ground level
ZS3	= 84.6	(in.)	Height of c.g. of the third sprung mass, above ground level
R1	= 21	(in.)	Rolling radius of tires on the first axle
R2	= 20	(in.)	Rolling radius of tires on the second axle
R3	= 20	(in.)	Rolling radius of tires on the third axle
HR1	= 22	(in.)	Height of roll centre of the first suspension above ground level
HR2	= 29	(in.)	Height of roll centre of the second suspension above ground level
HR3	= 29	(in.)	Height of roll centre of the third suspension above ground level

ZS	= 48	(in.)	Height of the fifth wheel centre above ground level
MFR	= 40,000	(lb/deg.)	Tractor Frame torsional stiffness
ZFR	= 34	(in.)	Tractor frame torsional axis height above ground level
KT11	= 8000	(lb/in.)	Vertical stiffness of tires per side on the first axle
KT21	= 10,000	(lb/in.)	Vertical stiffness of tires per side on the second axle
KT31	= 10,000	(lb/in.)	Vertical stiffness of tires per side on the third axle
KRS1	= 0	(lb/in./deg.)	Auxiliary roll stiffness of the first suspension
KRS2	= 20,000	(lb/in./deg.)	Auxiliary roll stiffness of the second suspension
KRS3	= 0	(lb/in./deg.)	Auxiliary roll stiffness of the third suspension
LASH5	= 0.1	(in.)	Lash in the fifth wheel
COULFR	= 11,000	(in.lb/deg.)	Coulomb friction present in the tractor frame
M5	= 1,000,000	(in./lb/deg.)	Torsional stiffness of the fifth wheel
MOMSEP	= 900,000	(in./lb)	Roll moment that causes separation of the fifth wheel plates
WS	= 30,000	(lb)	Vertical load on the fifth wheel
WS2	= 500	(lb)	Weight of the second sprung mass
KOVT1	= 1000	(in./lb/deg.)	Overturning stiffness of the tires on the first axle
KOVT2	= 2000	(in./lb/deg.)	Overturning stiffnesses of the tires on the second axle
KOVT3	= 2000	(in./lb/deg.)	Overturning stiffnesses of the tires on the third axle

a) First Suspension Spring Table

<u>Force (lb)</u>	<u>Deflection (in.)</u>
-10,000	-5
14,250	7.13
40,000	7.25

b) Second Suspension Spring Table

<u>Force (lb)</u>	<u>Deflection (in.)</u>
-20,000	-10
20,000	10

c) Third Suspension Spring Table

<u>Force (lb)</u>	<u>Deflection (in.)</u>
-15,000	-1.75
0	-0.75
0	0
15,000	1